

Internal Combustion Engines – MAK 493E

# MIXTURE PREPERATION in CI ENGINES

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## Internal Combustion Engines – MAK 493E

# Mixture Preparation

- Introduction
- Atomization of fuel, sprays
- Fuel systems - in-line systems, distributor systems
- Common-rail systems, unit pump and unit injector systems

# Fuel Atomization

In Diesel engines, fuel-air mixture is prepared as a result of fuel injection into the cylinder at the end of compression stroke (usually before TDC) during a limited time (crank angle) interval.

The purpose of fuel injection is to provide good mixing of air and fuel in the limited time available, by increasing the surface area of the liquid fuel as a result of atomization

Dividing unit fuel volume into droplets of  $100 \times 10^{-6}$  [m] diameter would increase the total surface area by 10,000 times.

# Liquid Atomization

Liquid atomization is effected by internal and external forces.

controlled by **Reynolds number** defining the balance between inertia effects and viscous effects

$$Re = \rho u D / \mu$$

# Atomization at Low Speeds

**Rayleigh regime** : internal effects are dominant

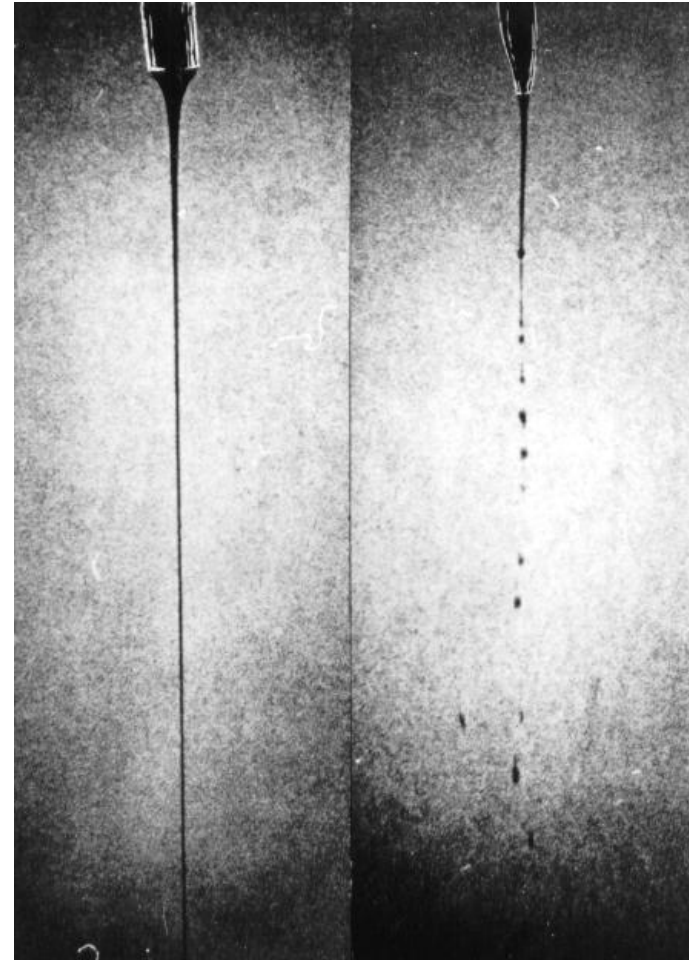
surface tension, fuel density,  
liquid column diameter

Any disturbance produces break-up of liquid column

**Weber** : indicated the importance of viscous effects

Heavy fuel, (left) 5000 cS

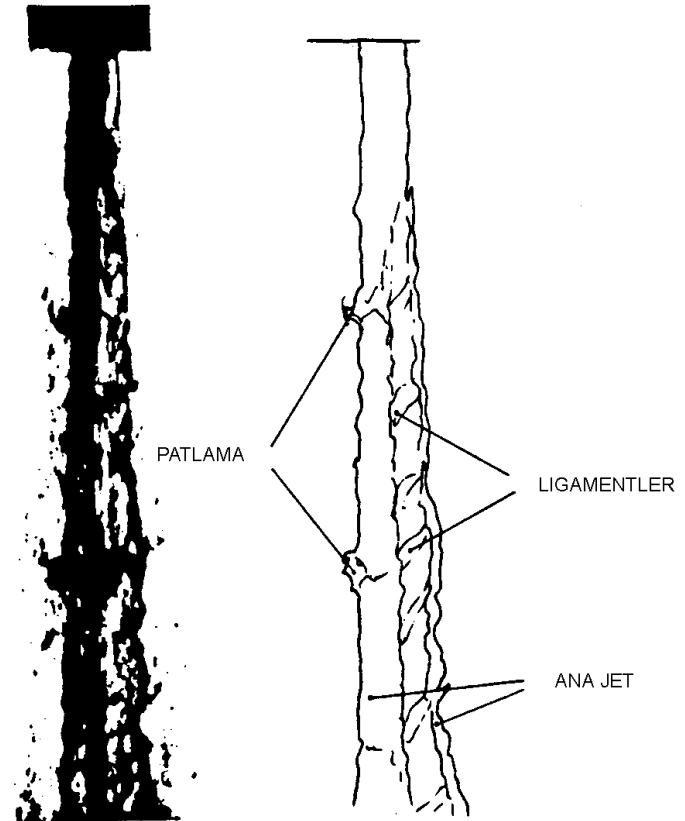
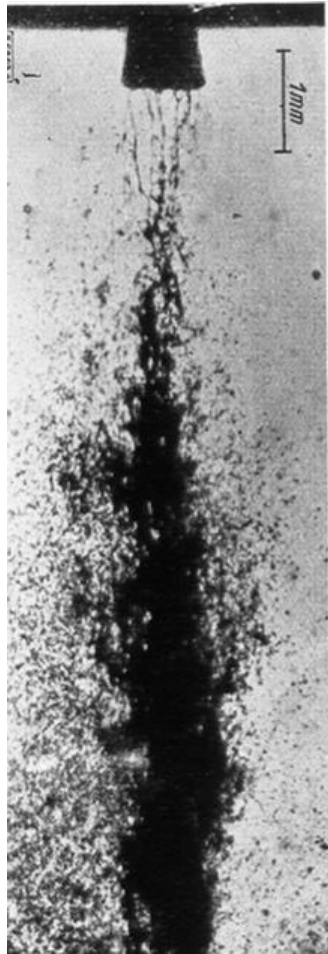
Diesel fuel (right) 6 cS



# Atomization at High Speeds



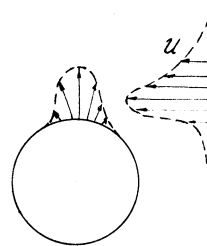
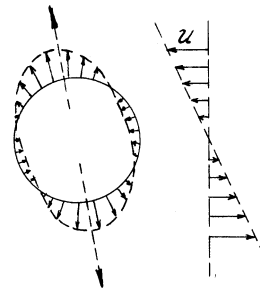
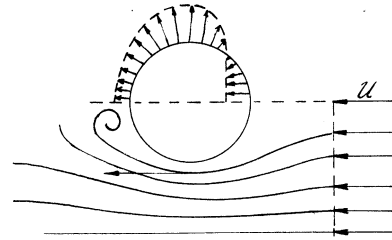
# Atomization at High Speeds



# Secondary Atomization

Weber number

$$W_e = \frac{\rho_h u^2 D}{\sigma}$$



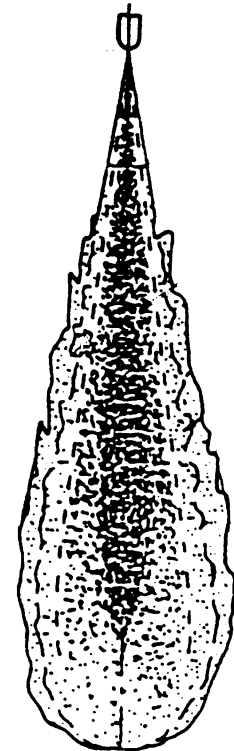


# Spray Structure

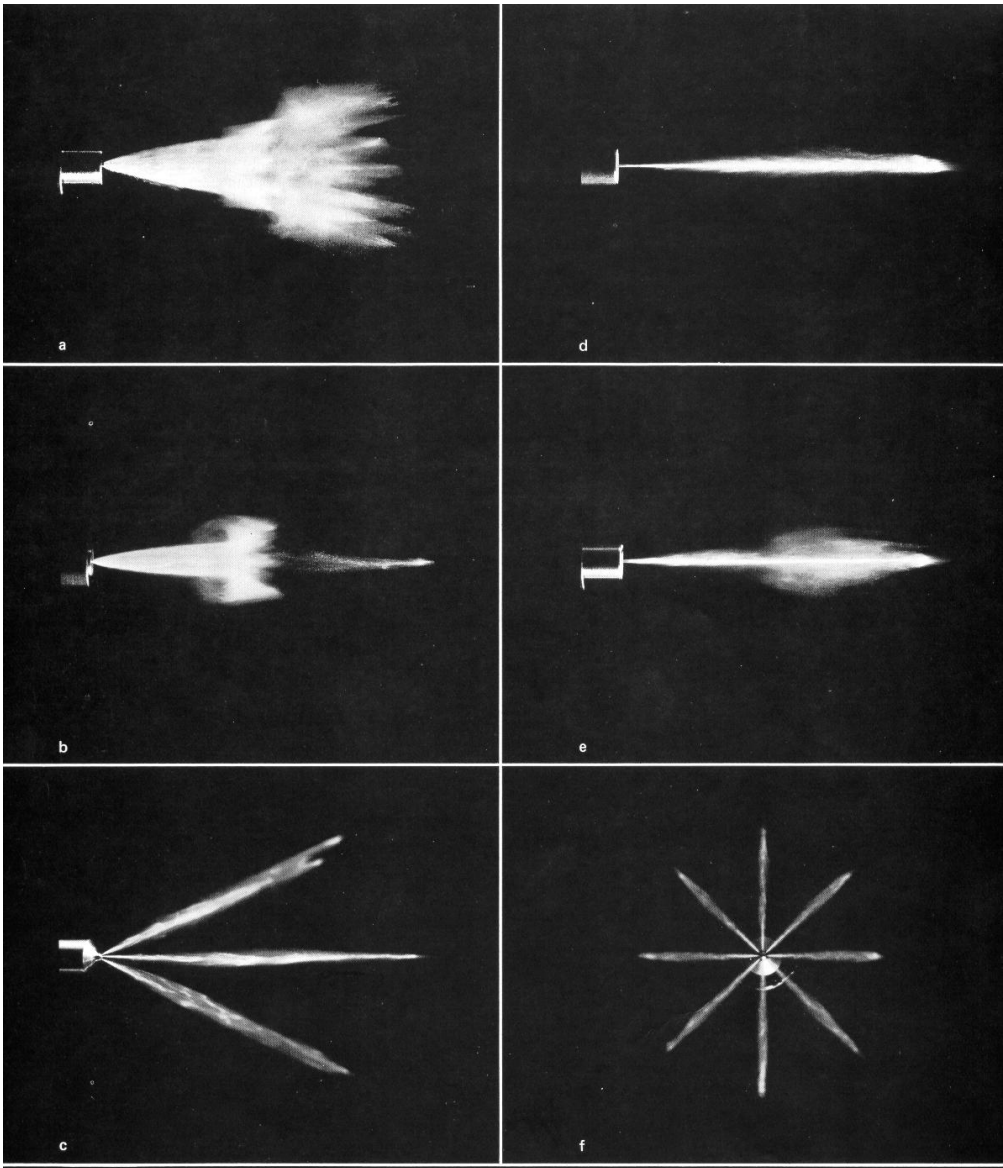
In diffusion flames, combustion is controlled by the **mixing rate of the fuel and air**.

The **local conditions** in the combustion chamber such as the air-fuel ratio, temperature, pressure control the ignition of the fuel and the combustion process. Heat and mass transfer in the combustion chamber and the fluid flow (air flow) also effects this process.

