

MUKAVEMET

GERİLME

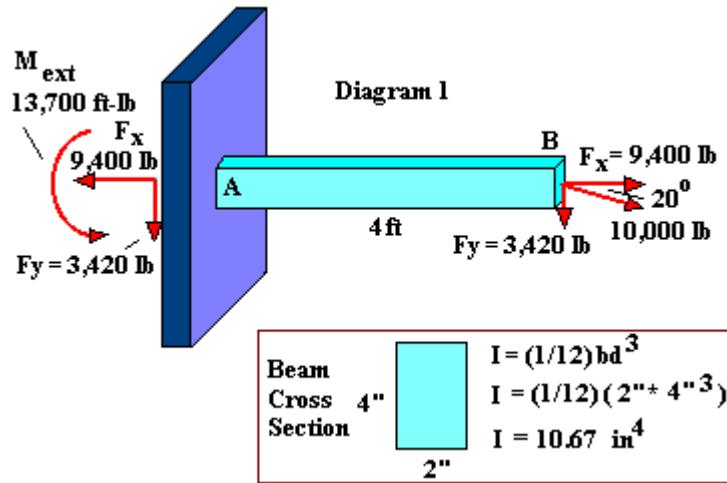
Gerilme birim alana gelen kuvvettir, (N/mm², kg/cm²)

σ Normal gerilme: Çubukun ekseni doğrultusunda oluşan gerilmedir.

τ kayma gerilmesi: çubukun eksenine dik doğrultuda oluşan gerilmedir.

$$\text{Gerilme} = F/A.$$

Konsol kirişte gerilmeler



$$\text{Normal Gerilme} = \text{Kuvvet}/\text{Alan} = 9,400 \text{ lb.} / (2'' \times 4'') = 1175 \text{ lb/in}^2.$$

$$\text{Kayma Gerilmesi} = \text{Kuvvet}/\text{Alan} = 3,420 \text{ lb.} / (2'' \times 4'') = 427.5 \text{ lb/in}^2.$$

DÜZLEM GERİLME 3 Boyutlu gerilme hali

Plane stress- 2D State of stress

3D - State of stress

$$\left[\sigma_{ij} \right] = \begin{bmatrix} \sigma_x & \tau_{xy} & \tau_{xz} \\ \tau_{yx} & \sigma_{yy} & \tau_{yz} \\ \tau_{zx} & \tau_{zy} & \sigma_{zz} \end{bmatrix} - 6 \text{ components}$$

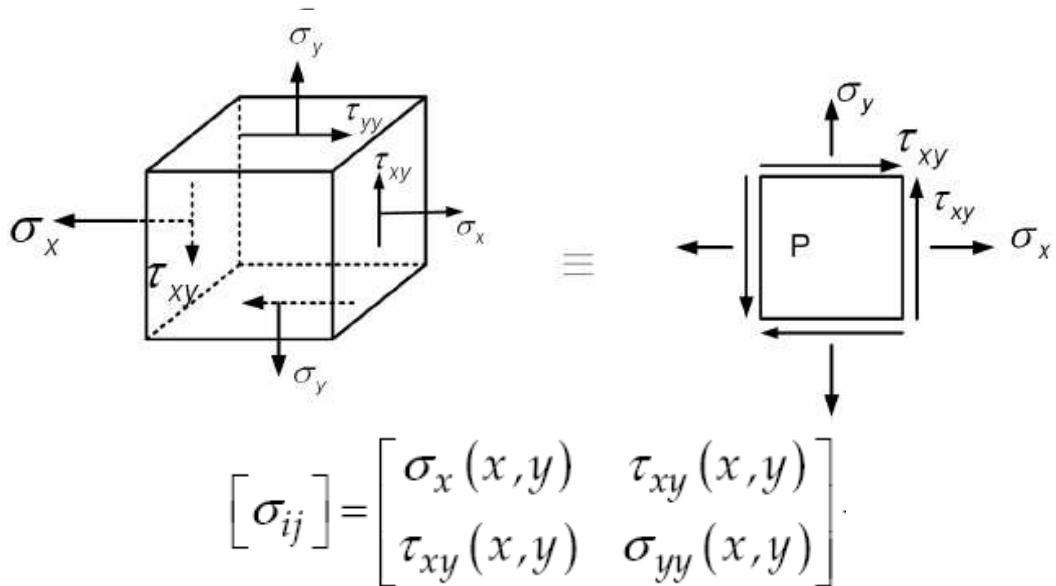
İki boyutlu gerilme hali

y
 x
 z

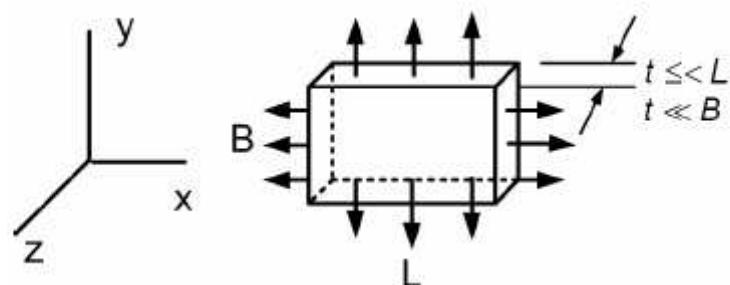
2D – State of stress

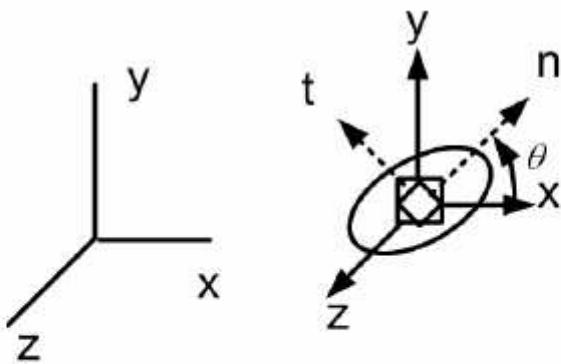
$$\begin{bmatrix} \sigma_{ij} \end{bmatrix} = \begin{bmatrix} \sigma_x & \tau_{xy} & 0 \\ \tau_{xy} & \sigma_y & 0 \\ 0 & 0 & 0 \end{bmatrix} = \begin{bmatrix} \sigma_x & \tau_{xy} \\ \tau_{yx} = \tau_{xy} & \sigma_y \end{bmatrix}$$

Stress components in plane xy

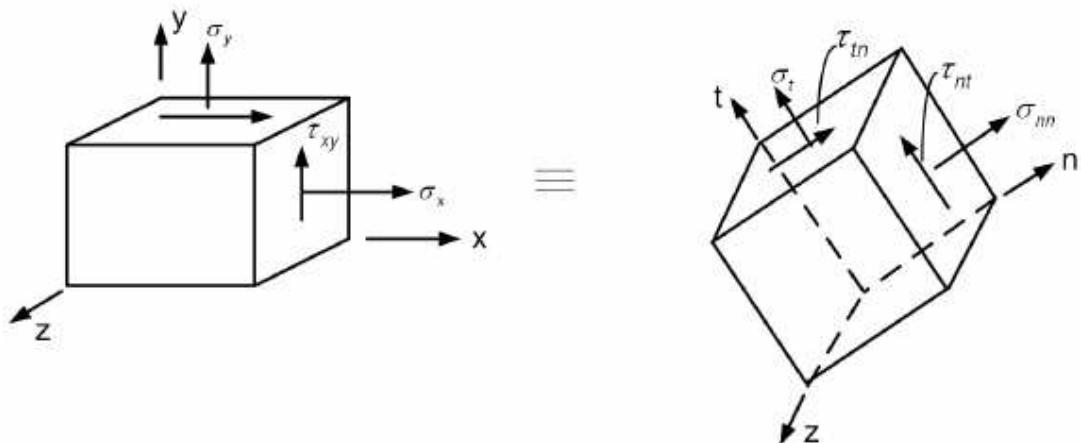


Kalınlık diğer boyutların yanında çok küçük olduğundan düzlem olarak göz önüne alınabilir.





$$\begin{bmatrix} \sigma_{ij} \end{bmatrix} = \begin{bmatrix} \sigma_x & \tau_{xy} \\ \tau_{xy} & \sigma_y \end{bmatrix} = \begin{bmatrix} \sigma_{nn} & \tau_{nt} \\ \tau_{nt} & \sigma_{tt} \end{bmatrix}$$



Eğik düzlemede

$A' = A/\cos(\theta)$.