Spray structure,  
core  
breakup length  
spray tip penetration



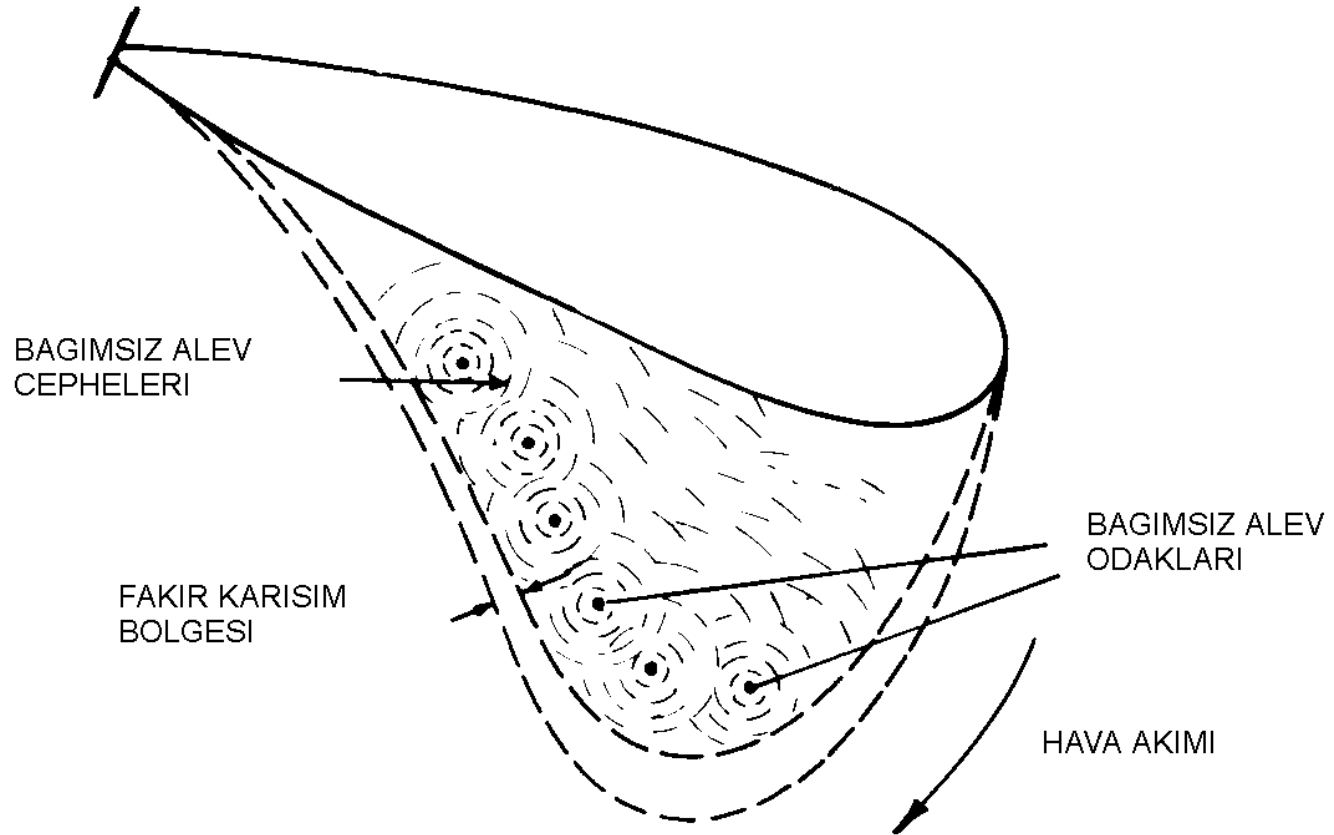
# Fuel Sprays



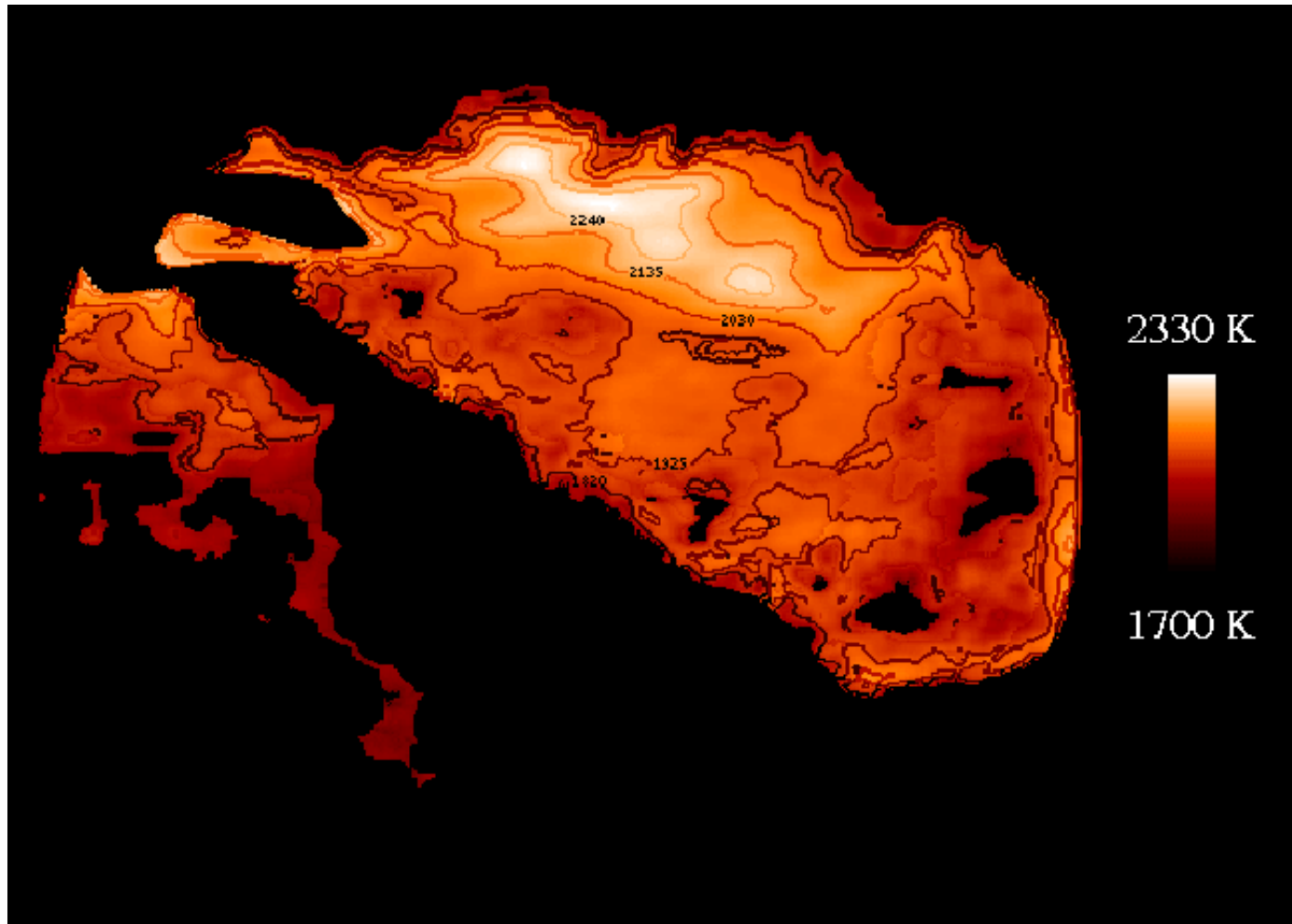
# Fuel Sprays



# Fuel Sprays

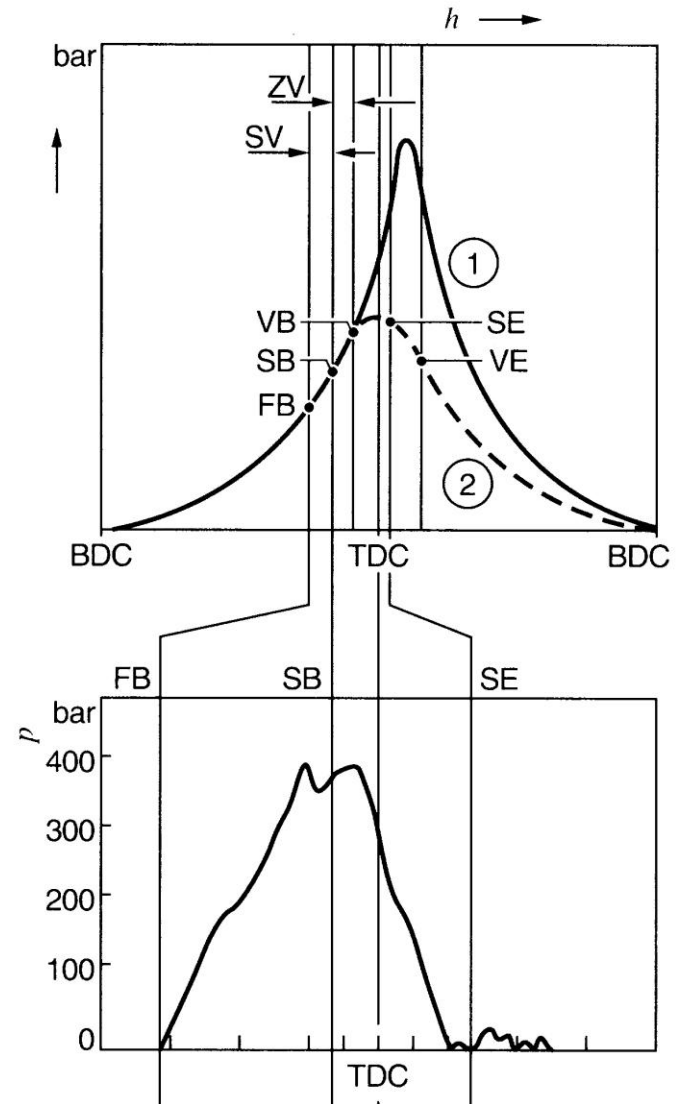
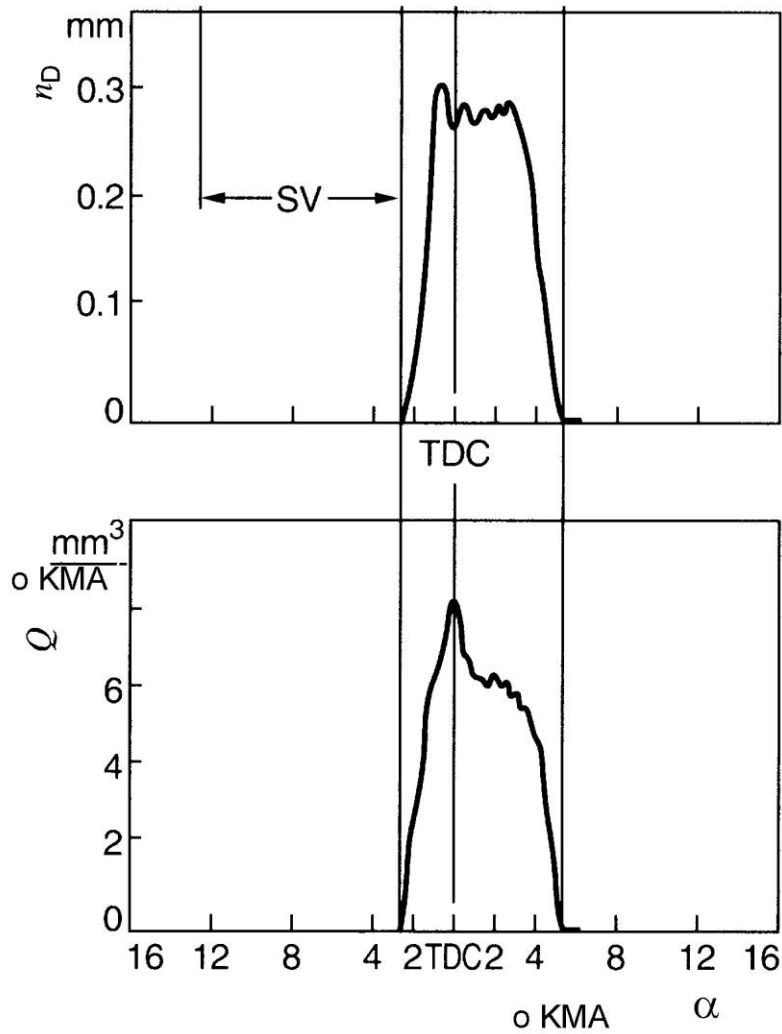


# Temperature Contours



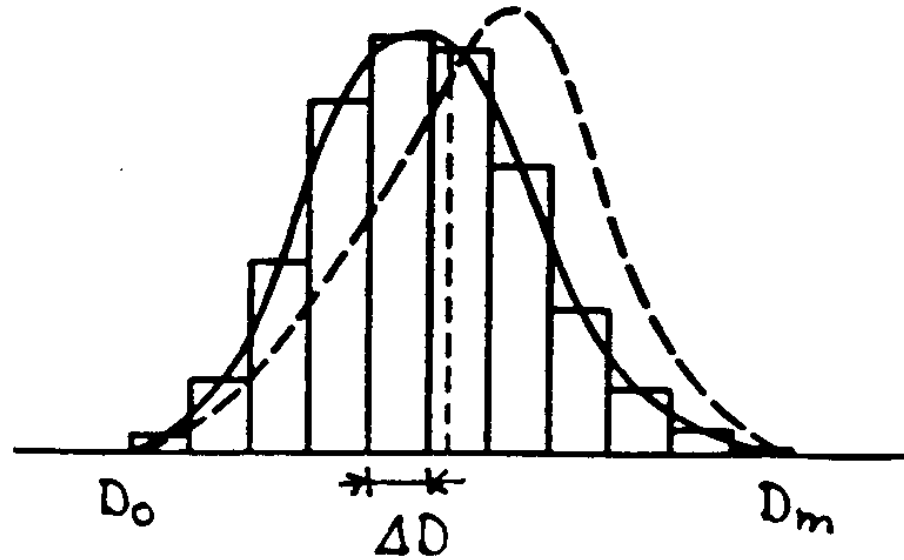
Temperature distribution inside the cylinder

# Diesel Combustion Phases

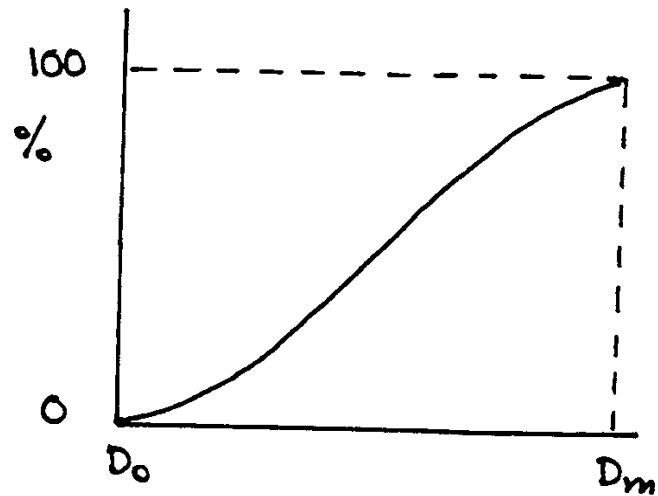
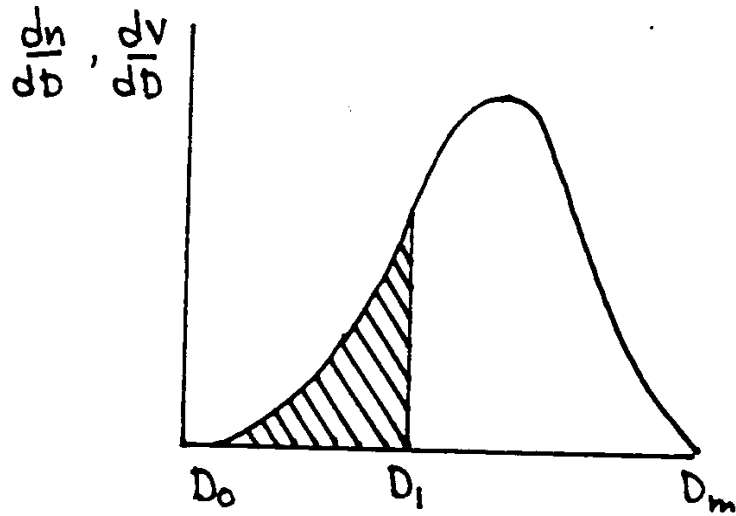


# Droplet Size Distribution

$$\frac{\Delta n}{\Delta D}$$



# Droplet Size Distribution





# Sauter Mean Diameter

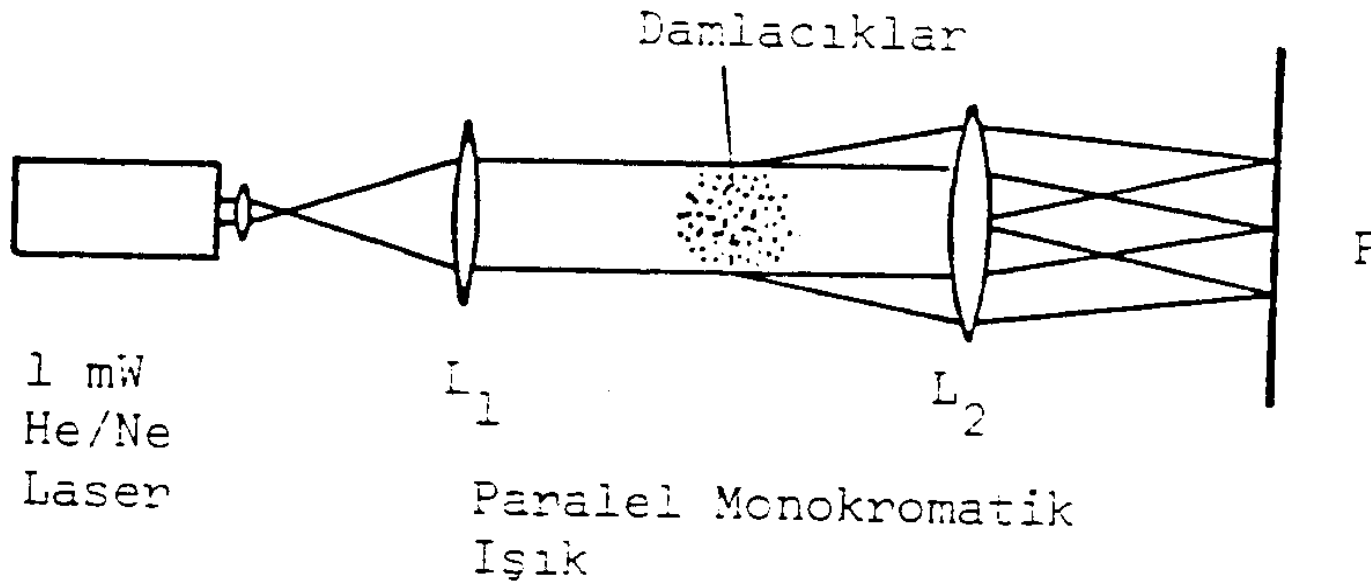
$$D_{qp}^{q-p} = \frac{\int_{D_m}^{D_o} D^q \frac{dn}{dD} dD}{\int_{D_m}^{D_o} D^p \frac{dn}{dD} dD} = \frac{\int_{D_m}^{D_o} D^{q-3} \frac{dV}{dD} dD}{\int_{D_m}^{D_o} D^{p-3} \frac{dV}{dD} dD}$$

$p = 2$  and  $q = 3$  : Sauter Mean Diameter

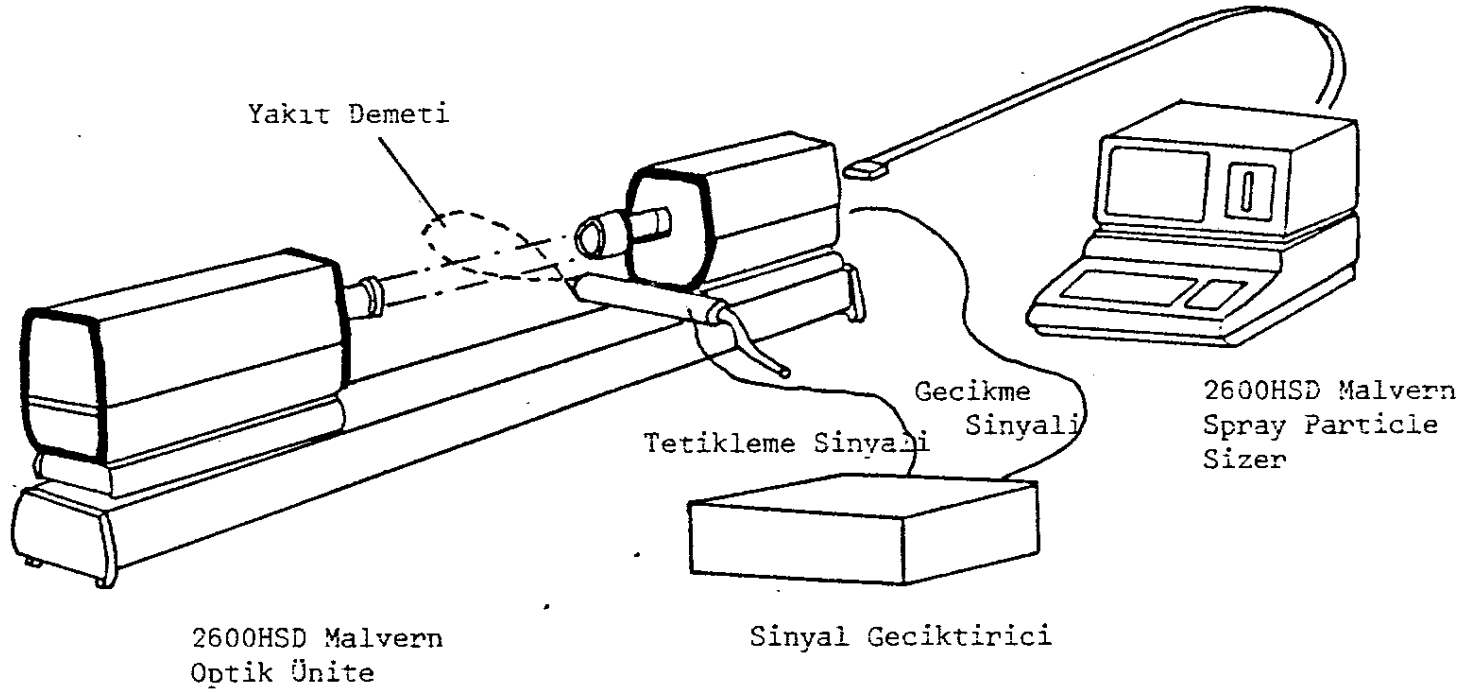
indicates the total Area-to-Volume ratio applicable for the whole spray