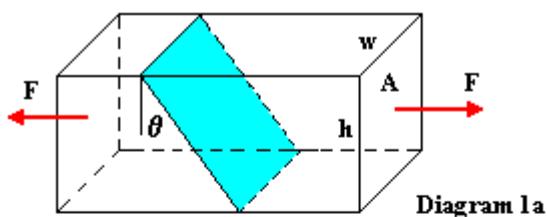


Diagram 1a

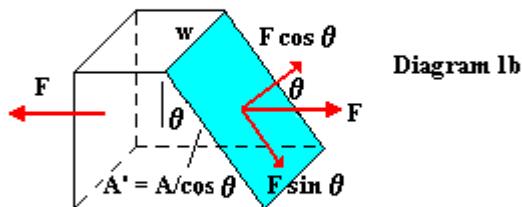


Diagram 1b

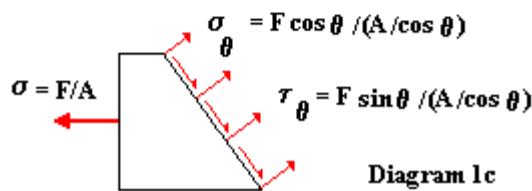


Diagram 1c

Normal Gerilme = $F \cos (\theta) / A / \cos (\theta) = (F/A) \cos^2(\theta)$

Kayam Gerilmesi = $F \sin (\theta) / A / \cos (\theta) = (F/A) \sin(\theta) * \cos(\theta)$

veya

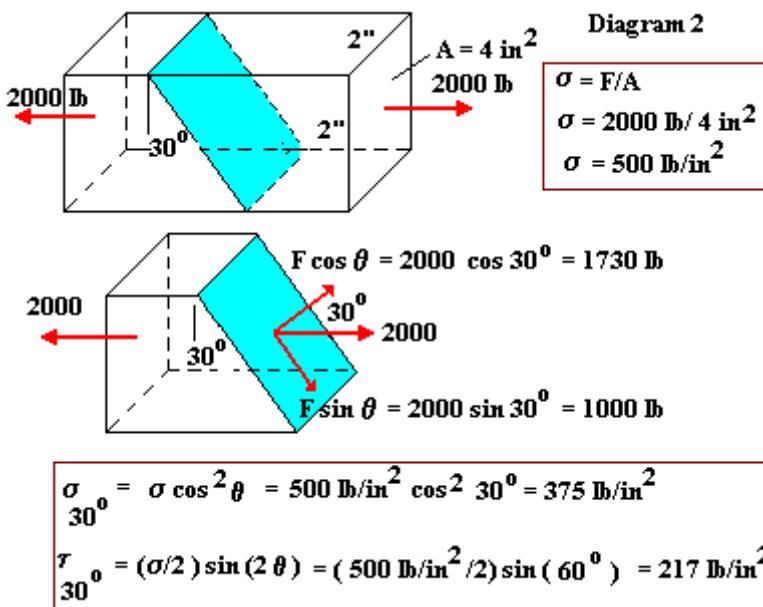
Kayma Gerilmesi = $(F/2A)\sin (2*\theta)$,

$$\sigma_\theta = \sigma \cos^2 \theta$$

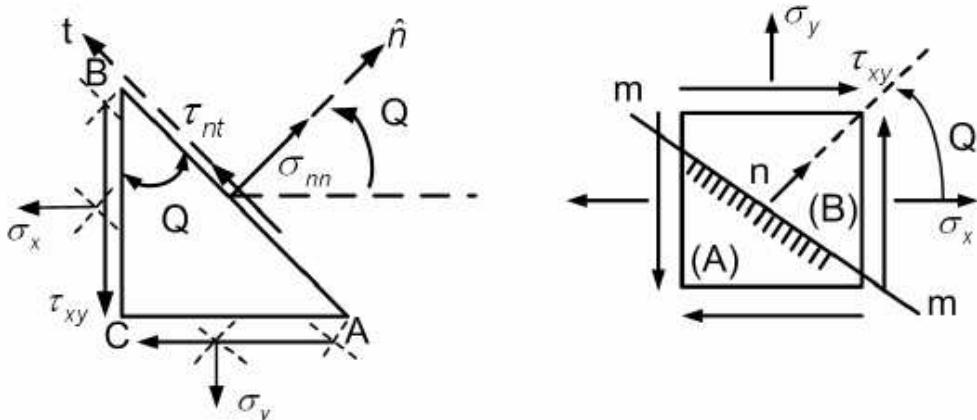
$$\tau_\theta = (\sigma / 2) \sin 2\theta$$

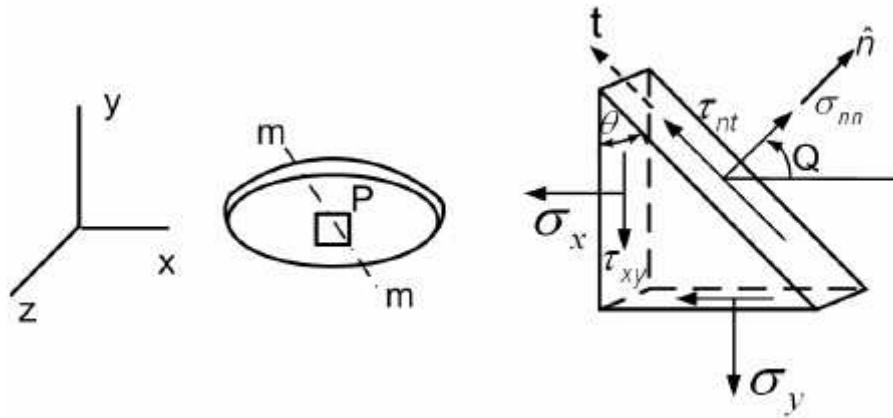
Örnek

Kare kesitli kiriş te, 30 derece lik düzlemdeki gerilmeler



Q yapan düzlemdeki gerilmelerin hesabı





Diferansiyel elemanda boyutlar

$$dA = \text{Area of } AB$$

$$dACs\theta = \text{Area of } BC$$

$$dASin\theta = \text{Area of } AC$$

Denge denklemlerinden q açısı yapan düzlemdeki gerilmeler hesaplanıyor

$$\left[\sum F_n \nearrow + = 0 \right]$$

$$\begin{aligned} \sigma_{nn}dA - \sigma_x dACos\theta Cos\theta - \tau_{xy} dACos\theta Sin\theta - \tau_{xy} dASin\theta Cos\theta - \\ \sigma_{yy} dASin\theta Sin\theta = 0 \end{aligned}$$

$$\sigma_{nn} - \sigma_x Cos^2\theta - 2\tau_{xy} Sin\theta Cos\theta - \sigma_{yy} Sin^2\theta = 0$$

$$\begin{aligned} \sigma_{nn} &= \sigma_x Cos^2\theta + \sigma_y Sin^2\theta + 2\tau_{xy} Sin\theta Cos\theta \\ \sigma_{nn} &= \frac{\sigma_x + \sigma_y}{2} + \frac{\sigma_x - \sigma_y}{2} Cos2\theta + \tau_{xy} Sin2\theta \end{aligned}$$

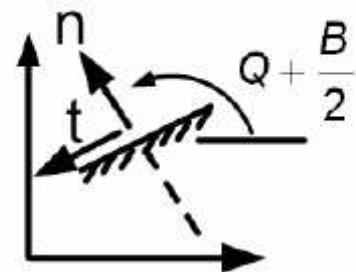
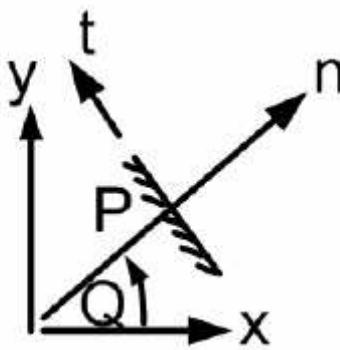
$$\left[\sum F_n \nwarrow + = 0 \right]$$

$$\begin{aligned} \sigma_{nt}dA - \sigma_x dACos\theta Sin\theta - \tau_{xy} dACos\theta Cos\theta + \tau_{xy} dASin\theta Sin\theta - \\ \sigma_y dASin\theta Cos\theta = 0 \end{aligned}$$

$$\tau_{nt} = -\sigma_x Cos\theta Sin\theta + \sigma_y Sin\theta Cos\theta + \tau_{xy} (Cos^2\theta - Sin^2\theta)$$

$$\tau_{nt} = -\cos\theta \sin\theta (\sigma_x - \sigma_y) + \tau_{xy} (\cos^2\theta - \sin^2\theta)$$

$$\tau_{nt} = -\frac{(\sigma_x - \sigma_y)}{2} \sin 2\theta + \tau_{xy} \cos 2\theta$$



2θ cinsinden aynı ifadeler tekrar yazılıyor

$$\sigma_{nn} = \frac{\sigma_x + \sigma_y}{2} + \frac{\sigma_x - \sigma_y}{2} \cos 2\theta + \tau_{xy} \sin 2\theta$$

$$\sigma_t = \frac{\sigma_x + \sigma_y}{2} + \frac{\sigma_x - \sigma_y}{2} \cos 2\theta - \tau_{xy} \sin 2\theta$$

$$\tau_{nt} = -\frac{\sigma_x - \sigma_y}{2} \sin 2\theta + \tau_{xy} \cos 2\theta$$

Düzlem gerilmelerde değişmezler (invaryantlar)
Birinci değişmez

$$\sigma_n + \sigma_t = \sigma_x + \sigma_y = \sigma_{x'} + \sigma_{y'} = I_1$$

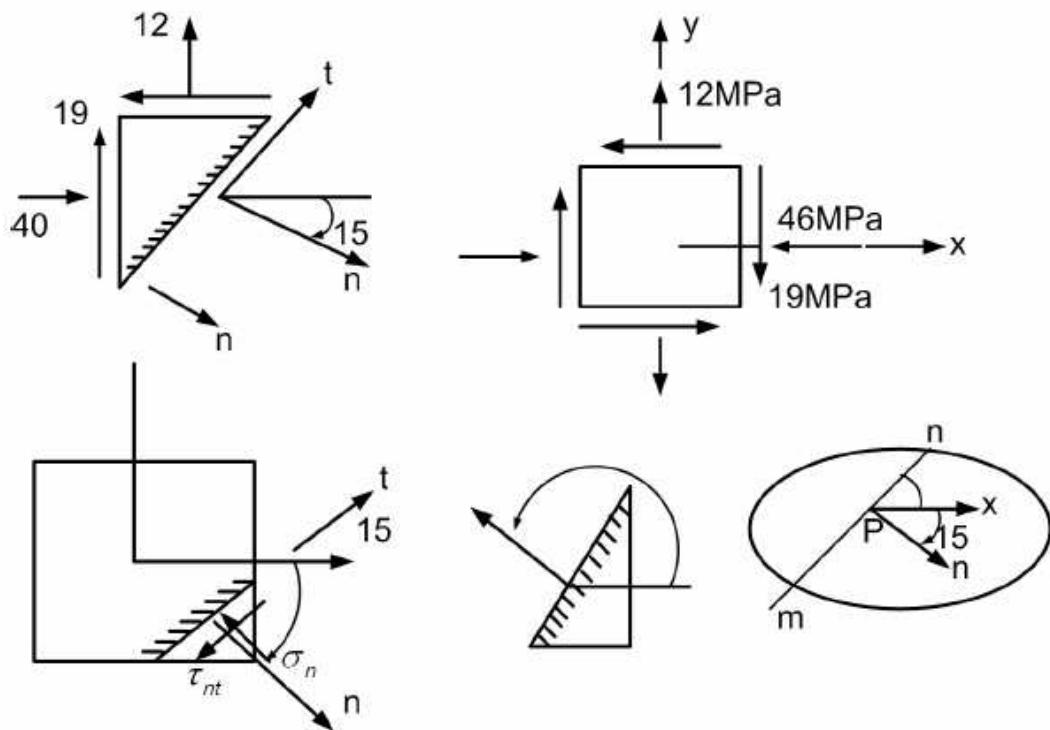
İkinci değişmez

$$\sigma_n \sigma_t - \tau_{nt}^2 = \sigma_x \sigma_y - \tau_{xy}^2 = \sigma_{x'} \sigma_{y'} - \tau_{x'y'}^2 = I_2$$