$$D_{32} = \frac{\sum D_i^3 n_i}{\sum D_i^2 n_i}$$

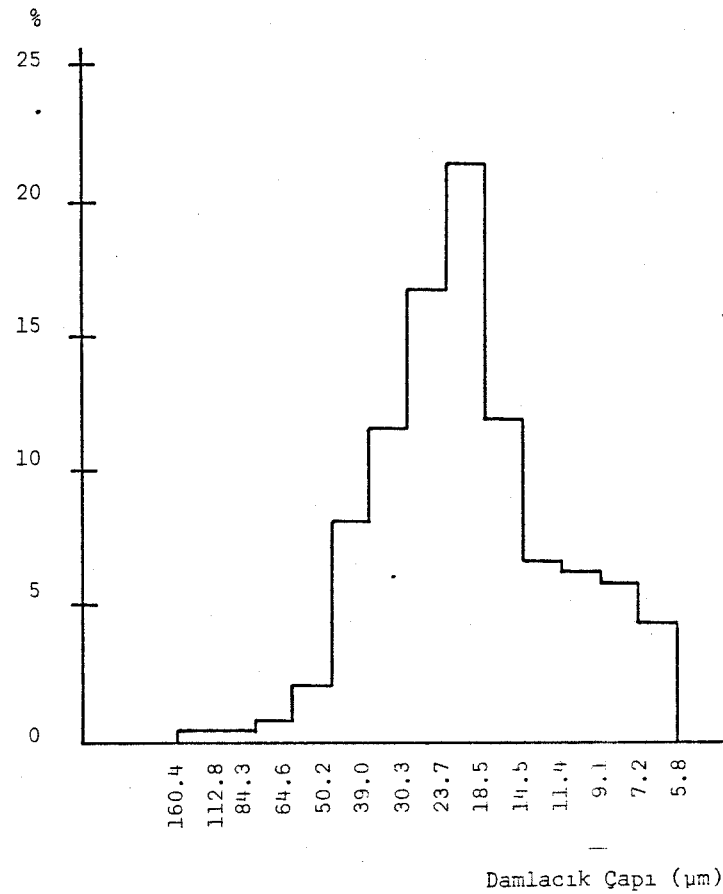
# Droplet Size Measurements



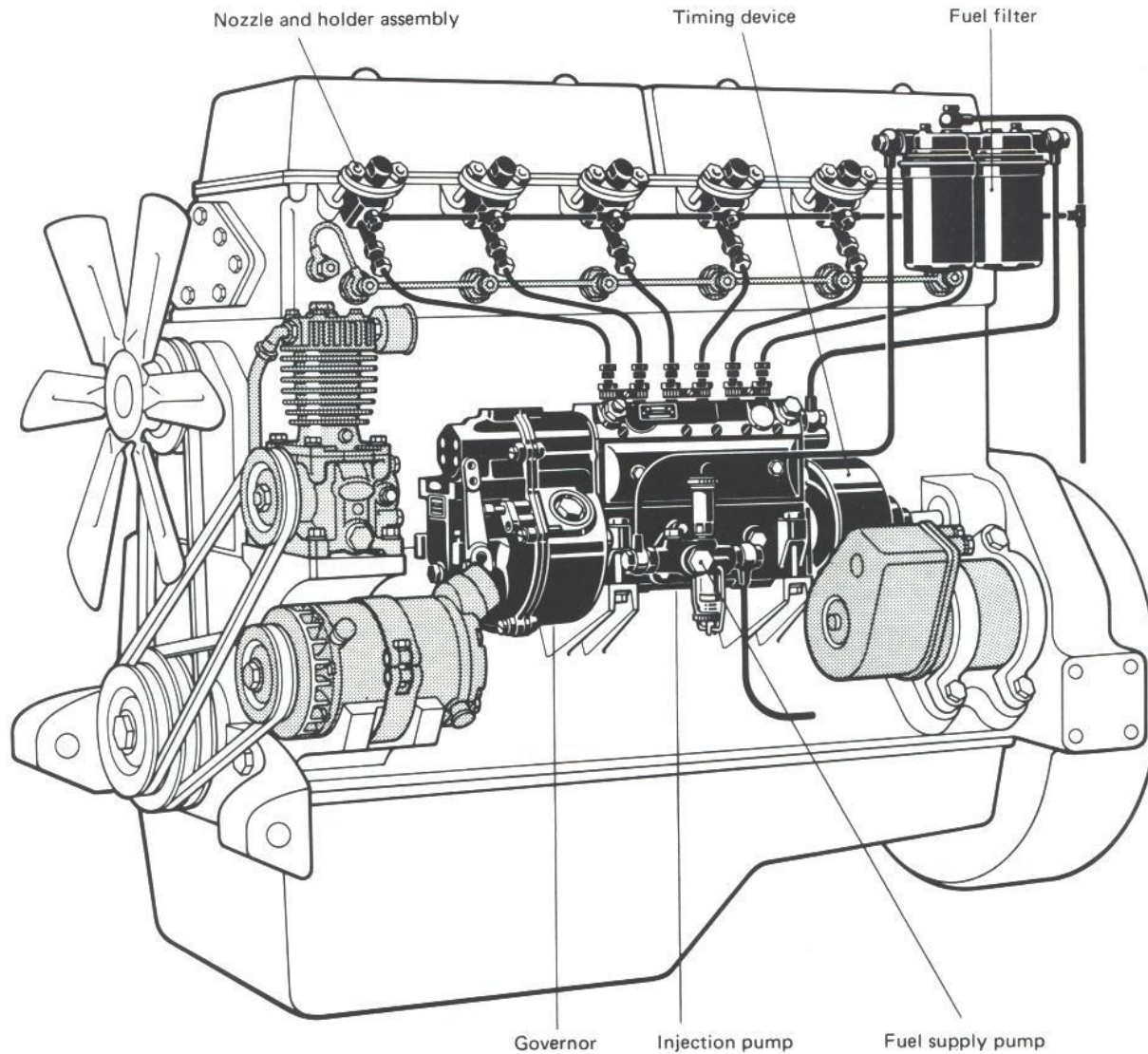
# Droplet Size Measurements



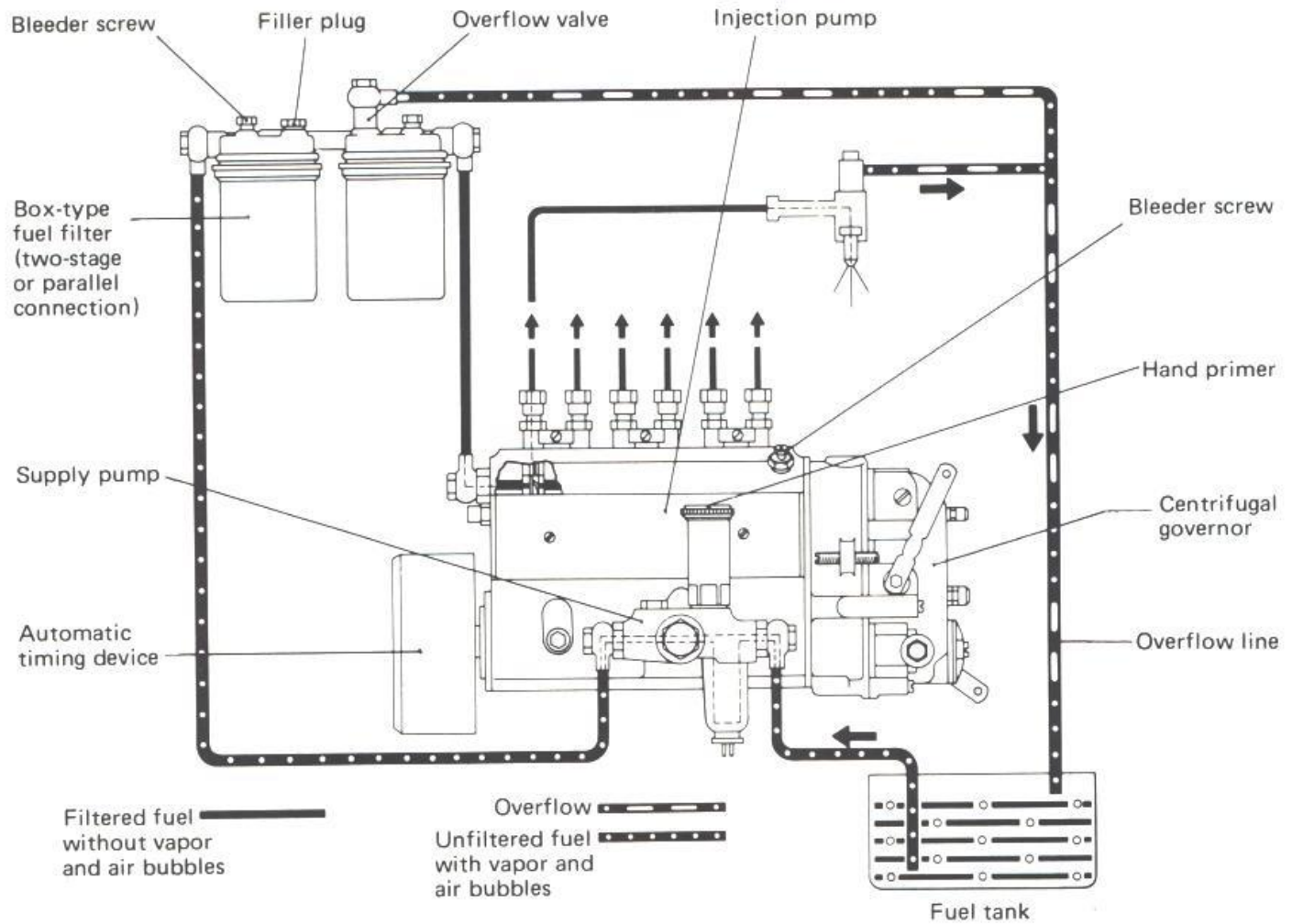
# Droplet Size Measurements



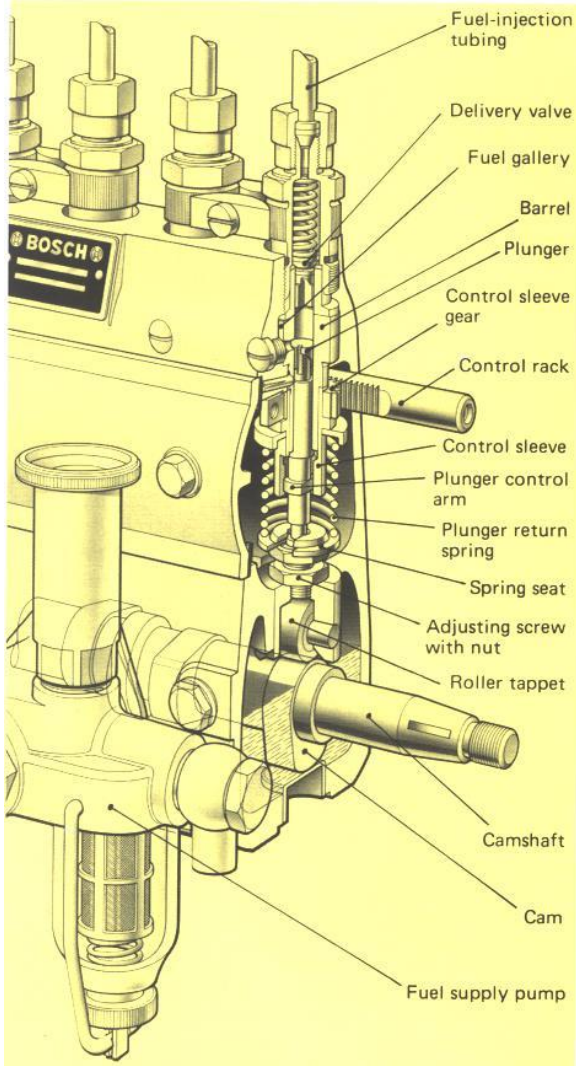
# Diesel Engines



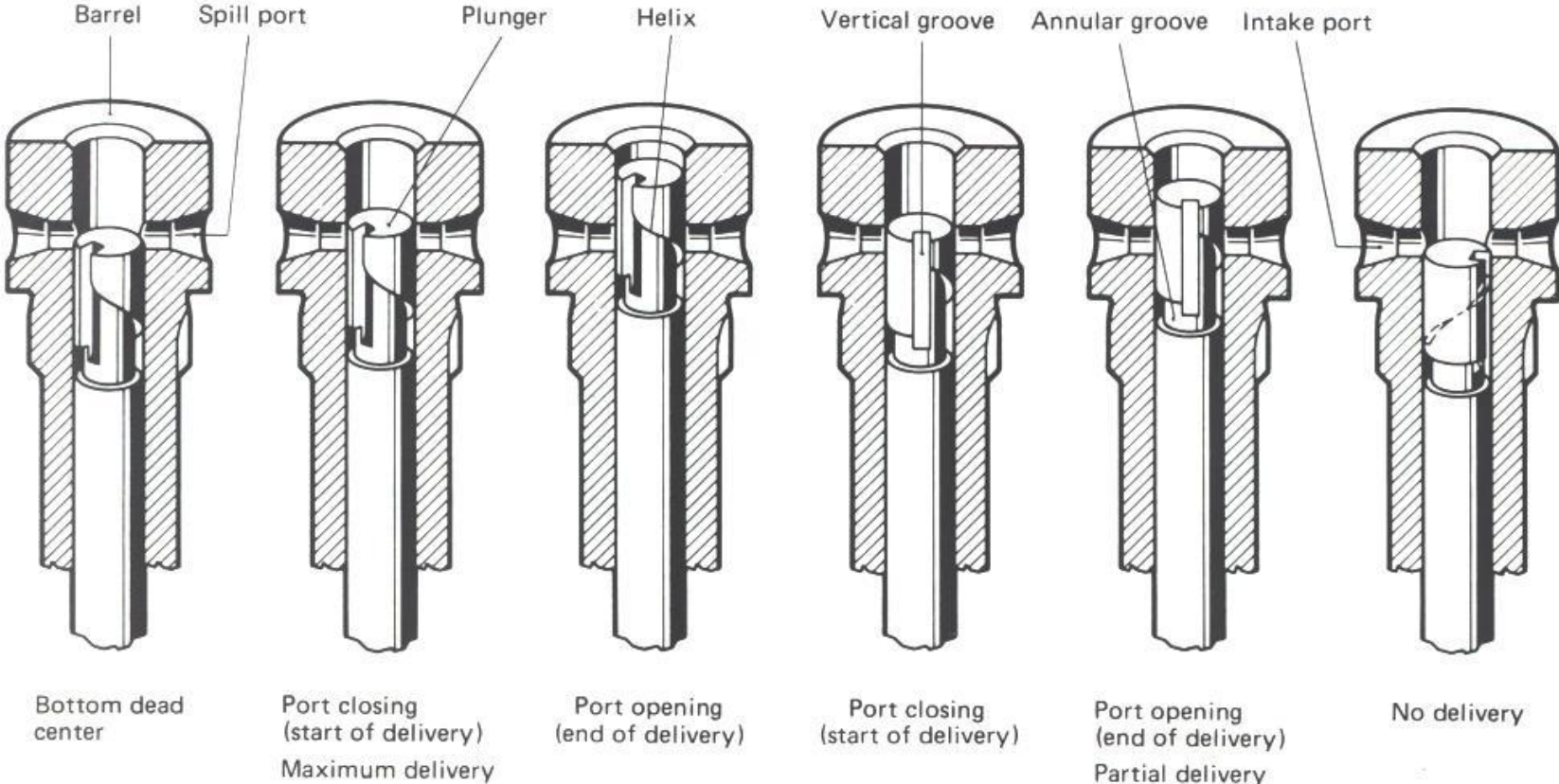
# Fuel system



# Injection Pump – in line system

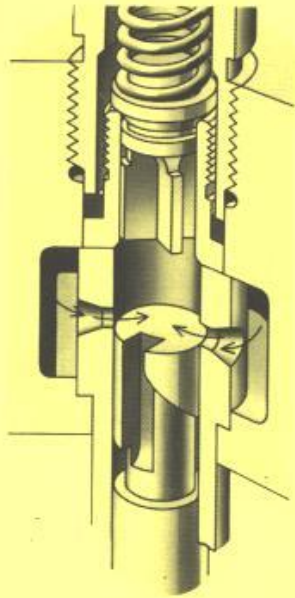


# Injection Pump





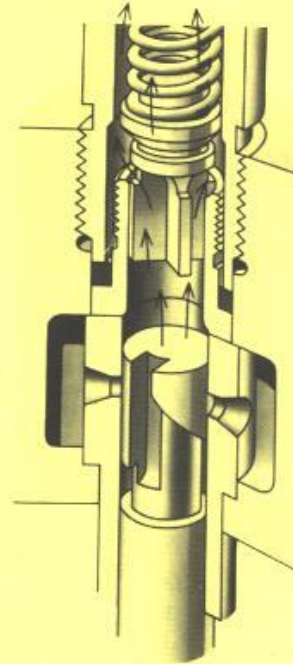
# Injection Pump



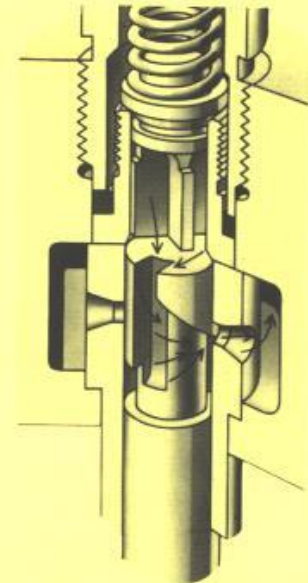
Bottom dead center



Port closing  
(start of delivery)

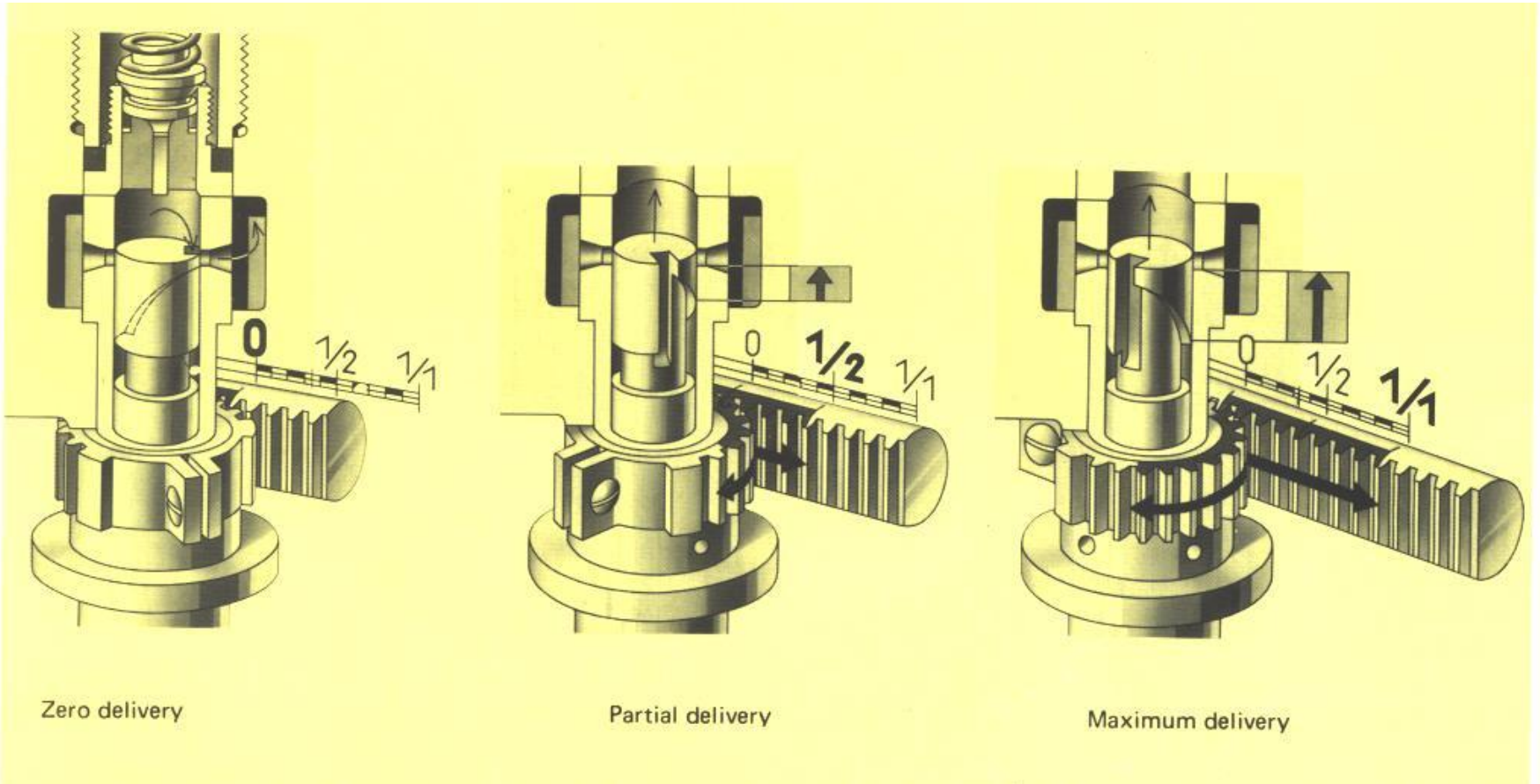


Delivery

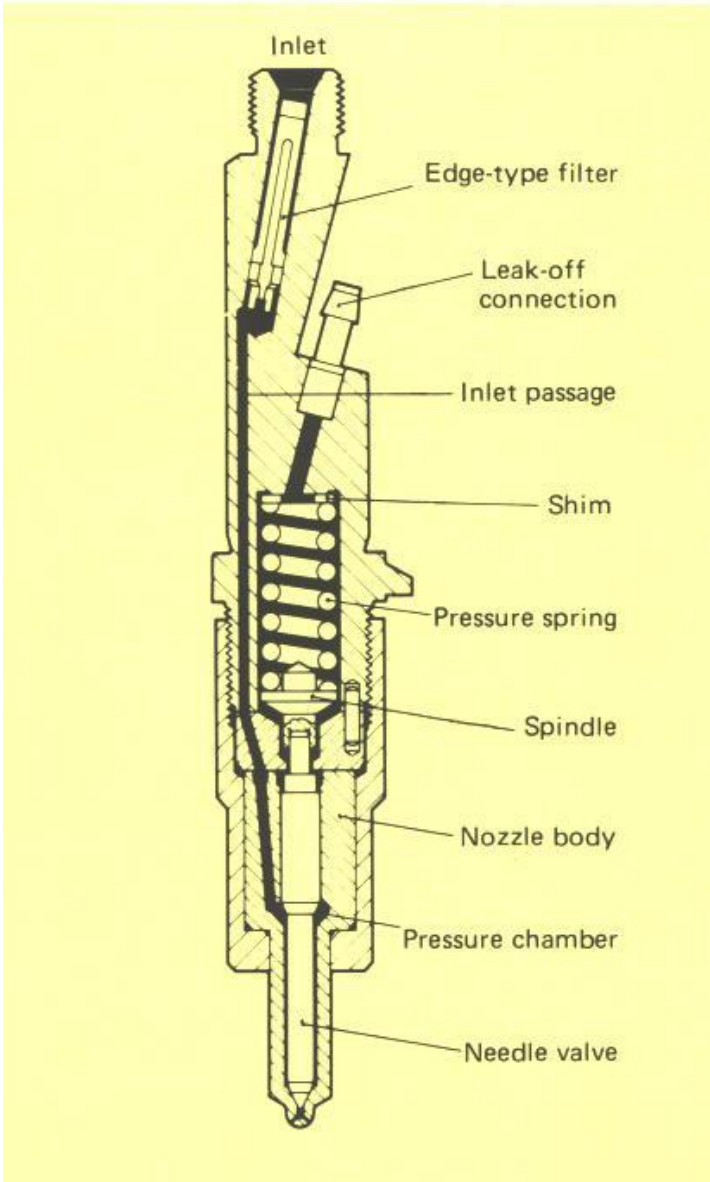


Port opening  
(end of delivery)

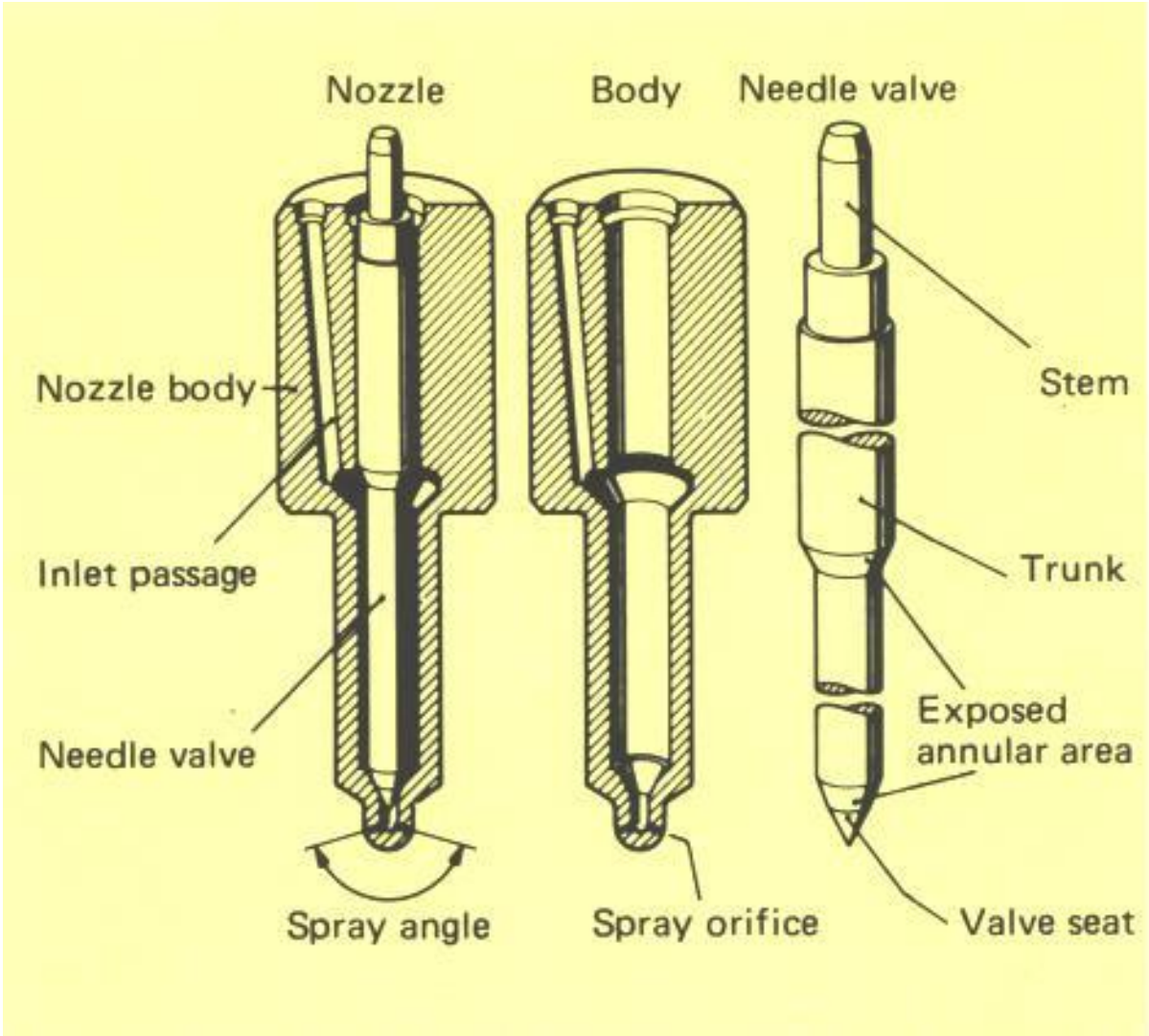
# Injection Pump – fuel metering



# Injector



# Injector



# Nozzles

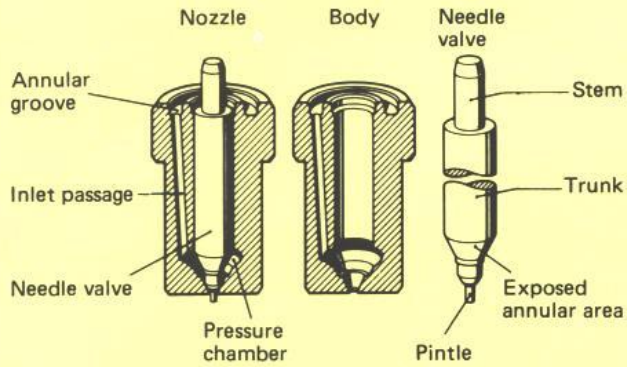
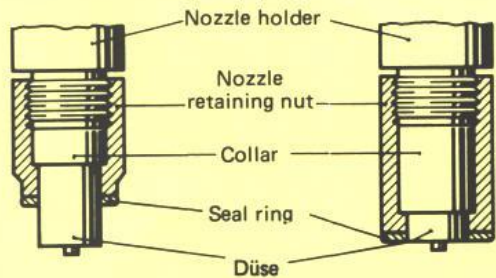


Fig. 48 Pintle nozzle



Standard pintle nozzle

Roller-type nozzle



closed

open

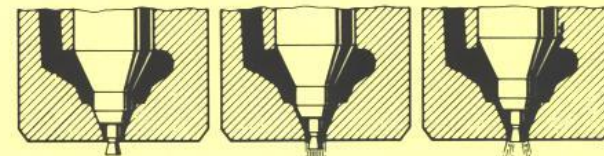
Fig. 50 Pintle nozzle (DN..), with cylindrical pintle (narrow spray)



closed

open

Fig. 51 Pintle nozzle (DN..), with conical pintle (wide spray)

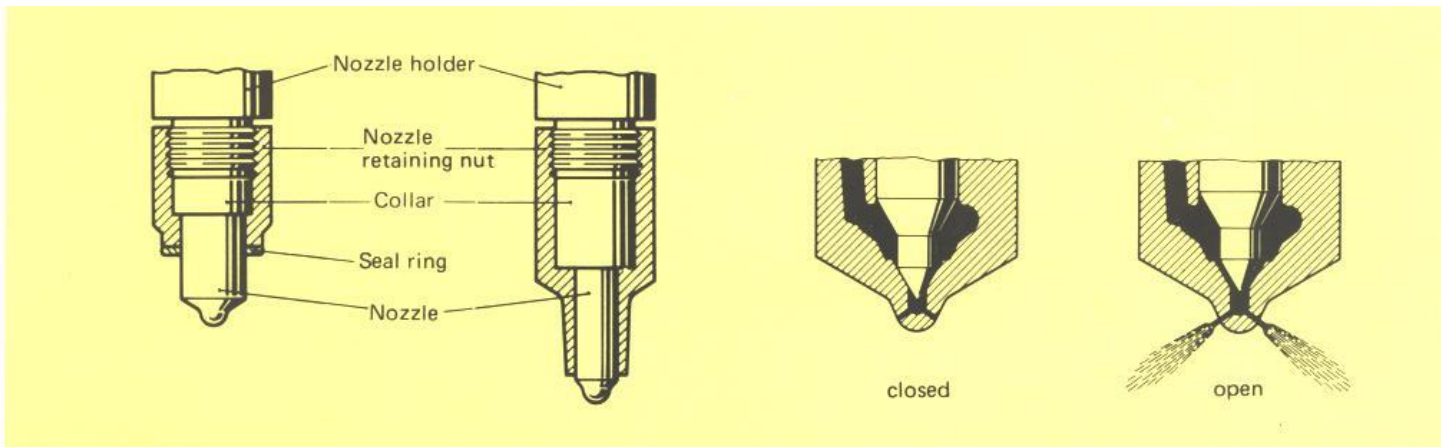
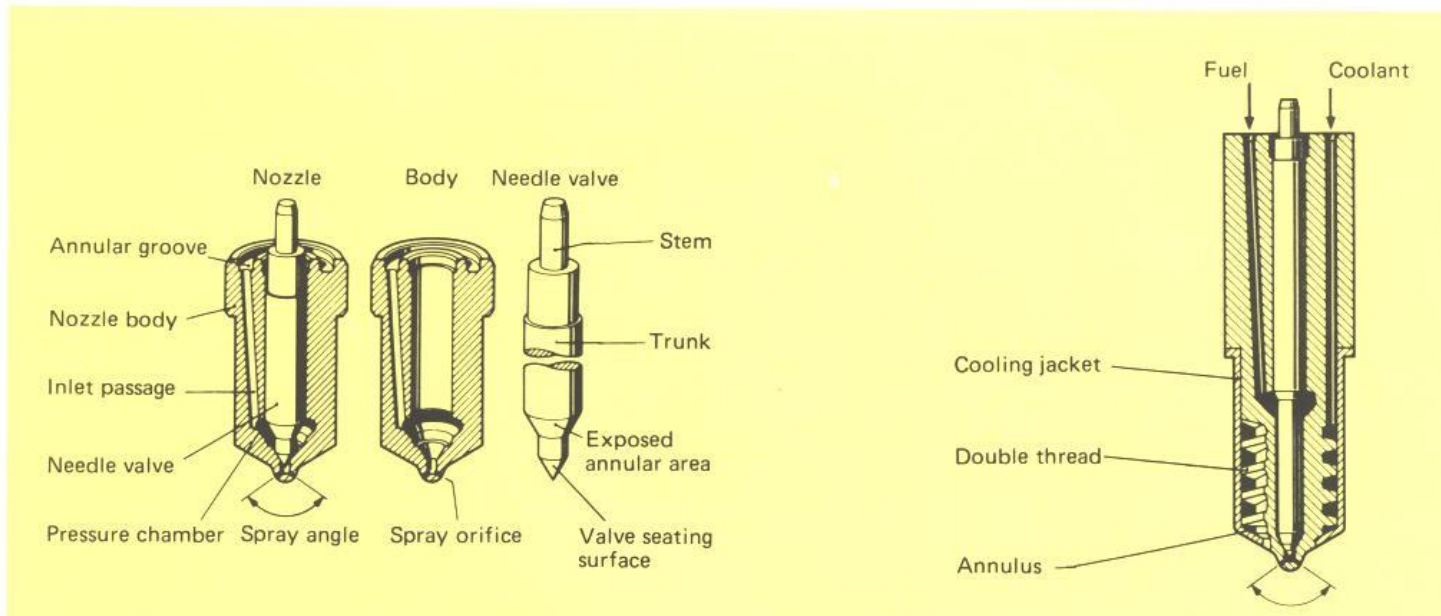


closed

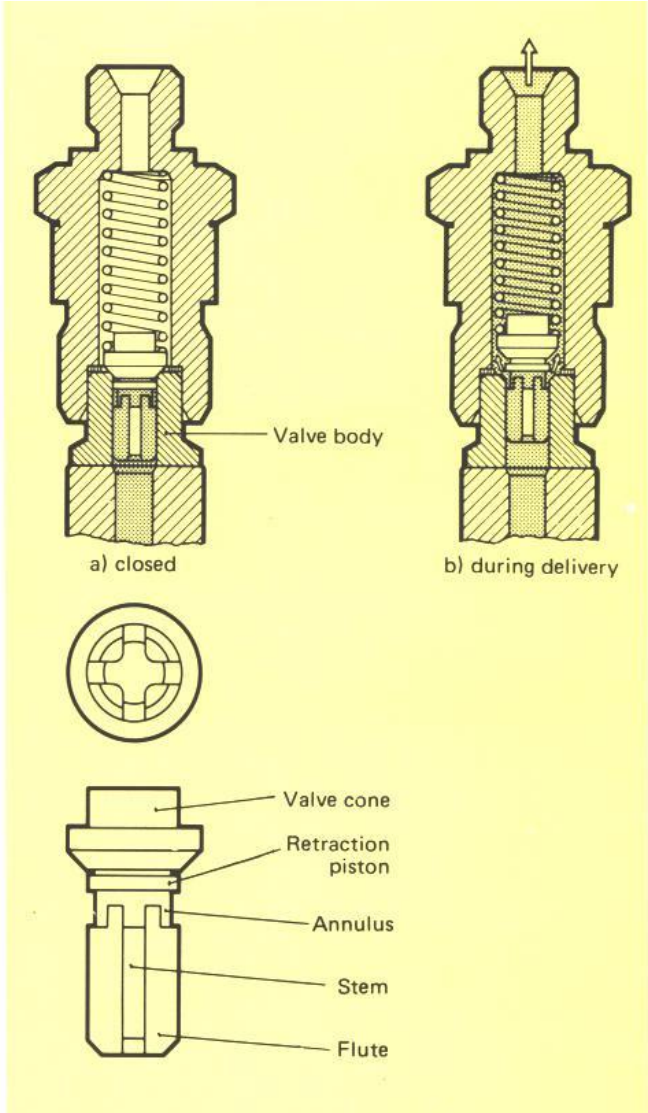
slightly open  
(initial spray)

Fully open  
(main spray)