ÖRNEK

Düzlem gerilme durumunda bir noktadaki gerilme şekilde verilmiştir
Saat yönünde 15 derece açı yapan düzlemdeki gerilmeleri bulunuz

ÇÖZÜM



$$\sigma_x = -46$$

$$\sigma_y = 12$$

$$\tau_{xy} = -19$$

$$Q = -15^\circ$$

$$\frac{\sigma_x + \sigma_y}{2} = \frac{-46 + 12}{2} = \frac{-34}{2} = -17 \text{ MPa}$$

$$\frac{\sigma_x - \sigma_y}{2} = \frac{-46 - 12}{2} = \frac{-58}{2} = -29 \text{ MPa}$$

$$\sin 2\theta = \sin 2(-15) = -0.5; \cos 2\theta = \cos 2(-15) = 0.866$$

$$\sigma_n = -17 - 29 \times 0.866 + 19 \times 0.5$$

$$\left[\sigma_n = \frac{\sigma_x + \sigma_y}{2} + \frac{\sigma_x - \sigma_y}{2} \cos 2\theta + \tau_{xy} \sin 2\theta \right]$$

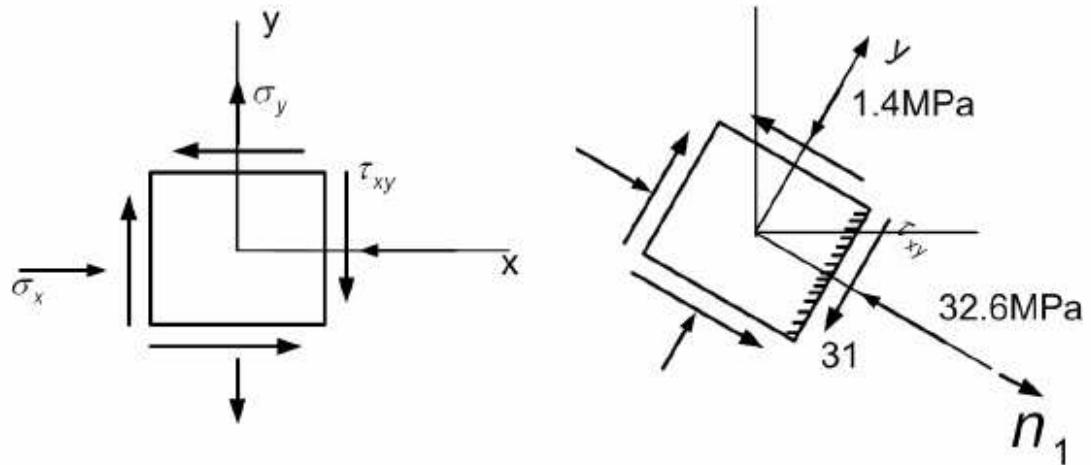
$$\sigma_{n_1} = -32.6 \text{ MPa}$$

$$\theta = -15^\circ \text{ in } \tau_{nt}$$

$$\left[\tau_{nt} = -\frac{\sigma_x - \sigma_y}{2} \sin 2\theta + \tau_{xy} \cos 2\theta \right]$$

$$\tau_{n_1 t_1} = -29 \times 0.5 - 19 \times 0.866$$

$$\tau_{n_1 t_1} = -31 \text{ MPa}$$



$$\left[\sigma_t = \sigma_{n_2} = \tau_{nt} \Big|_{\theta=75^\circ} \right]$$

$$\therefore \sigma_t = -17 - 29 \cos 150 - 19 \sin 150$$

$$\sigma_t = -1.4 \text{ MPa}$$

$$\left[\tau_{tn} = \tau_{n_2 t_2} = \tau_{nt} \Big|_{\theta=75^\circ} \right]$$

$$\begin{aligned}\tau_{tn} &= +29 \times \sin 150 - 19 \times \cos 150 \\ &= 31 \text{ MPa}\end{aligned}$$

$$\theta = 145^\circ$$

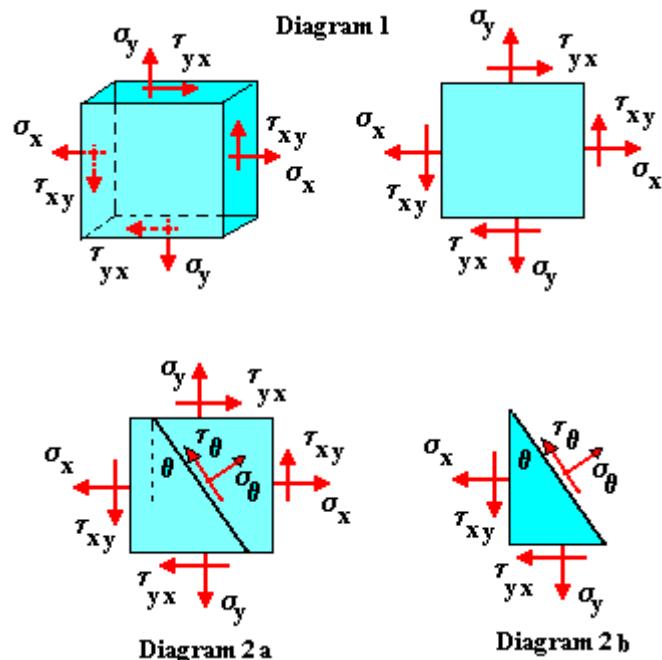
$$\begin{aligned}\sigma_n &= -17 - 29 \cos 2 \times 165 - 19 \sin 2 \times 165 \\ &= -32 \text{ MPa}\end{aligned}$$

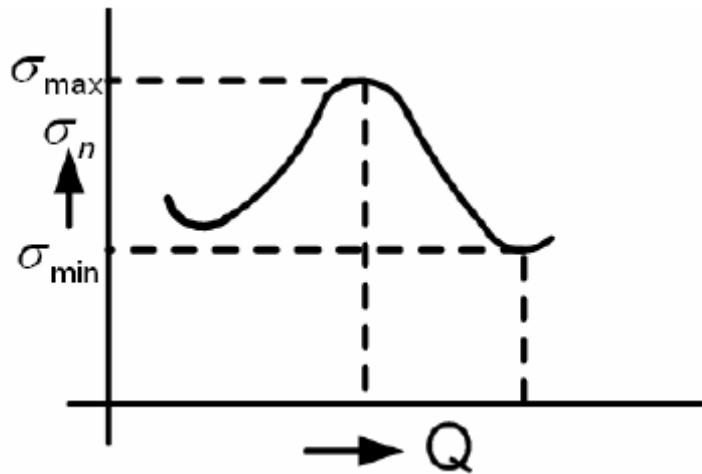
$$\begin{aligned}\tau_{nt} &= 0.29 \sin 330 - 19 \cos 330^0 \\ \tau_{nt} &= -31 \text{ MPa}\end{aligned}$$

KONTROL

$$\sigma_n + \sigma_t = \sigma_x + \sigma_y = -32.6 - 1.4 = -34 \text{ MPa} = -46 + 12s$$

Asal gerilmeler





$$\sigma_n = \sigma_n(\theta) = \frac{\sigma_x + \sigma_y}{2} + \frac{\sigma_x - \sigma_y}{2} \cos 2\theta + \tau_{xy} \sin 2\theta$$

$$\sigma_n = \frac{\sigma_x + \sigma_y}{2} + \frac{\sigma_x - \sigma_y}{2} \cos 2\theta + \tau_{xy} \sin 2\theta$$