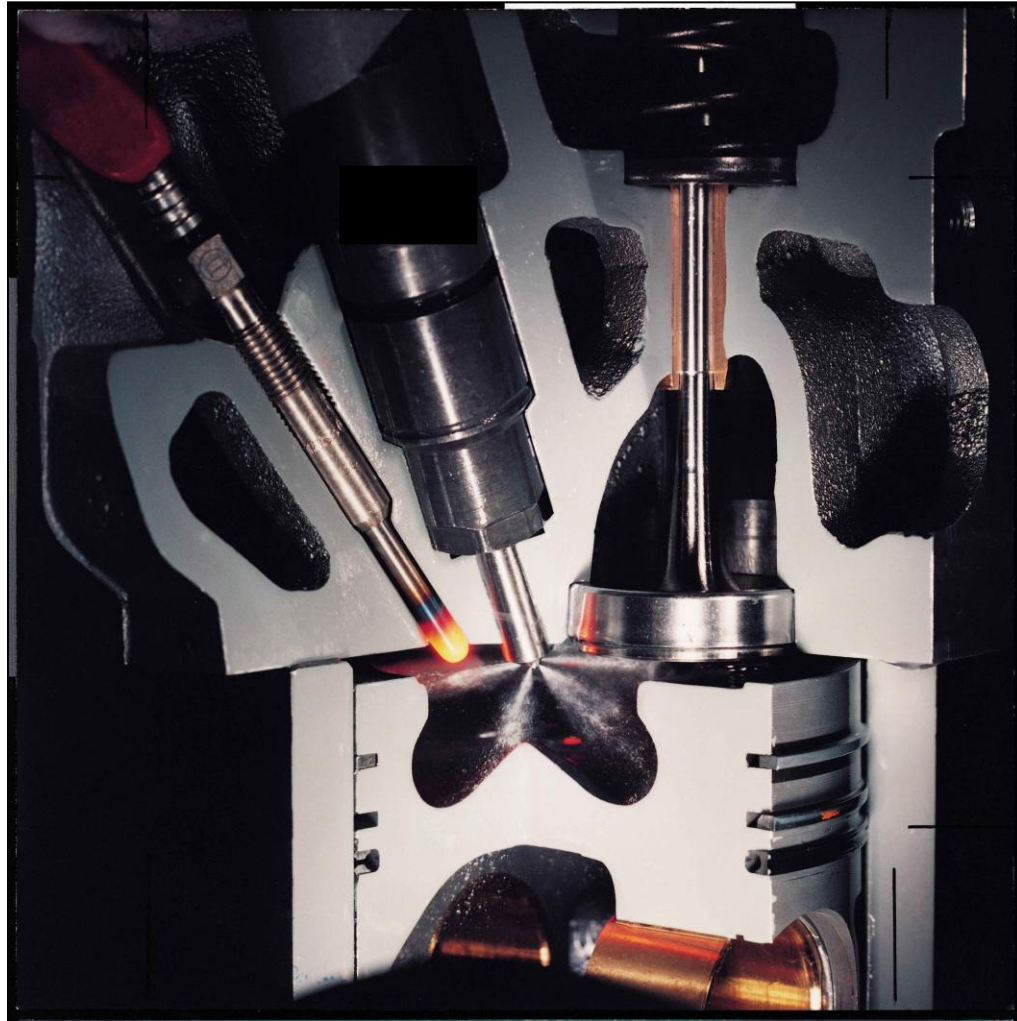
# Nozzles



# Delivery Valve



# Direct Injection Diesel Engine

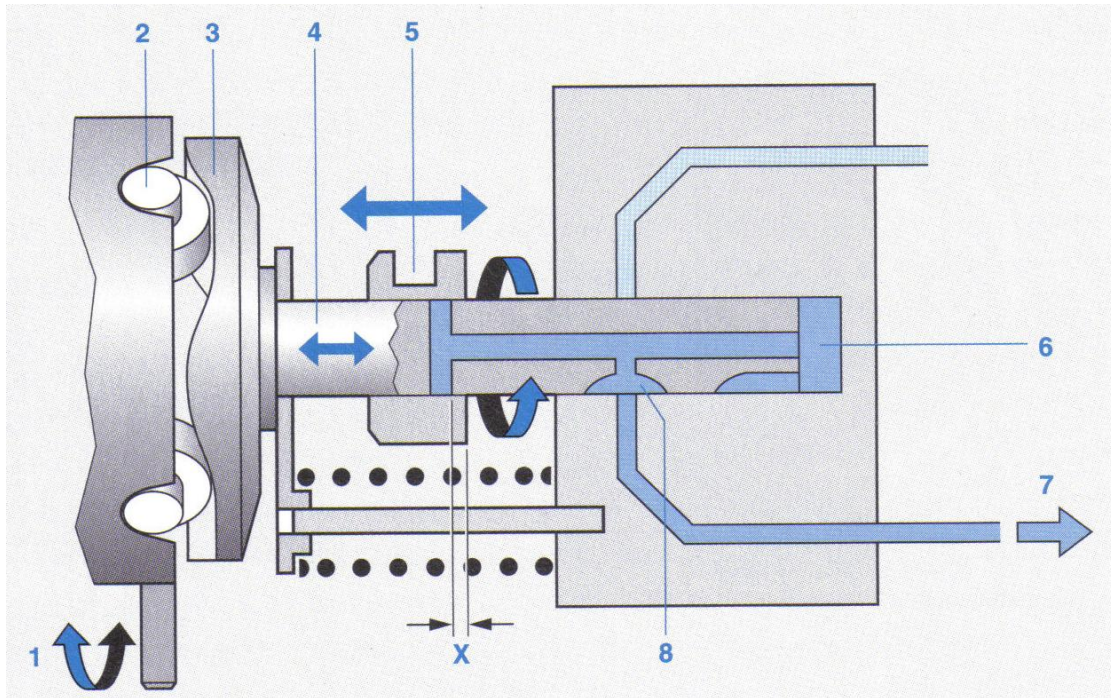




# Indirect Injection Diesel Engine



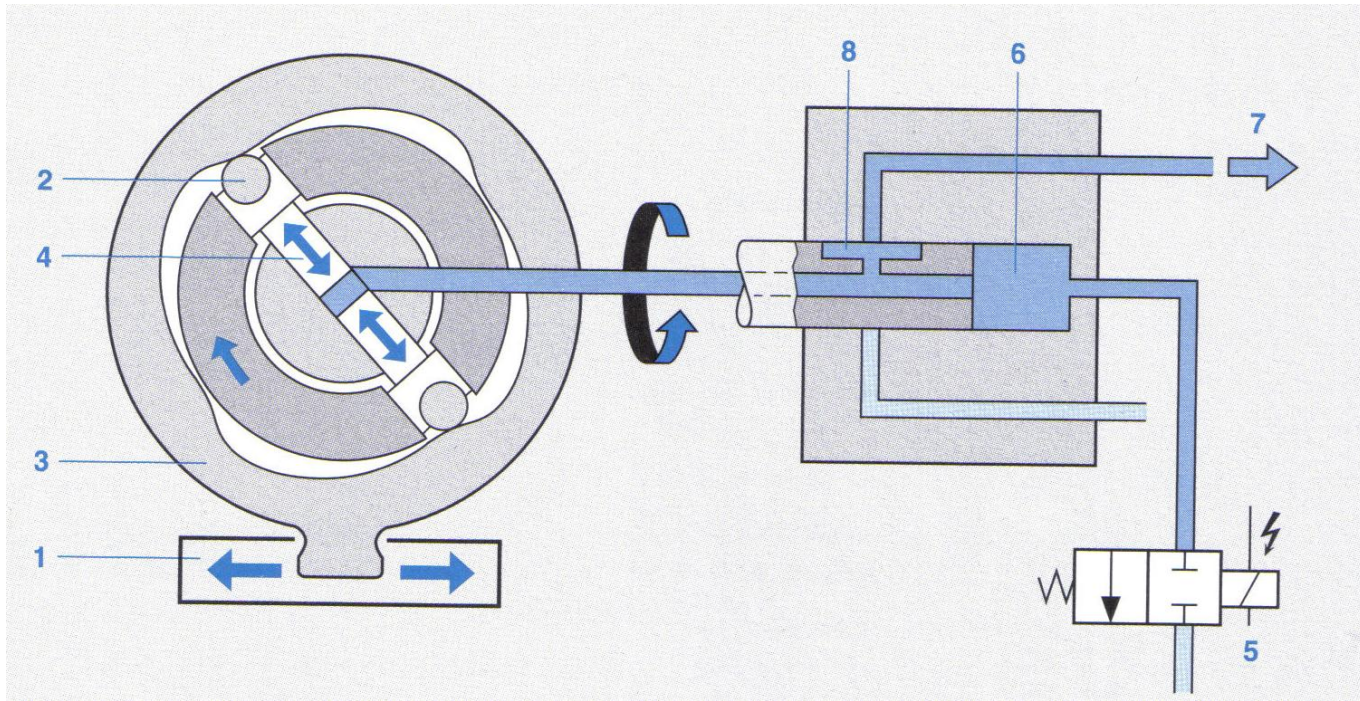
# Distributor-Type Injection Pump



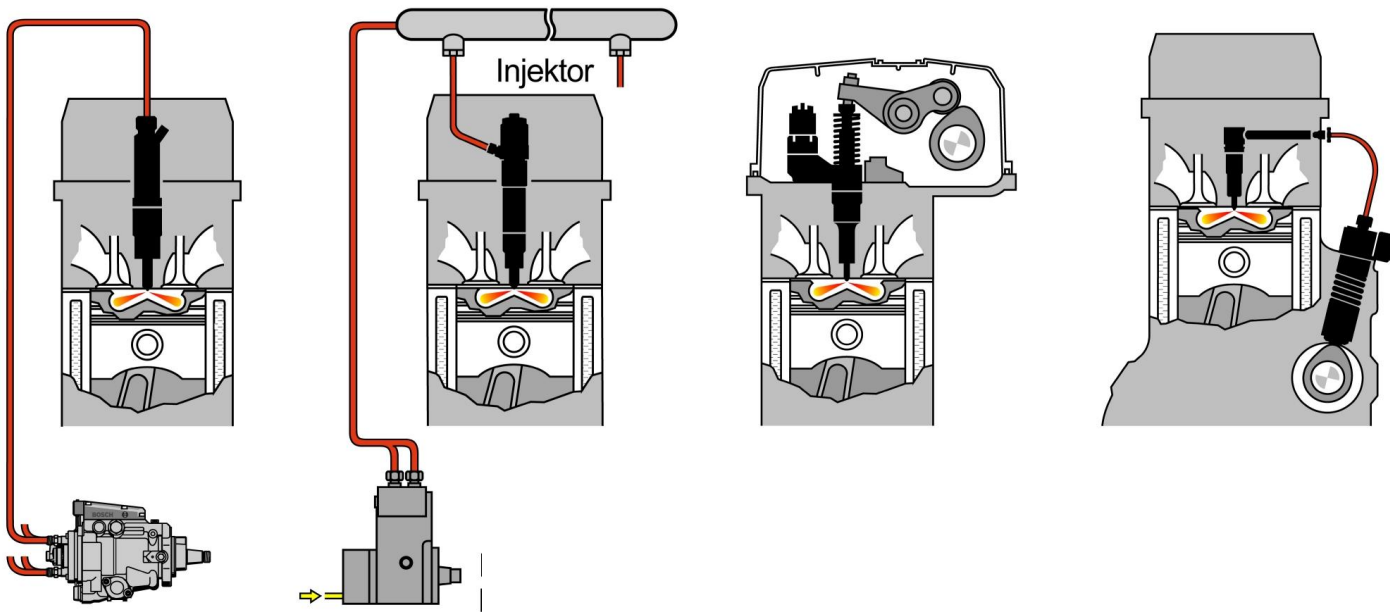
1. Injection Timing Adjustment
2. Roller
3. Cam Plate
4. Axial Piston
5. Control Sleeve
6. High-Pressure Chamber
7. Fuel Outflow to Nozzle
8. Leak-Off
- x. Effective Stroke