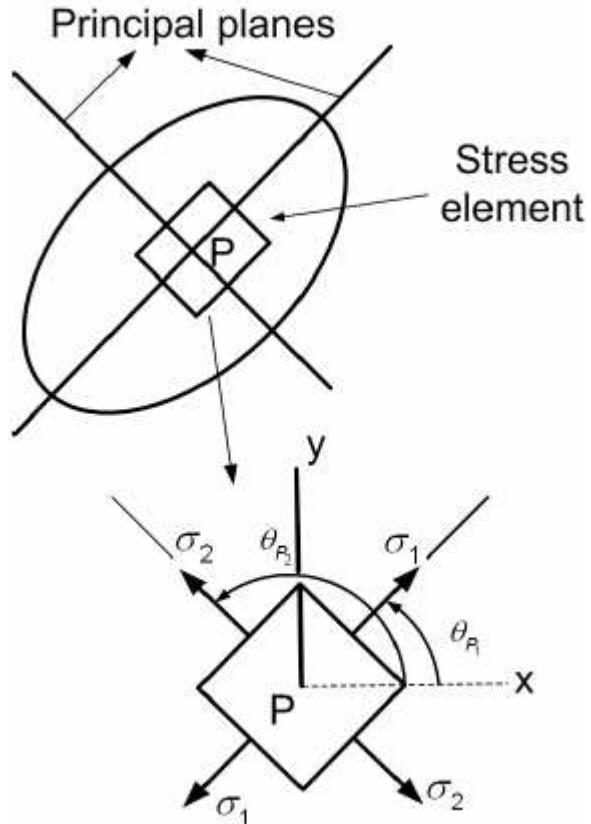
MAKSİ MUM İ Çİ N

$$\frac{d\sigma_n}{d\theta} = 0 = -(\sigma_x - \sigma_y) \sin 2\theta + 2\tau_{xy} \cos 2\theta$$

$$\tan 2\theta = \frac{2\tau_{xy}}{\sigma_x - \sigma_y}$$

$$\sigma_1 = \frac{\sigma_x + \sigma_y}{2} + \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2}$$

$$\sigma_2 = \frac{\sigma_x + \sigma_y}{2} - \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2}$$



$$\tau_{max} = +\sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2}$$

$$\sigma_n = \frac{\sigma_x + \sigma_y}{2} + \frac{\sigma_x - \sigma_y}{2} \cos 2\theta + \tau_{xy} \sin 2\theta$$

$$\sigma_n - \frac{\sigma_x + \sigma_y}{2} = \frac{\sigma_x - \sigma_y}{2} \cos 2\theta + \tau_{xy} \sin 2\theta$$

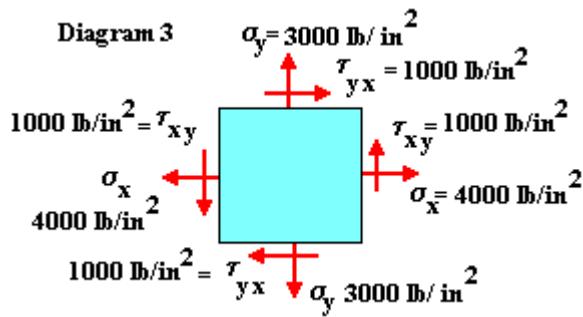
$$\tau_{nt} = -\left(\frac{\sigma_x - \sigma_y}{2}\right) \sin 2\theta + \tau_{xy} \cos 2\theta$$

$$\sigma_{max} = (\sigma_x + \sigma_y)/2 + \tau_{max}$$

$$\sigma_{min} = (\sigma_x + \sigma_y)/2 - \tau_{max}$$

$$\tau_{max} = (\sigma_{max} - \sigma_{min})/2$$

Örnek.



Çözüm

$$\tan 2\theta = \tau_{xy} / [(\sigma_x - \sigma_y) / 2] = 1000 \text{ lb/in}^2 / [(4000 \text{ lb/in}^2 - 3000 \text{ lb/in}^2) / 2] = 2,$$

$$2(\theta) = 63.4^\circ, \text{ ve } 243.4^\circ, \quad \theta = 31.7^\circ, \text{ ve } 121.7^\circ.$$

$$\sigma_\theta = (\sigma_x + \sigma_y) / 2 + [(\sigma_x - \sigma_y) / 2] (\cos 2\theta) + \tau_{xy} (\sin 2\theta) = (4000 \text{ lb/in}^2 + 3000$$

$$\text{lb/in}^2) / 2 + [(4000 \text{ lb/in}^2 - 3000 \text{ lb/in}^2) / 2] \cos(63.4^\circ) + 1000 \text{ lb/in}^2 * \sin(63.4^\circ)$$

$$= 3500 \text{ lb/in}^2 + 224 \text{ lb/in}^2 + 894 \text{ lb/in}^2 = 4618 \text{ lb/in}^2$$

2. düzleme

$$= (4000 \text{ lb/in}^2 + 3000 \text{ lb/in}^2) / 2 + [(4000 \text{ lb/in}^2 - 3000 \text{ lb/in}^2) / 2] \cos(243.4^\circ) + 1000 \text{ lb/in}^2 * \sin(243.4^\circ)$$

$$= 3500 \text{ lb/in}^2 - 224 \text{ lb/in}^2 - 894 \text{ lb/in}^2 = 2382 \text{ lb/in}^2$$

I kinci metot formüllerden

$$\sigma_{\max/\min} = (\sigma_x + \sigma_y) / 2 \pm \sqrt{[(\sigma_x - \sigma_y) / 2]^2 + \tau_{xy}^2} = (4000 \text{ lb/in}^2 + 3000$$

$$\text{lb/in}^2) / 2 \pm \sqrt{[(4000 \text{ lb/in}^2 - 3000 \text{ lb/in}^2) / 2]^2 + (1000 \text{ lb/in}^2)^2}$$

$$= 3500 \text{ lb/in}^2 + 1118 \text{ lb/in}^2 = 4618 \text{ lb/in}^2$$

$$= 3500 \text{ lb/in}^2 - 1118 \text{ lb/in}^2 = 2382 \text{ lb/in}^2$$

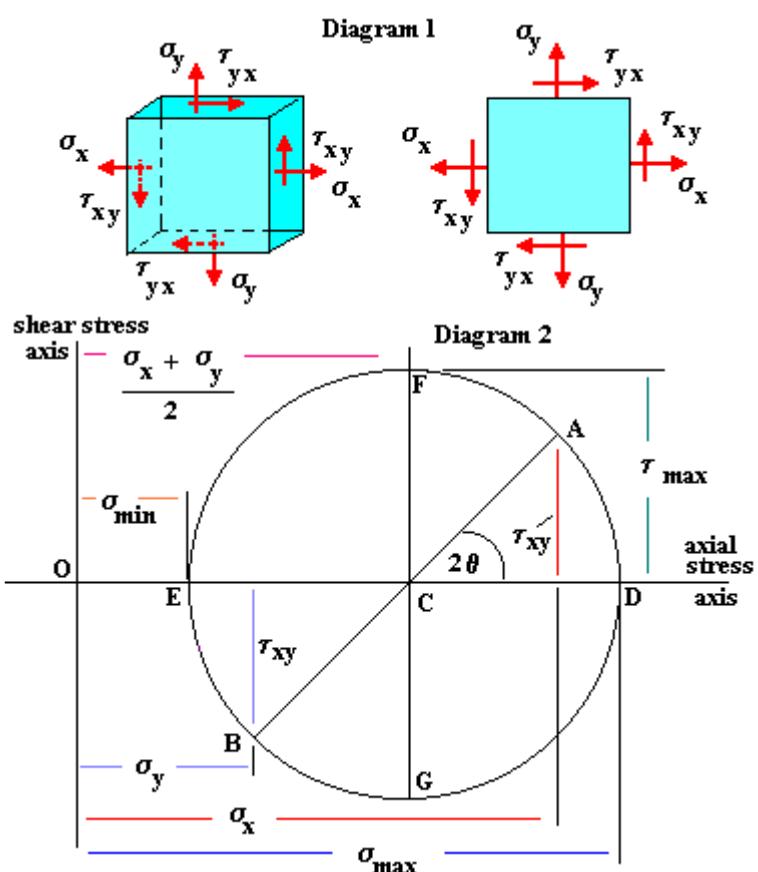
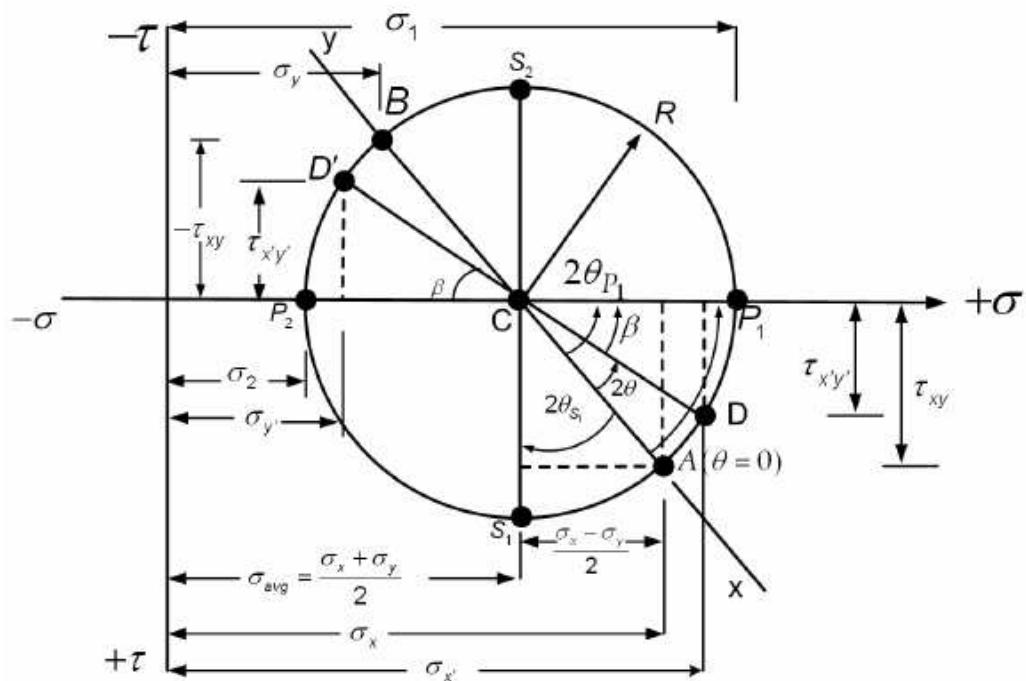
$$\tau_{\max} = \pm \sqrt{[(\sigma_x - \sigma_y) / 2]^2 + \tau_{xy}^2} = \sqrt{[(4000 \text{ lb/in}^2 - 3000 \text{ lb/in}^2) / 2]^2 + (1000 \text{ lb/in}^2)^2} = \pm 1118 \text{ lb/in}^2.$$

Mohr Dairesi

$$\left[\sigma_n - \left(\frac{\sigma_x + \sigma_y}{2} \right) \right]^2 + \tau_{nt}^2 = \left(\frac{\sigma_x + \sigma_y}{2} \right)^2 + \tau_{xy}^2$$

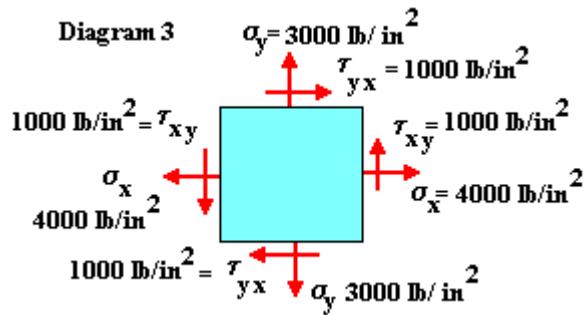
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$$(x - a)^2 + y^2 = R^2$$



$$R = \sqrt{[(\sigma_x - \sigma_y)/2]^2 + \tau_{xy}^2},$$

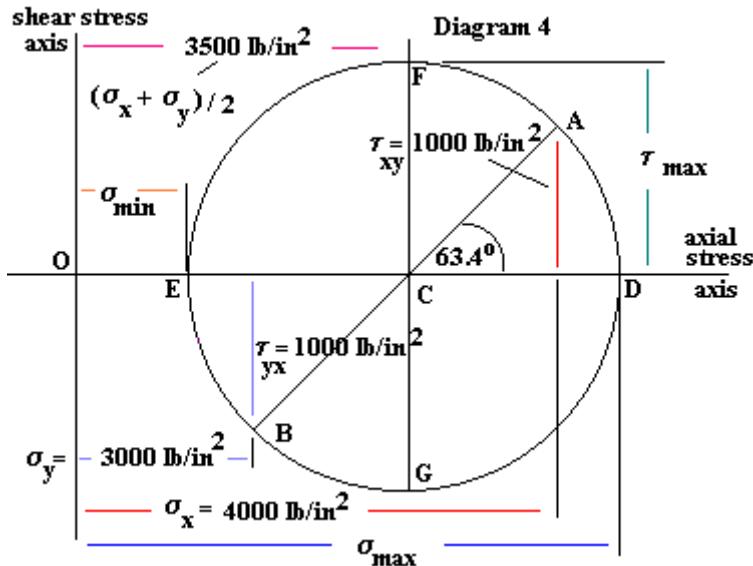
Örnek.



$$R = \sqrt{[(\sigma_x - \sigma_y)/2]^2 + \tau_{xy}^2} = \sqrt{[(4000 \text{ lb/in}^2 - 3000 \text{ lb/in}^2)/2]^2 + (1000 \text{ lb/in}^2)^2}$$

$$R = 1118 \text{ lb/in}^2.$$

Mohr Dairesi merkezi = $(4000 \text{ lb/in}^2 + 3000 \text{ lb/in}^2)/2 = 3500 \text{ lb/in}^2$.



Maximum Stress = Location of Center + Radius = $3500 \text{ lb/in}^2 + 1118 \text{ lb/in}^2 = 4618 \text{ lb/in}^2$

and

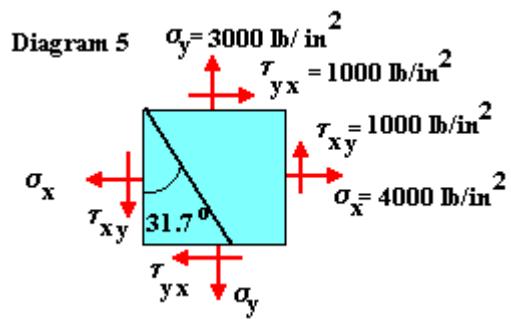
Minimum Stress = Location of Center - Radius = $3500 \text{ lb/in}^2 - 1118 \text{ lb/in}^2 = 2382 \text{ lb/in}^2$

Geometriden

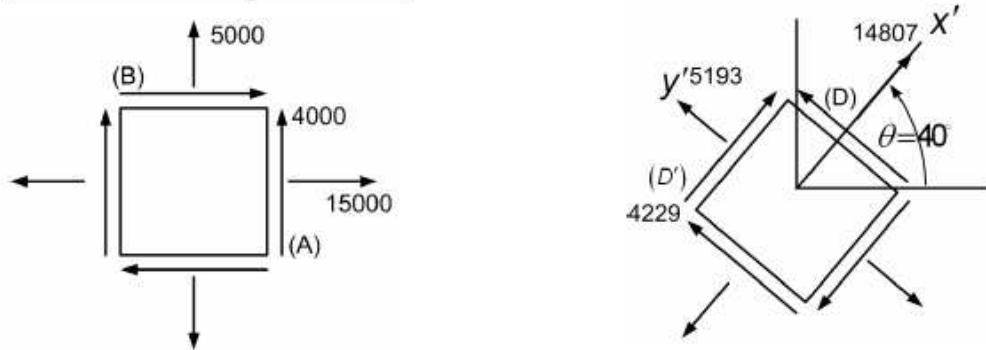
$$(2\theta) = (1000 \text{ lb/in}^2)/(4000 \text{ lb/in}^2 - 3500 \text{ lb/in}^2),$$

$$(2\theta) = 63.4^\circ \text{ and } 243.4^\circ,$$

$$\theta = 31.7^\circ \text{ and } 121.7^\circ.$$



Mohr's circle problem



Solution:

$$\frac{\sigma_x + \sigma_y}{2} = \frac{15000 + 5000}{2} = 10000 \text{ MPa}$$

$$A - (15000, 4000)$$

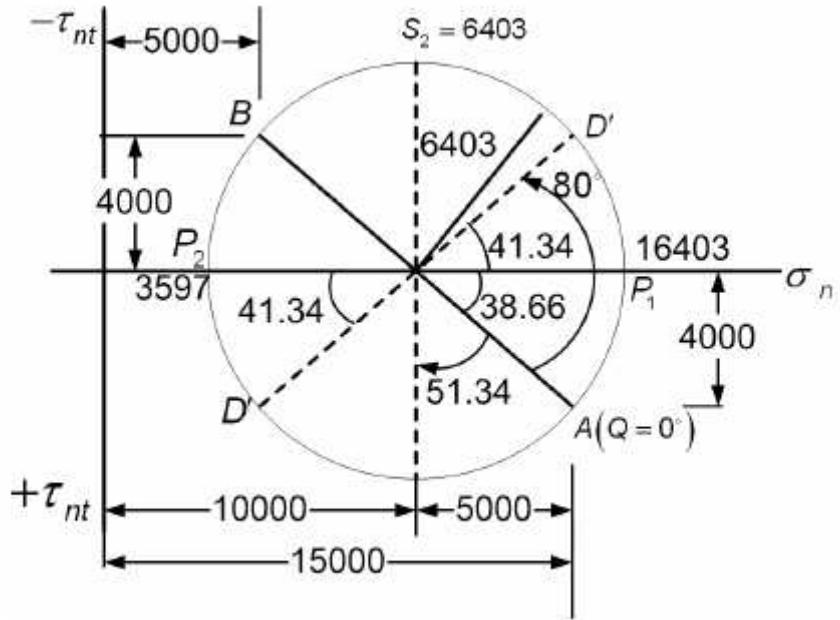
$$B - (5000, -4000)$$

$$R = \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2} = \sqrt{\left(\frac{15000 - 5000}{2}\right)^2 + 4000^2}$$

$$= \sqrt{5000^2 + 4000^2}$$

$$R = 6403 \text{ MPa}$$

$$(a) \quad \frac{\sigma_x - \sigma_y}{2} = 5000$$



$$\text{Point } D: \sigma_{x'} = 10000 + 6403 \cos 41.34 = 14807 \text{ MPa}$$

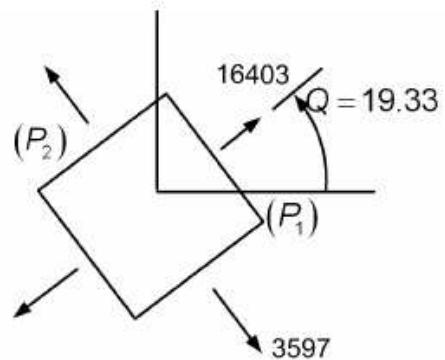
$$\tau_{x'y'} = -6403 \sin 41.34 = -4229 \text{ MPa}$$

$$\text{Point } D': \sigma_n = \sigma_{y'} = 10000 - 6403 \cos 41.34 = 593 \text{ MPa}$$

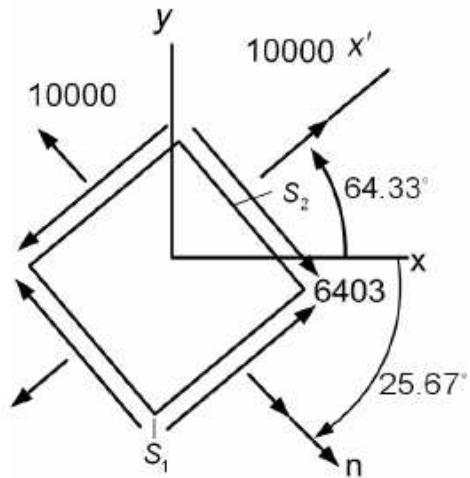
$$\tau_{nt} = \tau_{x'y'} = 6403 \sin 41.34 = 4229$$

b) $\sigma_1 = 16403 ; \theta_{P_1} = \frac{38.66}{2} = 19.33$

$$\sigma_2 = 3597 \text{ MPa}$$



c) $\tau_{max} = 6403 \text{ MPa} - \theta_{S_1} = 25.67^\circ = -25.67^\circ$

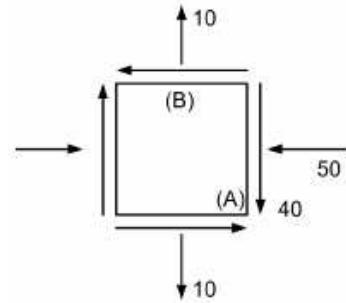


(2) $\theta = 45^\circ$

Principal stresses and principal shear stresses.