Yıldız pompa  
Distribütör tipi pompa

# Distributor-Type Injection Pump



# Diesel Fuel Systems

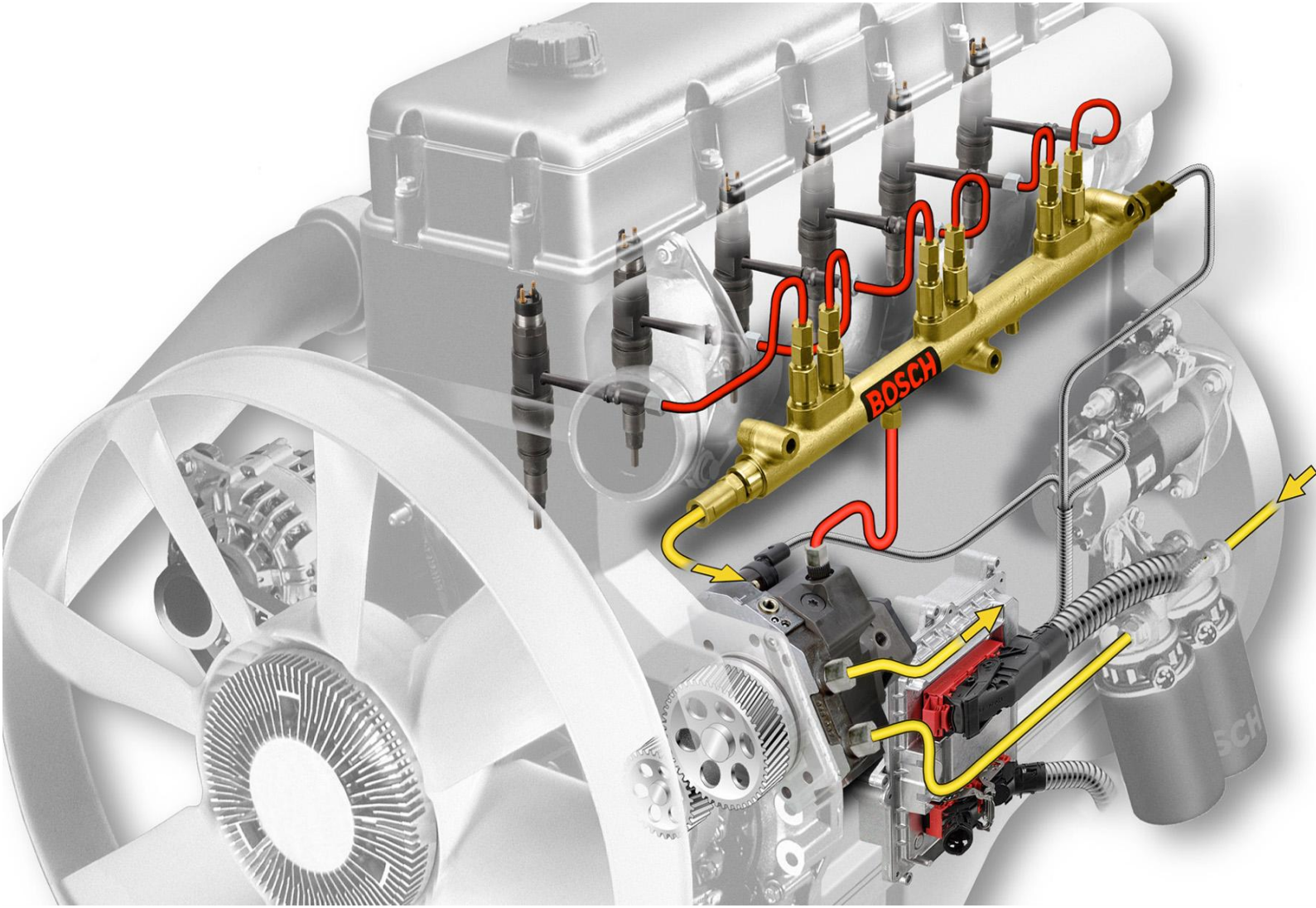


In-line pumps

Common Rail

Unit-injector/pump

# Common-Rail System (Truck engine)



# CR-System Production Location Bursa

## Products



Nozzles



Nozzle Holder Assemblies



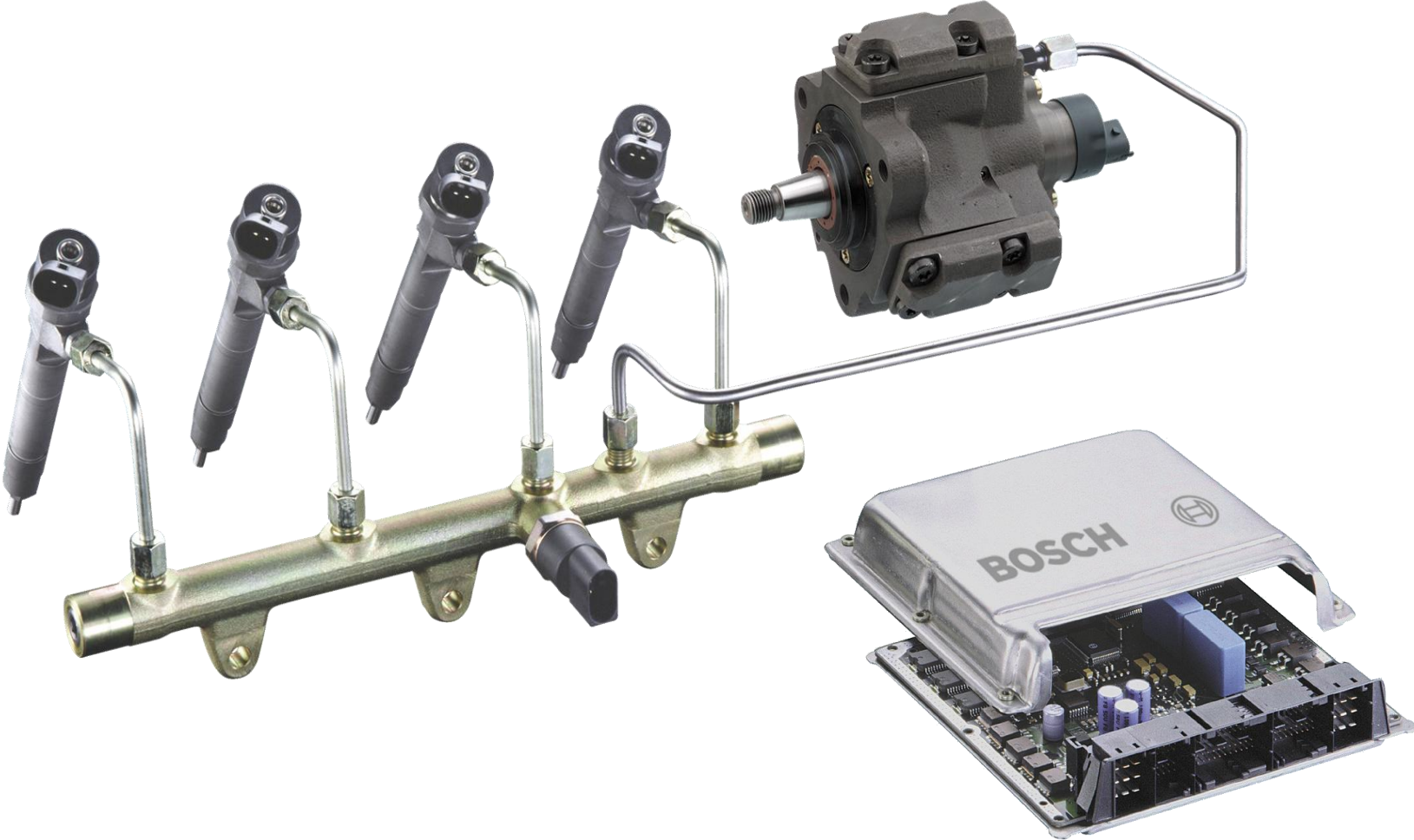
Pump Elements



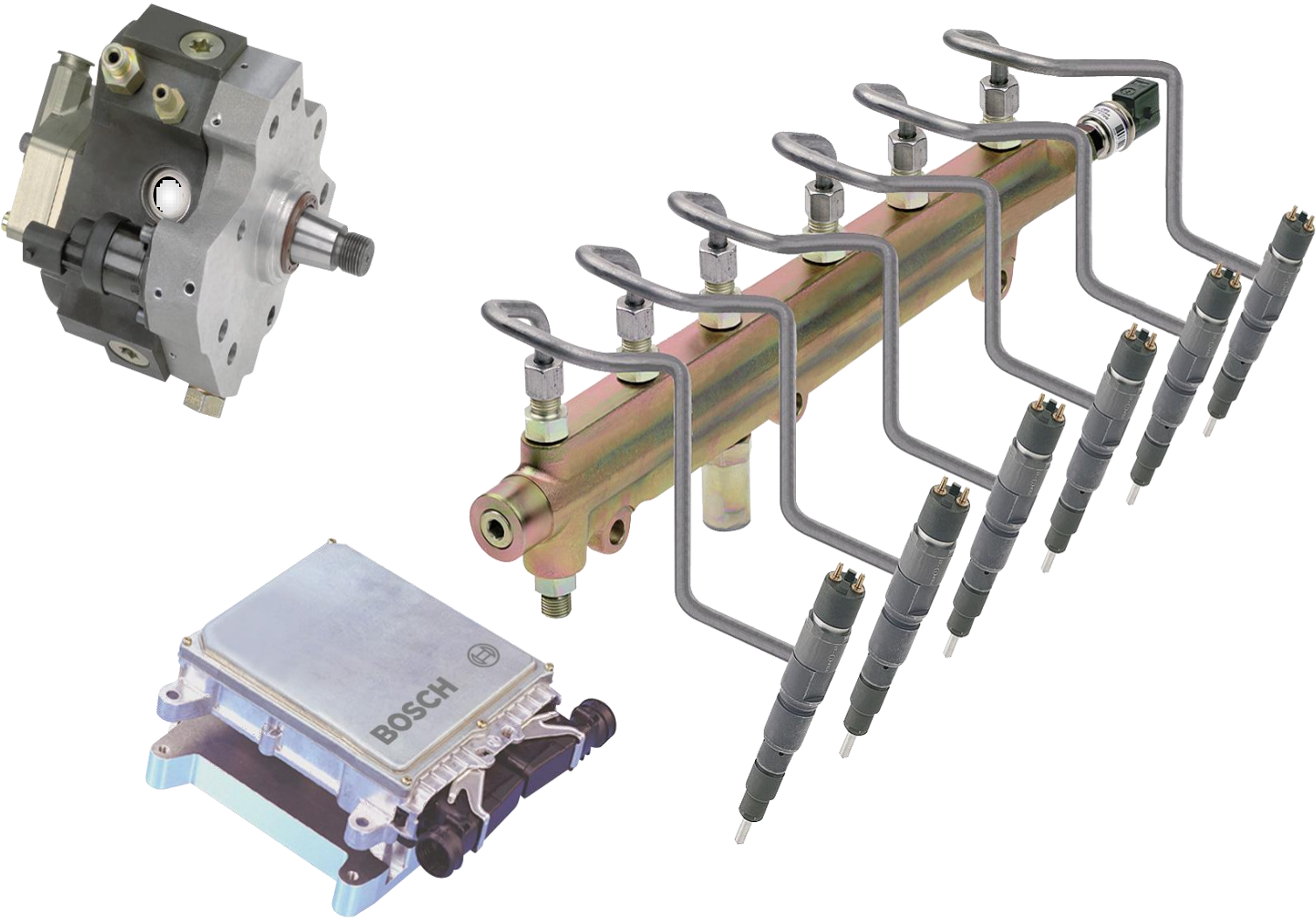
Common Rail Injector



# Common Rail System for Passenger Cars

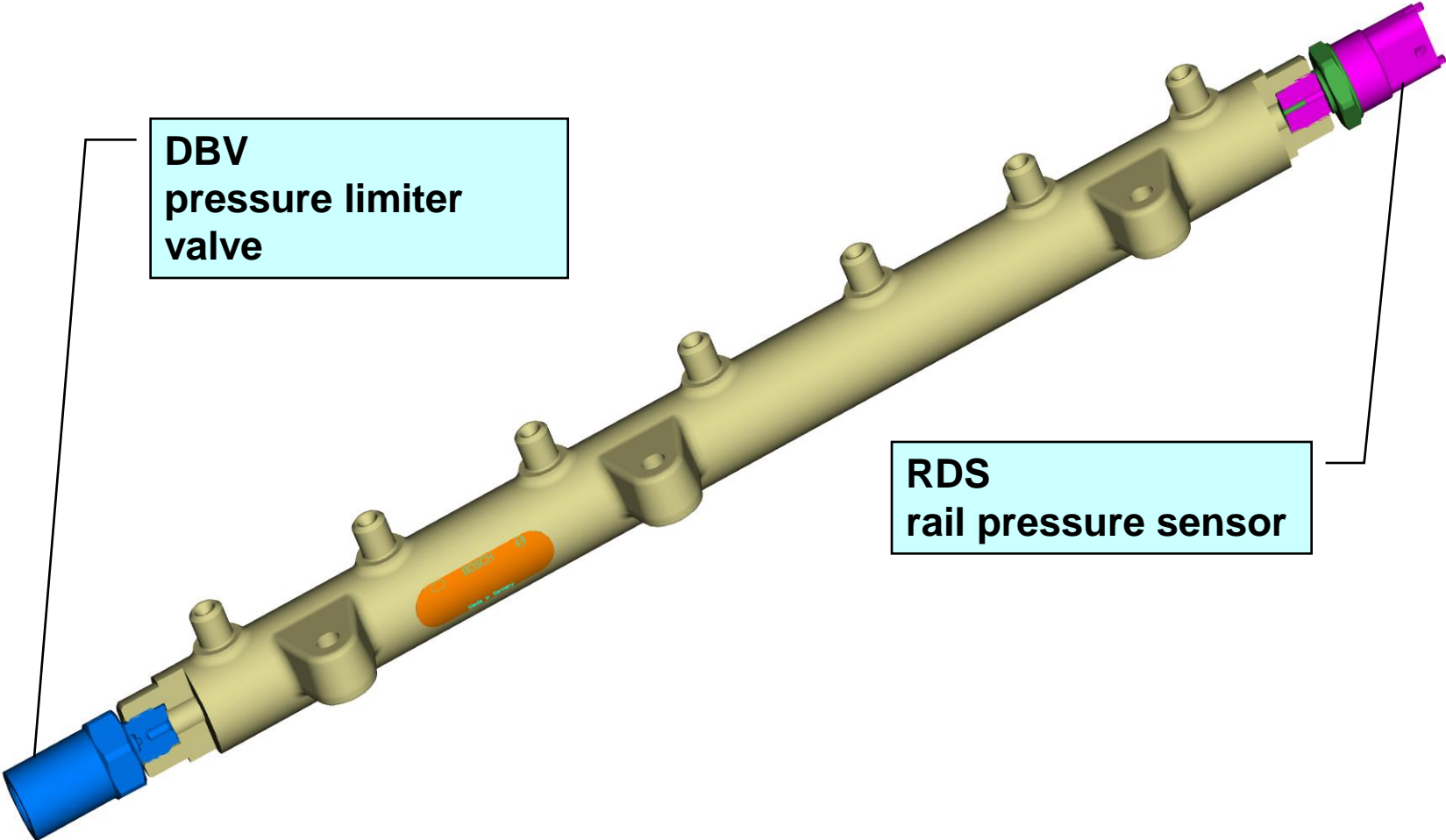


# Common-Rail System for Heavy Duty Engines





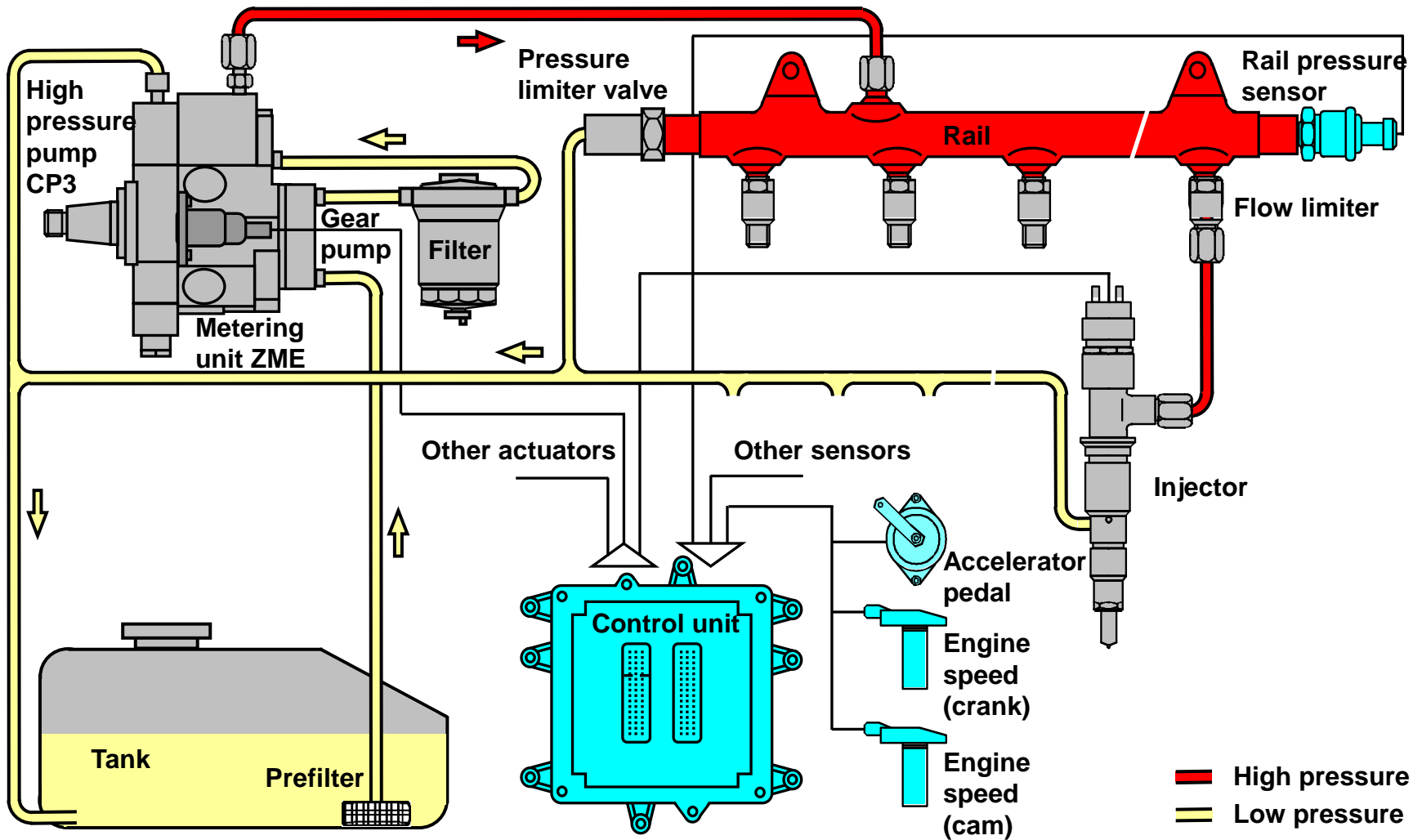
# Common-Rail (Truck Rail Basic Design)



**DBV  
pressure limiter  
valve**

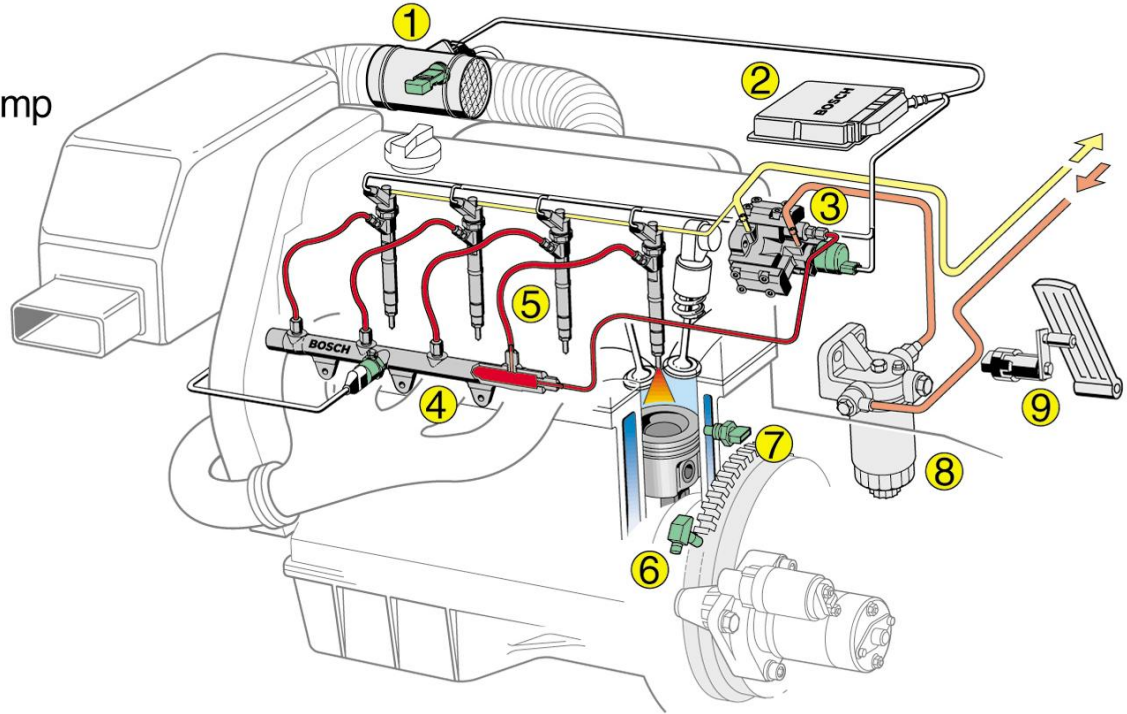
**RDS  
rail pressure sensor**

# Common-Rail System

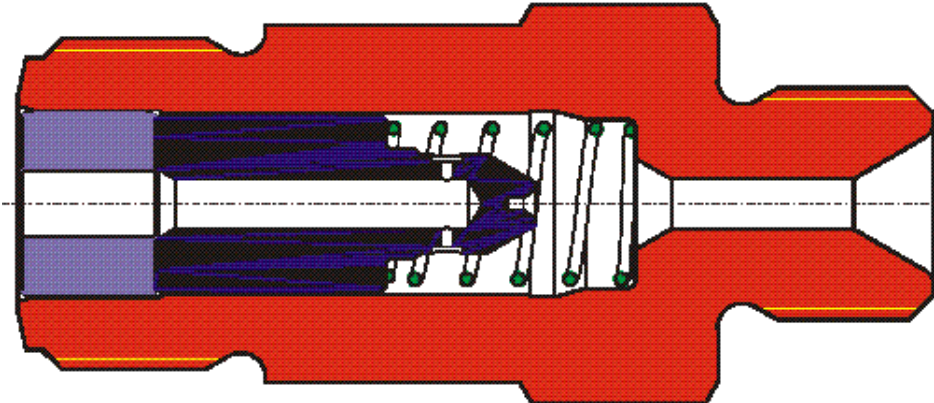


# Fuel System Control

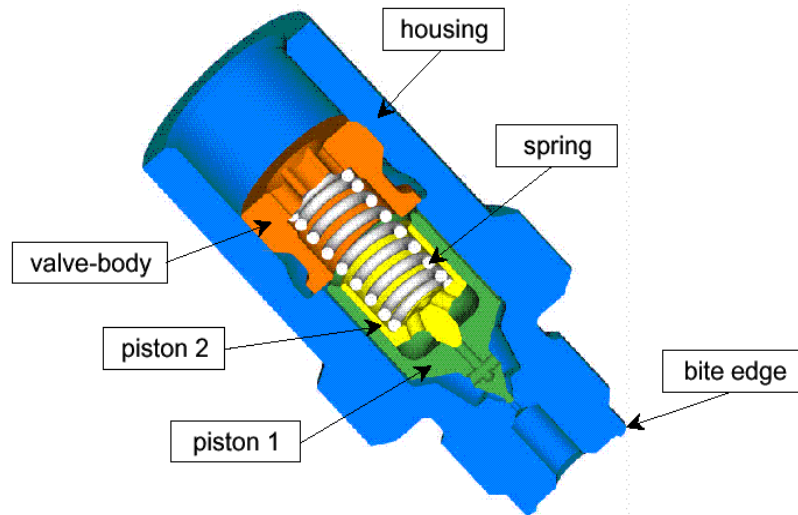
- ① Air mass meter
- ② Engine ECU
- ③ High pressure pump
- ④ Common rail
- ⑤ Injectors
- ⑥ Engine speed sensor
- ⑦ Coolant temp. sensor
- ⑧ Filter
- ⑨ Accelerator pedal sensor



# Flow Limiter



# Two-step Pressure Limiting Valve



# Rail Pressure Sensor Features

- Stainless-steel diaphragm with thin-film
- Complete separation of medium and electronics
- Compact, rugged, resistant against vibration
- Pressure ports:
  - M12 x 1,5 with sealing ring
  - M18 x 1,5 with self sealing edge
- Operating range: 0 - 150/180 MPa
- High accuracy:  $\pm 1,2\%$  FS (over life time/main op. range)



# Common Rail Fuel Injector (Heavy Duty)

## Valve Group:

- Reduced control volume
- Improved piston geometry

## Nozzle:

- Miniature sac hole or sac hole respectively
- Nozzle guided pushrod
- Single needle guidance
- Flat shoulder

## Magnet Group:

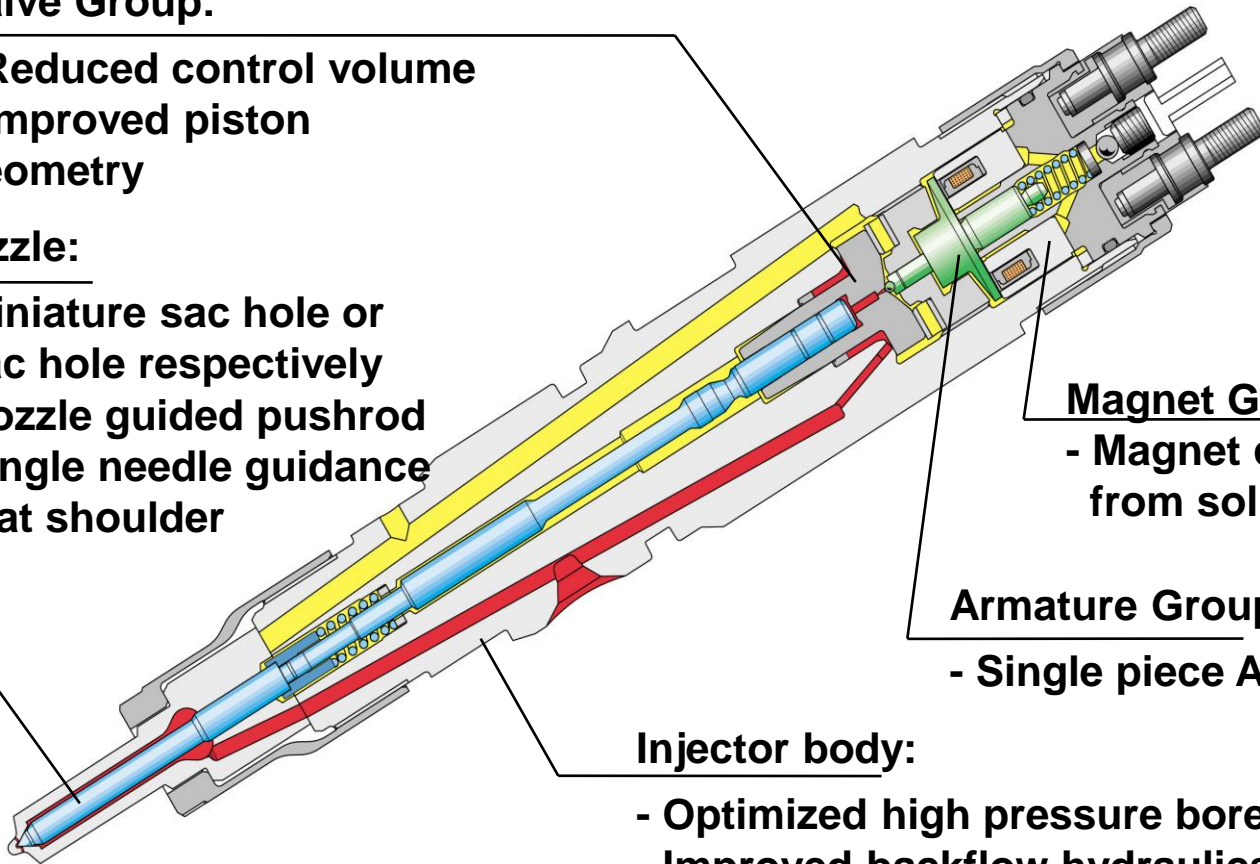
- Magnet core from solid

## Armature Group:

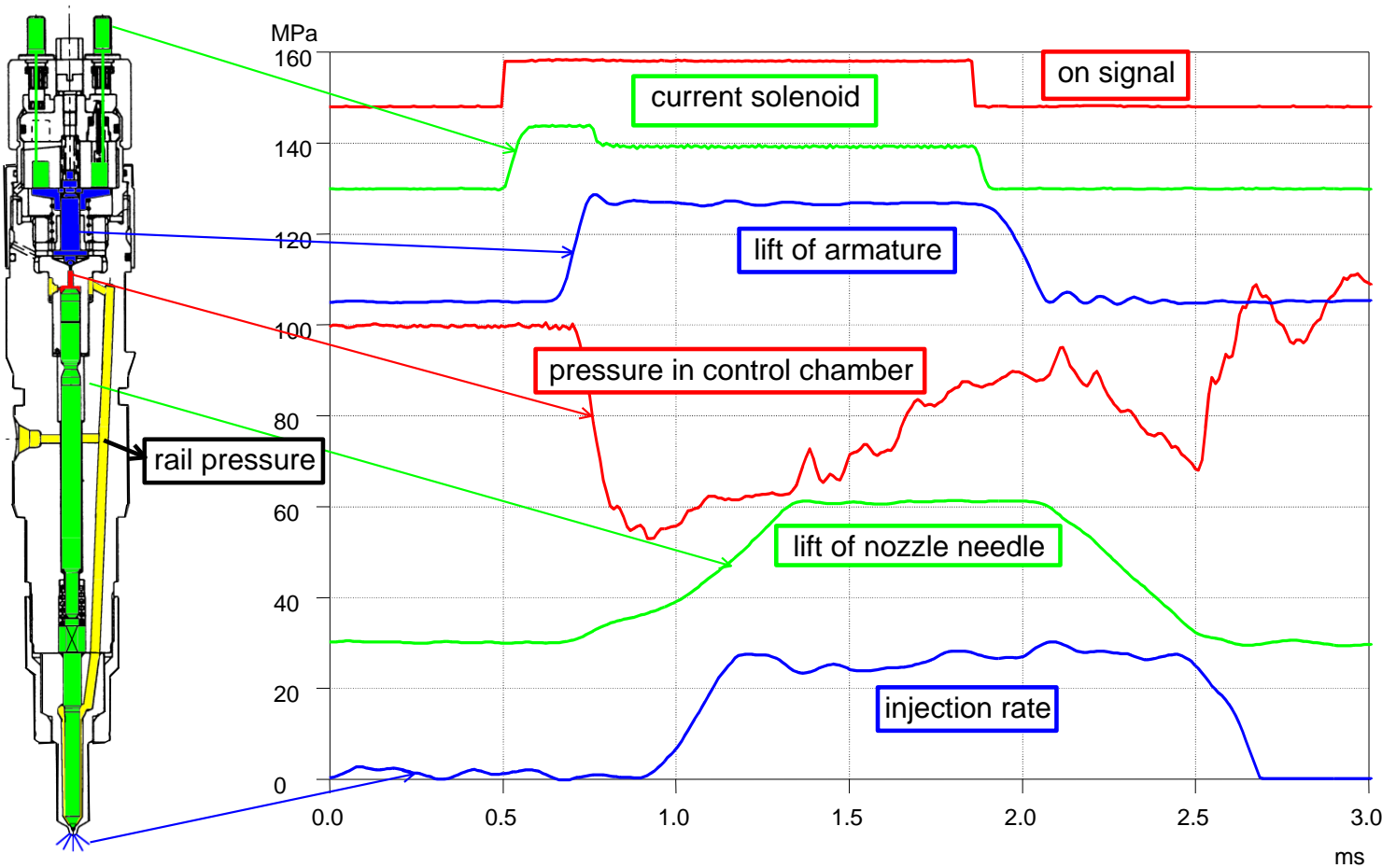
- Single piece Armature

## Injector body:

- Optimized high pressure bore geometry
- Improved backflow hydraulics

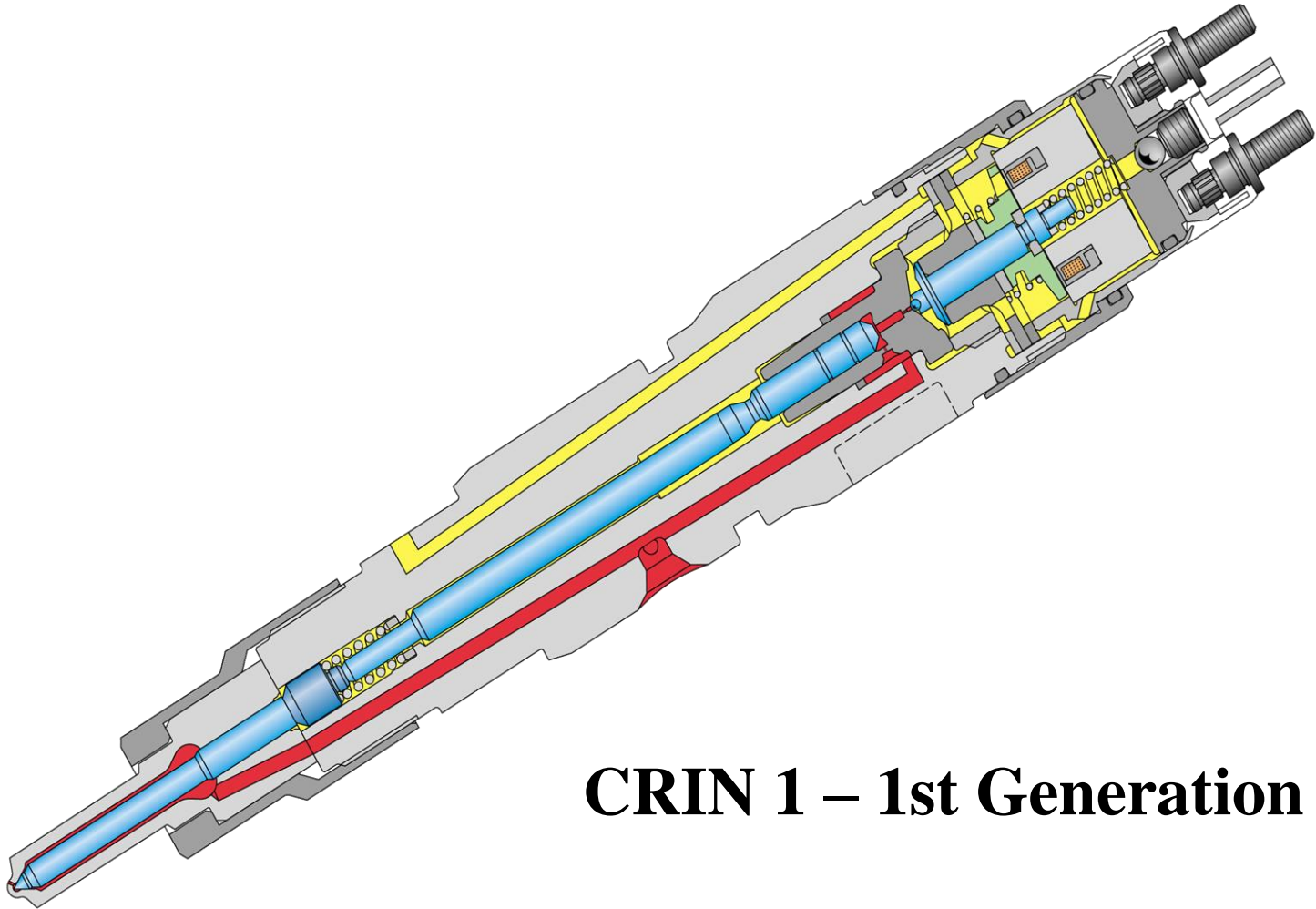


# Common Rail Fuel Injector



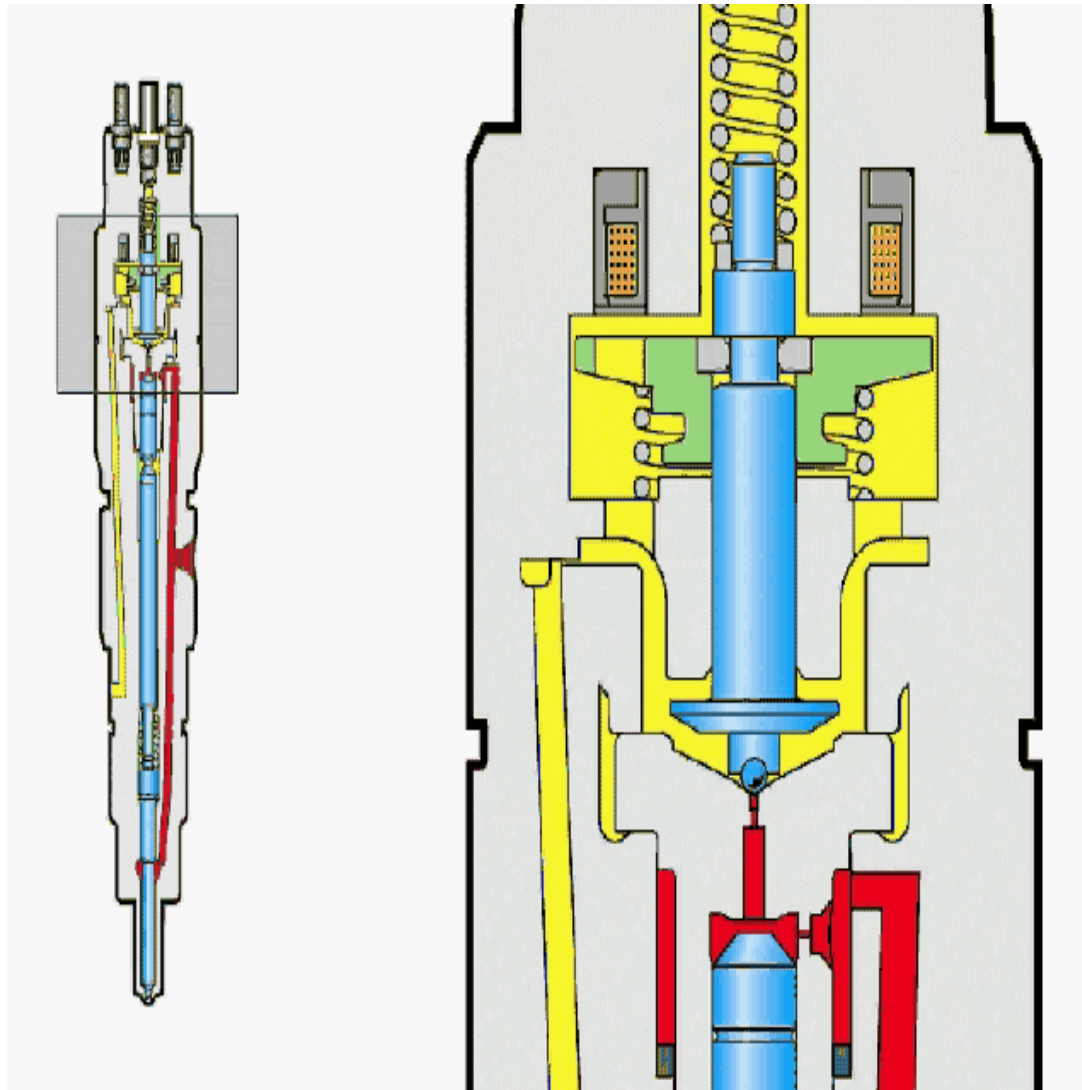


# Common Rail Fuel Injector – First Generation



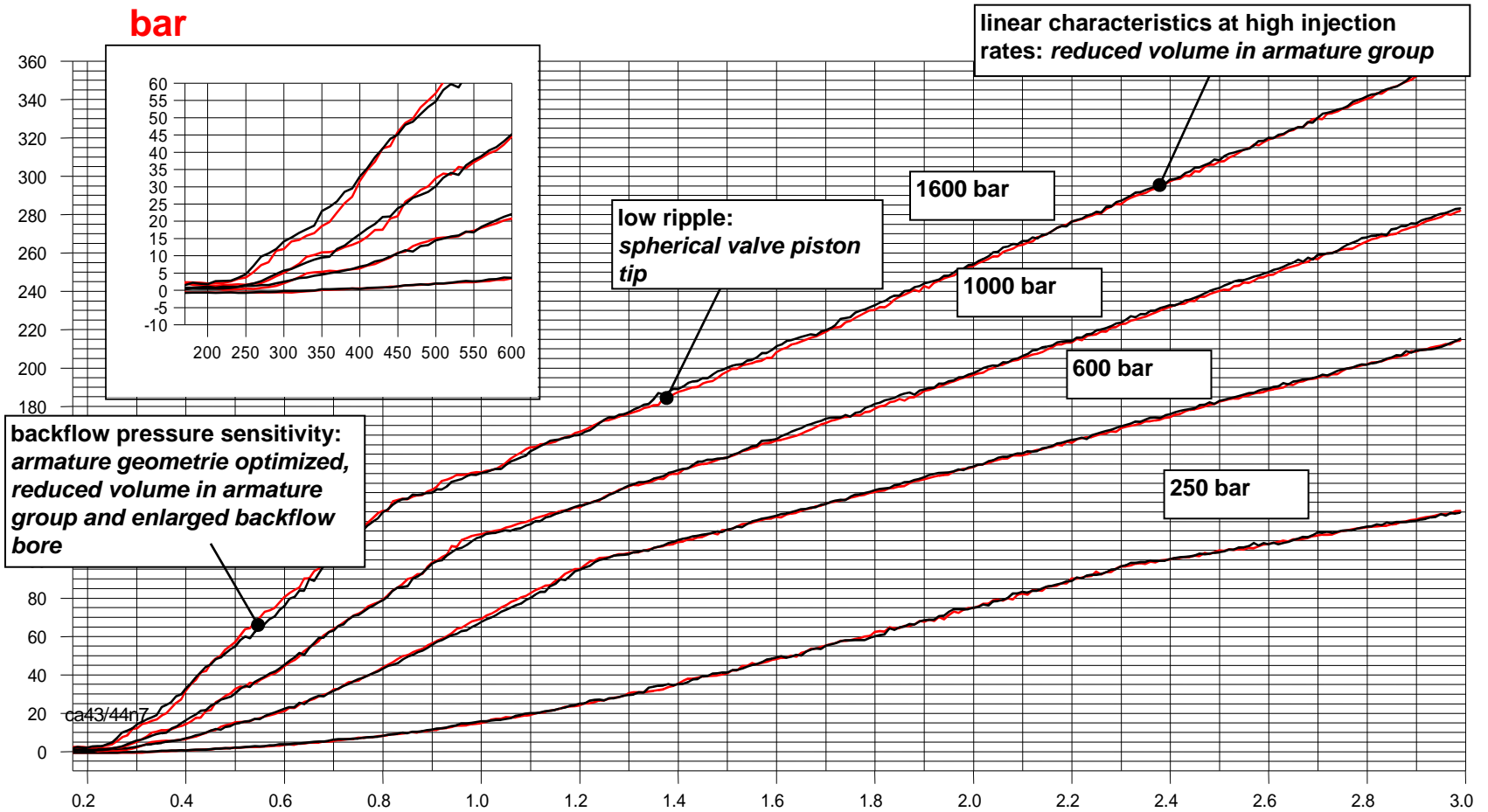
**CRIN 1 – 1st Generation**

# CR Injector – Principle of Function

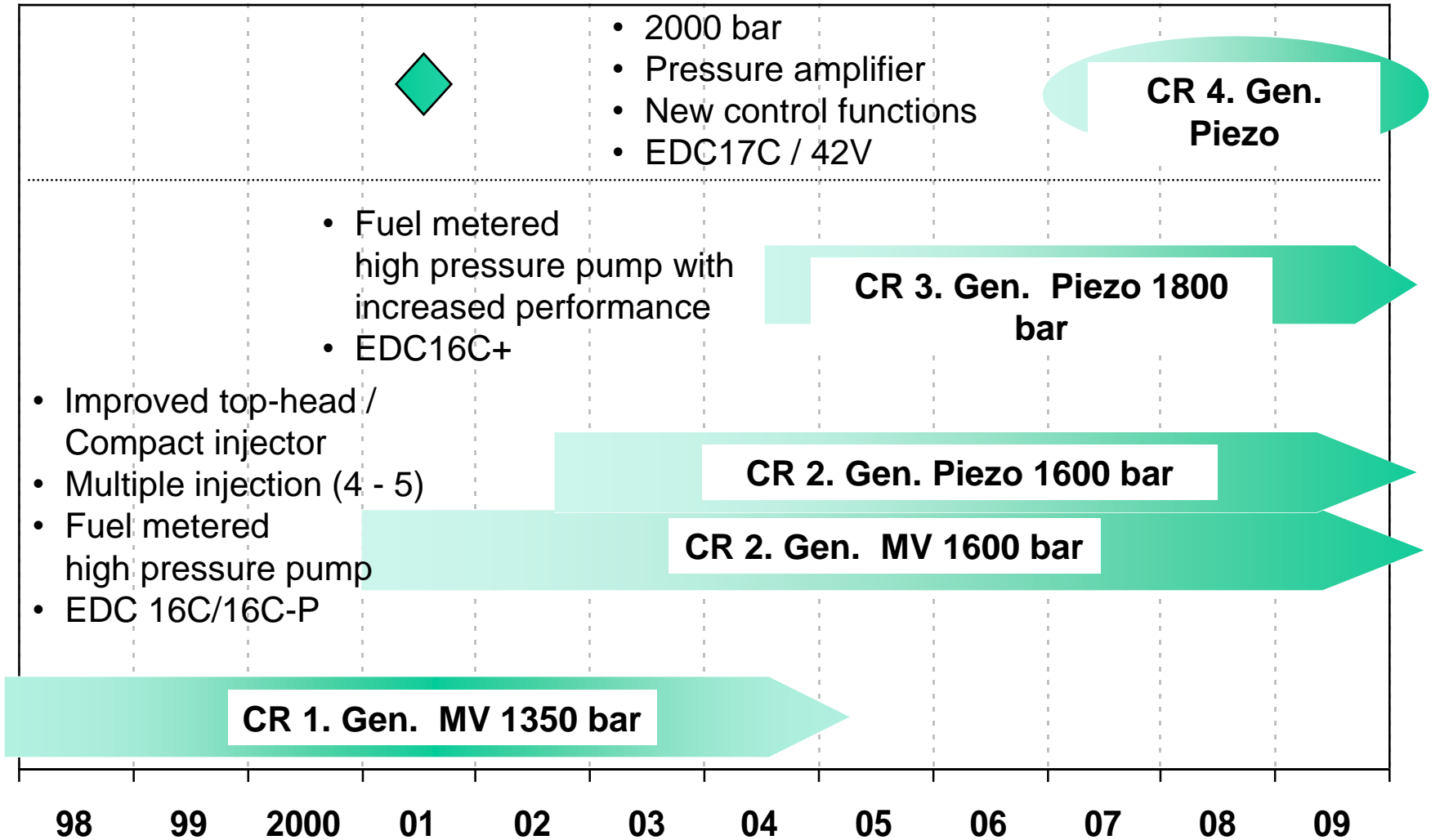