Solution:

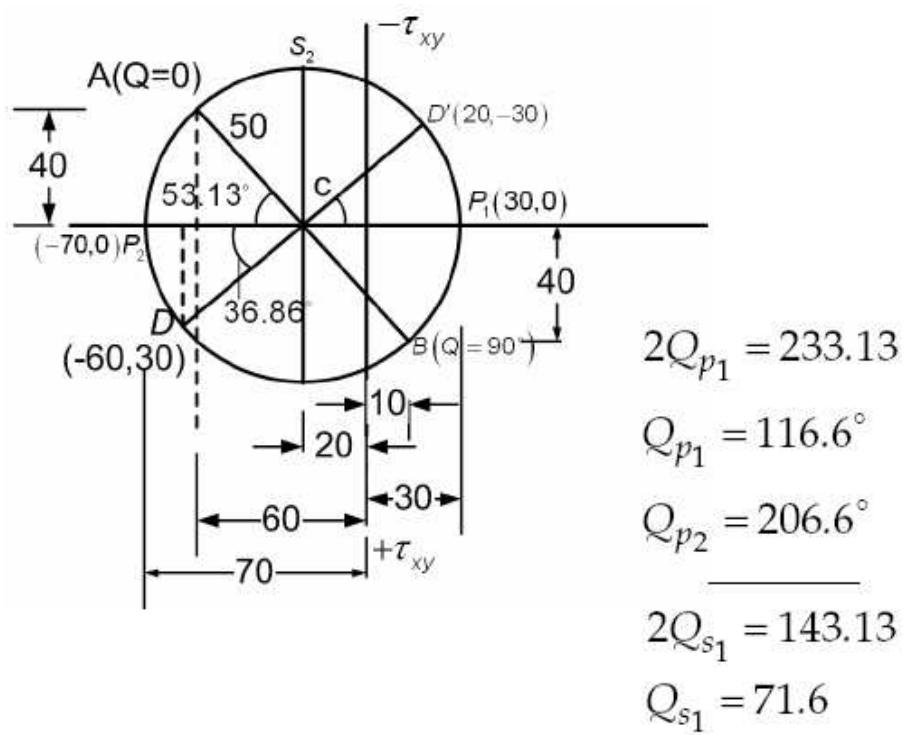
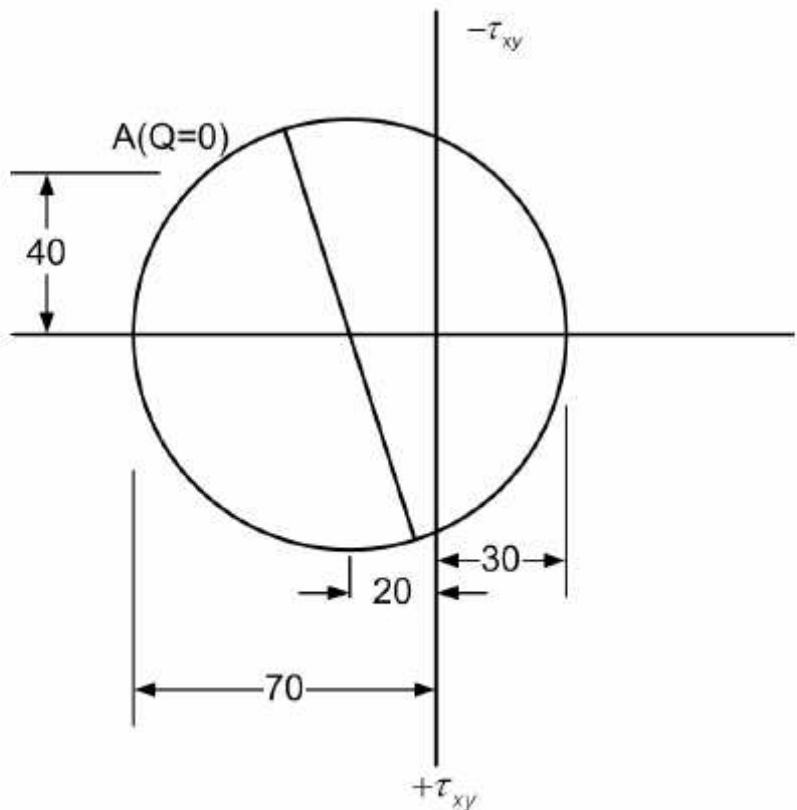
$$\frac{\sigma_x + \sigma_y}{2} = \frac{-50 + 10}{2} = -20$$



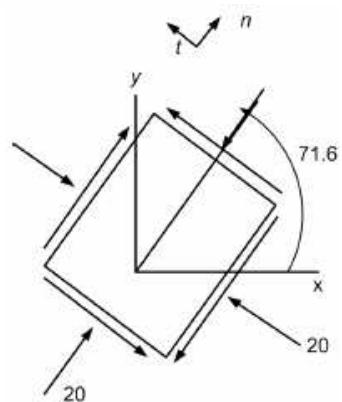
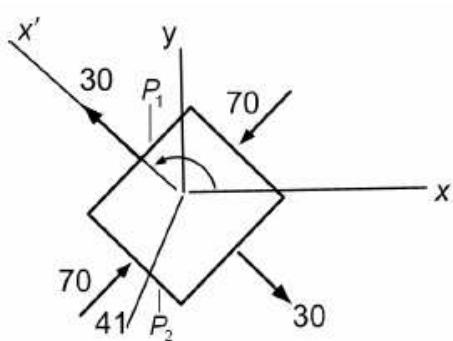
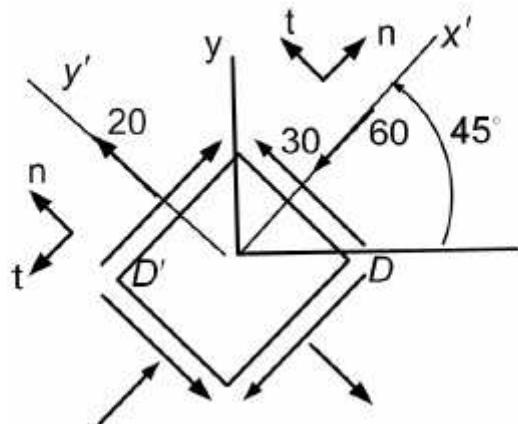
$$R = \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2} = \sqrt{\left(\frac{-50 - 10}{2}\right)^2 + (-40)^2} = 50 \text{ MPa}$$

$$A \rightarrow (-50, -40) \quad p_1 = \sigma_1 = \frac{\sigma_x + \sigma_y}{2} + R = -20 + 50 = 30 \text{ MPa}$$

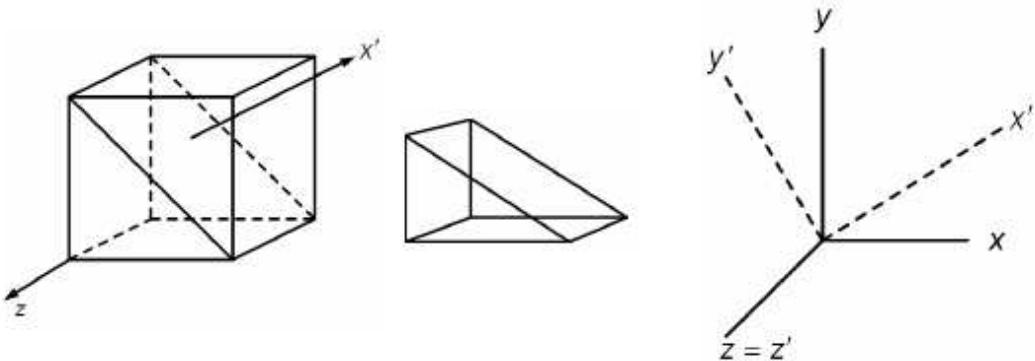
$$B \rightarrow (10, 40) \quad p_2 = \sigma_2 = \frac{\sigma_x + \sigma_y}{2} - R = -20 - 50 = -70 \text{ MPa}$$



$$Q_{s_2} = 161.6$$

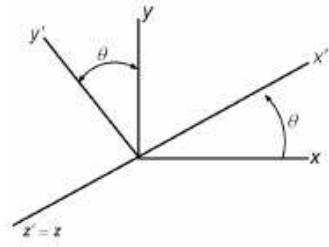


3 boyutlu uzayda gerilme transformasyonu



$$\begin{array}{l|l|l}
 n_{x'x} = \cos\theta & n_{y'x} = -\sin\theta & n_{z'x} = 0 \\
 n_{x'y} = \sin\theta & n_{y'y} = \cos\theta & n_{z'y} = 0 \\
 n_{x'z} = 0 & n_{y'z} = 0 & n_{z'z} = 1
 \end{array}$$

$\sigma_{z'} = 0; \tau_{x'z'} = 0; \tau_{y'z'} = 0$
 $= \sigma_z$



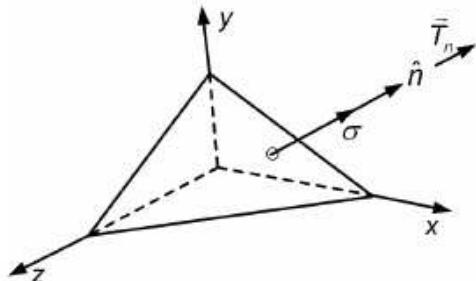
$$\begin{aligned}
 \sigma_{x'} &= \sigma_x \cos^2 \theta + \sigma_y \sin^2 \theta + 2\tau_{xy} \sin \theta \cos \theta & \begin{bmatrix} \sigma_x & \tau_{xy} & 0 \\ \tau_{xy} & \sigma_y & 0 \\ 0 & 0 & 0 \end{bmatrix} \\
 \sigma_{y'} &= \sigma_x \sin^2 \theta + \sigma_y \cos^2 \theta - 2\tau_{xy} \sin \theta \cos \theta \\
 \tau_{x'y'} &= -(\sigma_x - \sigma_y) \sin \theta \cos \theta + \tau_{xy} (\cos^2 \theta - \sin^2 \theta)
 \end{aligned}$$

Principal stresses

$$n_x, n_y, n_z$$

$$\bar{T}_n = \sigma \hat{n} = \sigma (n_x \hat{i} + n_y \hat{j} + n_z \hat{k})$$

$$\bar{T}_n = T_{nx} \hat{i} + T_{ny} \hat{j} + T_{nz} \hat{k}$$



$$T_{nx} = \sigma_x n_x + \tau_{yx} n_y + \tau_{zx} n_z$$

$$T_{ny} = \tau_{xy} n_x + \sigma_y n_y + \tau_{zy} n_z$$

$$T_{nz} = \tau_{xz} n_x + \tau_{yz} n_y + \sigma_z n_z$$

$$T n_x = \sigma n_x \quad | \quad T n_y = \sigma n_y \quad | \quad T n_z = \sigma n_z$$

$$\left. \begin{array}{l}
 (\sigma_x - \sigma) n_x + \tau_{yx} n_y + \tau_{zx} n_z = 0 \\
 \tau_{yx} n_x + (\sigma_y - \sigma) n_y + \tau_{zy} n_z = 0 \\
 \tau_{xz} n_x + \tau_{yz} n_y + (\sigma_z - \sigma) n_z = 0
 \end{array} \right\} \text{Syst. of linear homog. eqns.}$$

$$n_x = n_y = n_z = 0: n_x^2 + n_y^2 + n_z^2 = 1$$

$$\begin{bmatrix} \sigma_x - \sigma & \tau_{xy} & \tau_{zx} \\ \tau_{xy} & \sigma_y - \sigma & \tau_{zy} \\ \tau_{zx} & \tau_{yz} & \sigma_z - \sigma \end{bmatrix} \begin{Bmatrix} n_x \\ n_y \\ n_z \end{Bmatrix} = (0)$$

For non trivial solution $| \cdot |$ must be zero.

$$\begin{aligned} \sigma^3 - (\sigma_x + \sigma_y + \sigma_z)\sigma^2 + (\sigma_x\sigma_y + \sigma_y\sigma_z + \sigma_z\sigma_x - \tau_{xy}^2 - \tau_{yz}^2 - \tau_{zx}^2)\sigma \\ - (\sigma_x\sigma_y\sigma_z + 2\tau_{xy}\tau_{yz}\tau_{zx} - \sigma_x\tau_{yz}^2 - \sigma_y\tau_{zx}^2 - \sigma_z\tau_{xy}^2) = 0 \end{aligned}$$

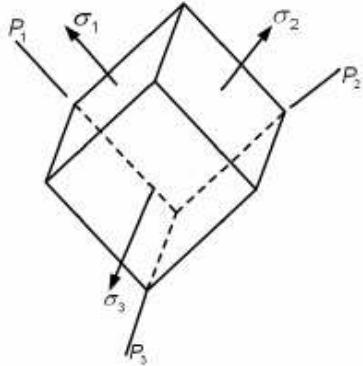
This has 3- real roots $\sigma_1, \sigma_2, \sigma_3$

$$\left. \begin{aligned} (\sigma_x - \sigma_1)n_x + \tau_{yx}n_y + \tau_{zx}n_z = 0 \\ \tau_{yx}n_x + (\sigma_y - \sigma_1)n_y + \tau_{zy}n_z = 0 \end{aligned} \right\}$$

and $n_x^2 + n_y^2 + n_z^2 = 1$

$$\Rightarrow n_x, n_y, n_z \rightarrow \sigma_1$$

$$\sigma_1 > \sigma_2 > \sigma_3$$



Stress invariants

$$\sigma^3 - I_1\sigma^2 + I_2\sigma - I_3 = 0 \quad (1)$$

$$\left. \begin{aligned} I_1 &= \sigma_x + \sigma_y + \sigma_z \\ I_2 &= \sigma_x\sigma_y + \sigma_y\sigma_z + \sigma_z\sigma_x - \tau_{xy}^2 - \tau_{yz}^2 - \tau_{zx}^2 \\ I_3 &= \sigma_x\sigma_y\sigma_z + 2\tau_{xy}\tau_{yz}\tau_{zx} - \sigma_x\tau_{yz}^2 - \sigma_y\tau_{zx}^2 - \sigma_z\tau_{xy}^2 \end{aligned} \right\} \text{stress invariants}$$

$$\sigma^3 - I'_1\sigma^2 + I'_3 = 0$$

$$I'_1 = \sigma_{x'} + \sigma_{y'} + \sigma_{z'} \quad \left| \quad I'_2 = \sigma_{x'}\sigma_{y'} + \sigma_{x'}\sigma_{z'} + \sigma_{y'}\sigma_{z'} - \tau_{x'y'}^2 - \tau_{y'z'}^2 - \tau_{x'z'}^2 \right.$$

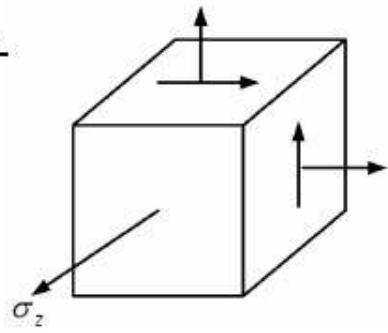
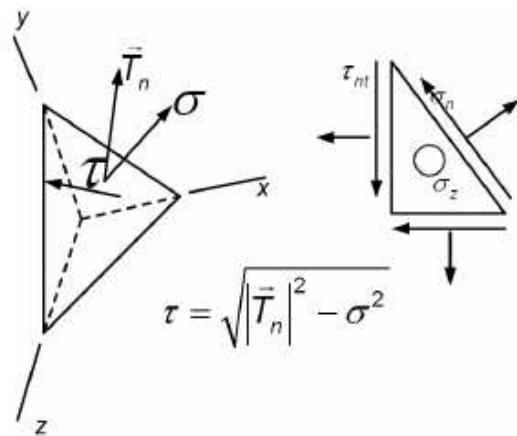
$$I'_1 = I'_1; \quad I'_2 = I'_2; \quad I'_3 = I'_3$$

3D	2D
$I_1 = \sigma_1 + \sigma_2 + \sigma_3$	$I_1 = \sigma_1 + \sigma_2$
$I_2 = \sigma_1\sigma_2 + \sigma_2\sigma_3 + \sigma_3\sigma_1$	$I_2 = \sigma_1\sigma_2$
$I_3 = \sigma_1\sigma_2\sigma_3$	$I_3 = 0$

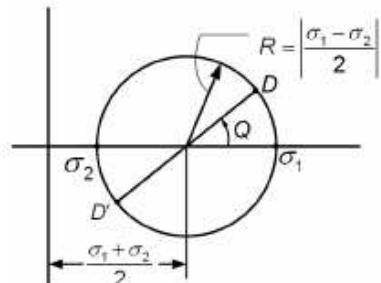
3-D Mohr's circle & principal shear

$$[\sigma_{ij}] = \begin{bmatrix} \sigma_x & \tau_{xy} & 0 \\ \tau_{xy} & \sigma_y & 0 \\ 0 & 0 & \sigma_z \end{bmatrix}$$

y



Once if you know σ_1 and σ_2

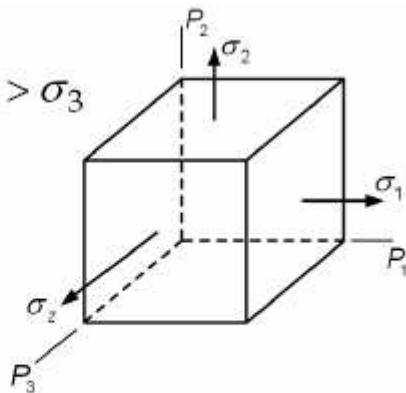


$$\tau_1 = \left| \frac{\sigma_2 - \sigma_3}{2} \right|$$

$$\sigma_{\tau_1} = \frac{\sigma_1 + \sigma_3}{2}$$

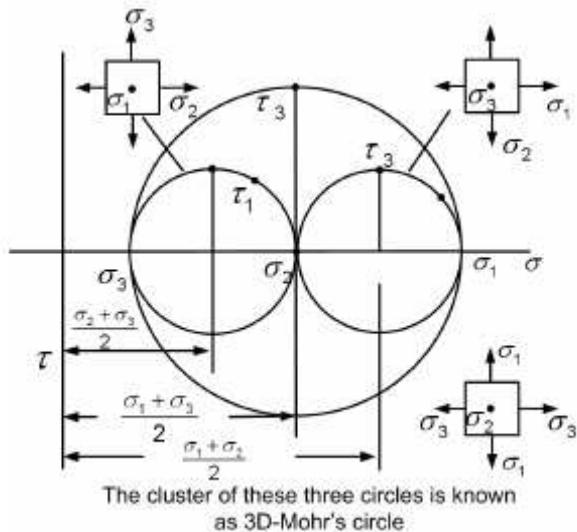
$$\tau_2 = \left| \frac{\sigma_1 - \sigma_3}{2} \right|$$

$$\sigma_{\tau_2} = \frac{\sigma_1 + \sigma_2}{2}$$



$$\tau_3 = \left| \frac{\sigma_1 - \sigma_2}{2} \right| \quad \tau_{max} = \max \left| \frac{\sigma_1 - \sigma_2}{2}, \frac{\sigma_2 - \sigma_3}{2}, \frac{\sigma_3 - \sigma_1}{2} \right|$$

$$\sigma_{\tau_3} = \frac{\sigma_1 - \sigma_2}{2}$$



Problem:

The state of stress at a point is given by

$$\sigma_x = 100 \text{ MPa}, \sigma_y = -40 \text{ MPa}, \sigma_z = 80 \text{ MPa} \text{ and}$$

$$\tau_{xy} = \tau_{yz} = \tau_{zx} = 0$$

Determine in plane max shear stresses and maximum shear stress at that point.

$$\sigma_1 = 100 \text{ MPa}, \sigma_2 = 80 \text{ MPa}, \sigma_3 = -40 \text{ MPa}$$

$$\tau_{12} = \frac{\sigma_1 - \sigma_2}{2} = \frac{100 - 80}{2} = 10 \text{ MPa}$$

$$\tau_{13} = \frac{\sigma_1 - \sigma_3}{2} = \frac{100 + 40}{2} = 70 \text{ MPa}$$

$$\tau_{23} = \frac{\sigma_2 - \sigma_3}{2} = \frac{80 + 40}{2} = 60 \text{ MPa}$$

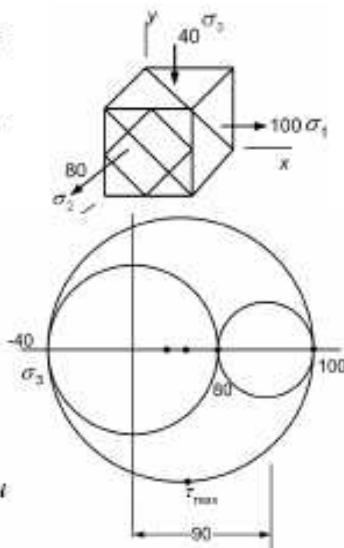
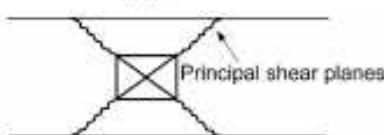
$$\sigma_{12} = \frac{\sigma_1 + \sigma_2}{2} = 90$$

$$\sigma_{13} = 30 \text{ MPa}$$

$$\sigma_{23} = 20 \text{ MPa}$$

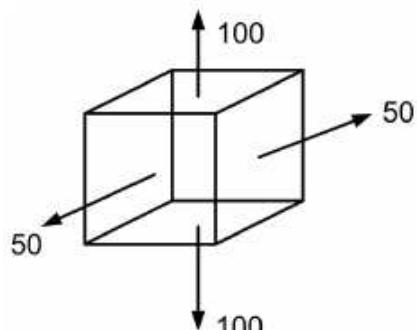
$$\tau_{\max} = \max |\tau_{12}, \tau_{13}, \tau_{23}|$$

$$\tau_{\max} = 70 \text{ MPa} \text{ This occurs at } \sigma_1 = 100 \text{ MPa}$$



Problem

At a point in a component, the state of stress is as shown. Determine maximum shear stress.



Solution:

$$[\sigma_{ij}] = \begin{bmatrix} 100 & 0 \\ 0 & 50 \end{bmatrix} \text{ - plane stress problem}$$

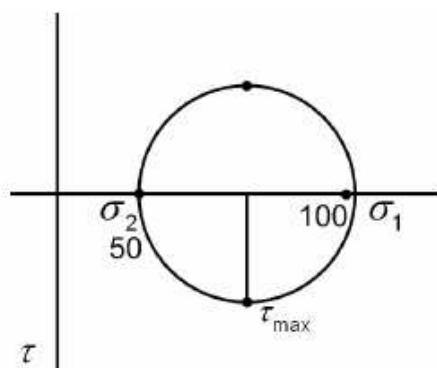
$$\text{We can also write the matrix as } [a_{ij}] = \begin{bmatrix} 100 & 0 & 0 \\ 0 & 50 & 0 \\ 0 & 0 & 0 \end{bmatrix}$$

$$\sigma_1 = 100$$

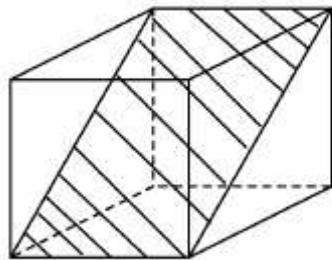
$$\sigma_2 = 50$$

$$\left| \frac{\sigma_1 - \sigma_2}{2} \right| = \frac{100 - 50}{2} = 25$$

$$\tau_{max} = 25 \text{ MPa}$$



(1)



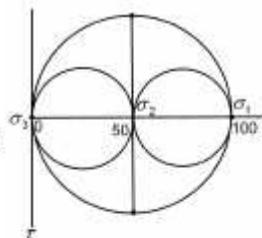
This interpretation is wrong

(2)

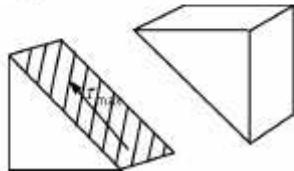
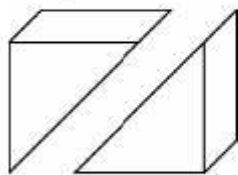
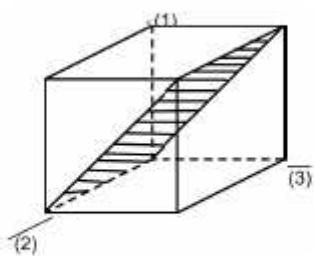
Now with $\sigma_1 = 100, \sigma_2 = 50, \sigma_3 = 0$

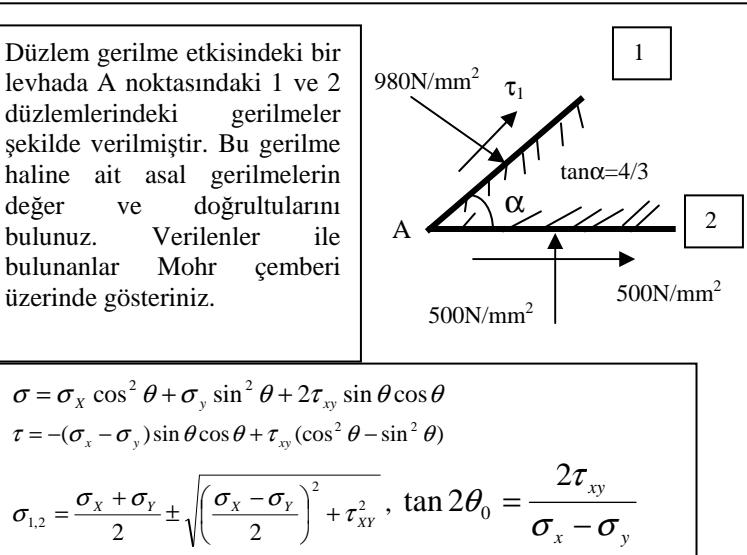
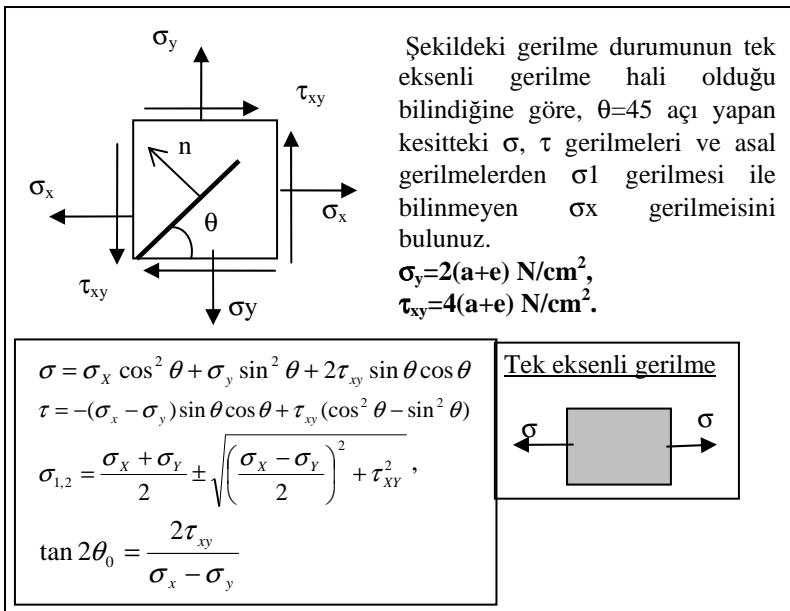
$$\tau_{max} = \left| \frac{\sigma_1 - \sigma_3}{2} \right| = 50 \text{ MPa}$$

Occurs in the plane 1-3 instead of 1-2



(2)





- Üçeksenli gerilme halinde $\sigma_1 + \sigma_2 + \sigma_3$ değeri hangisidir
- | | | |
|-----|-----|-----|
| 50 | 100 | 200 |
| 100 | 150 | 300 |
| 200 | 300 | 250 |
- A) 350 B) 450 C) 750
D) 550