# CR Injector Map

backflow counterpressure: 0 bar and 0.5 bar

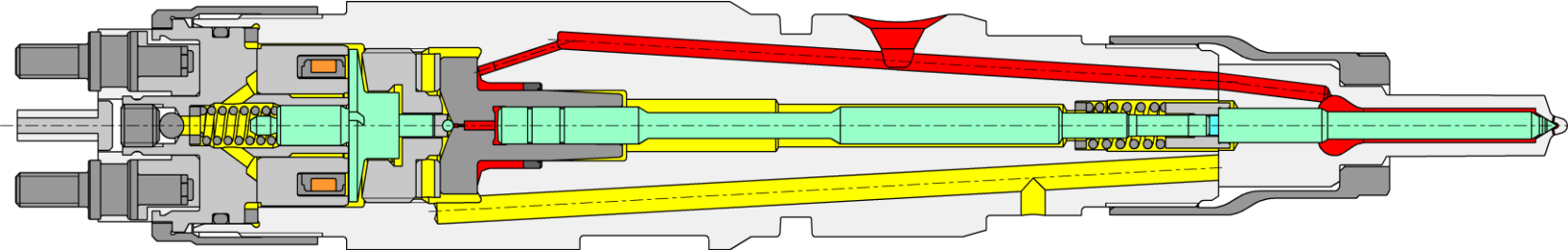


# CR System Roadmap (P Cars and Light Duty Trucks)



# CR Fuel Injector – Comparison

## 2. Generation

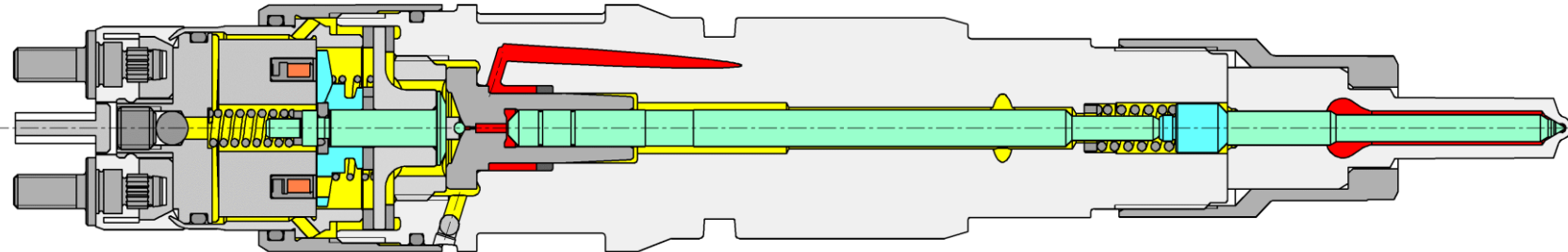


magnet core armature

control chamber geometry

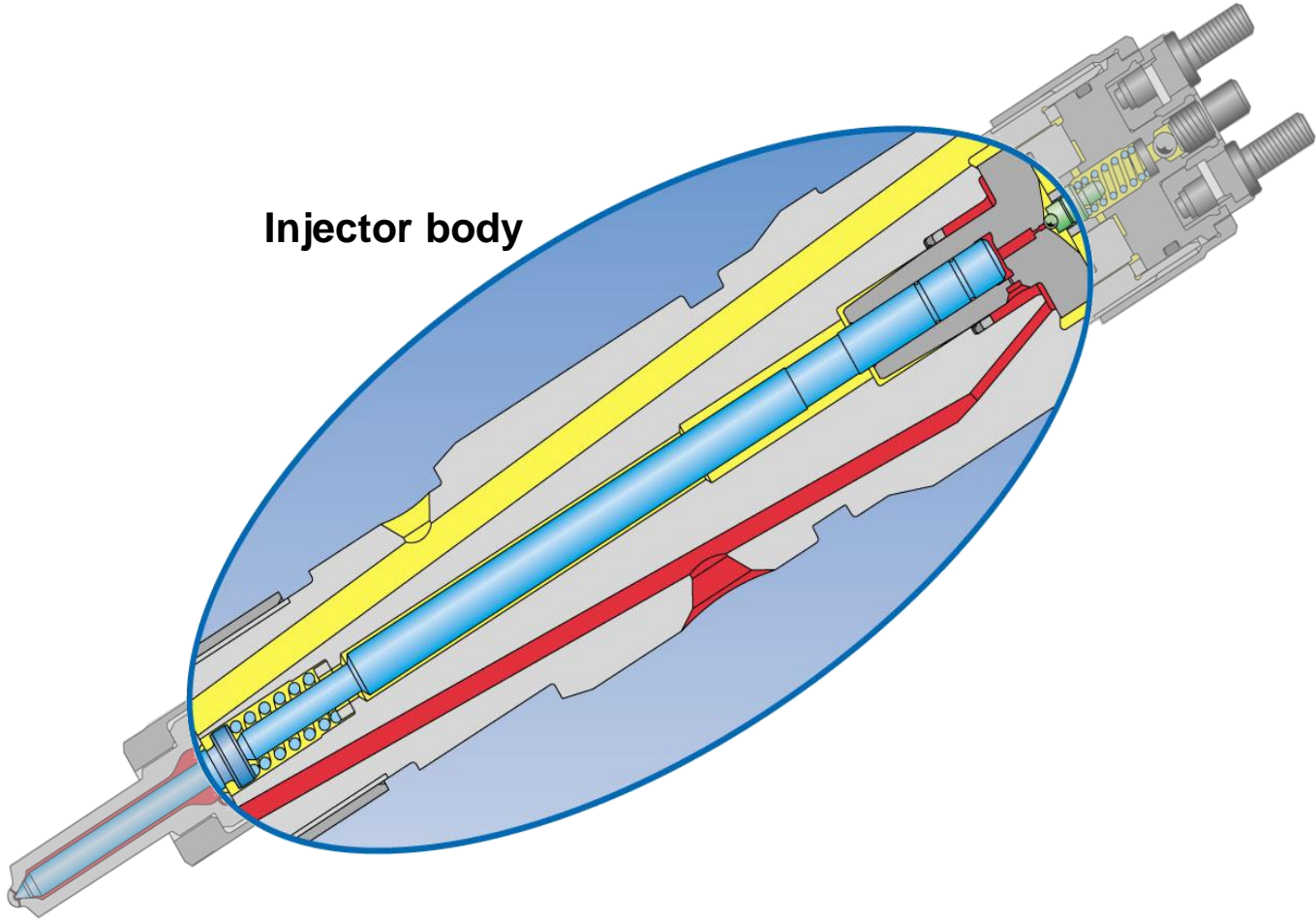
high pressure bore design

DGV intermediate pin

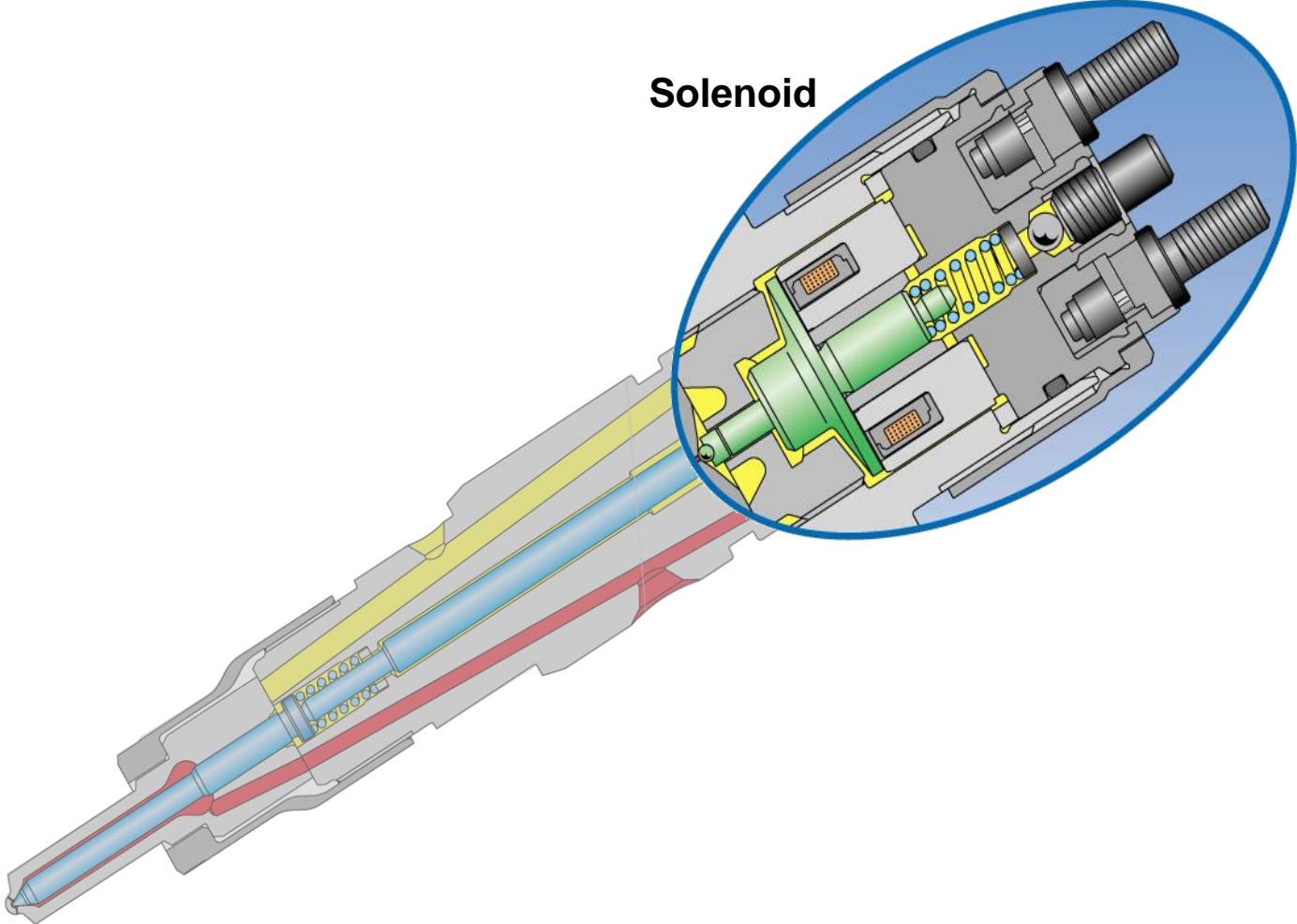


## 1. Generation

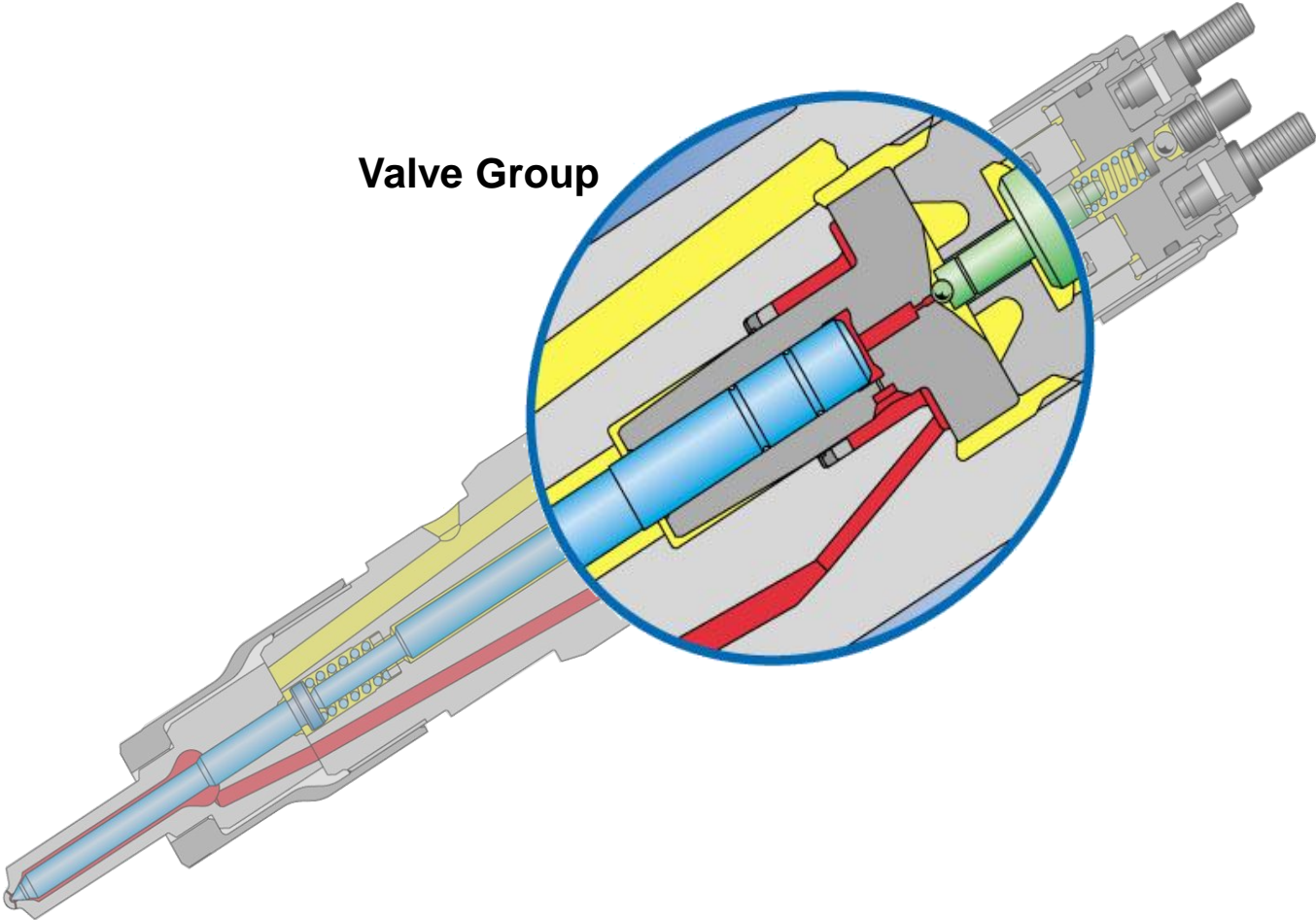
# CR Fuel Injector – 2. Generation (for Trucks)



# CR Fuel Injector – 2. Generation (for Trucks)

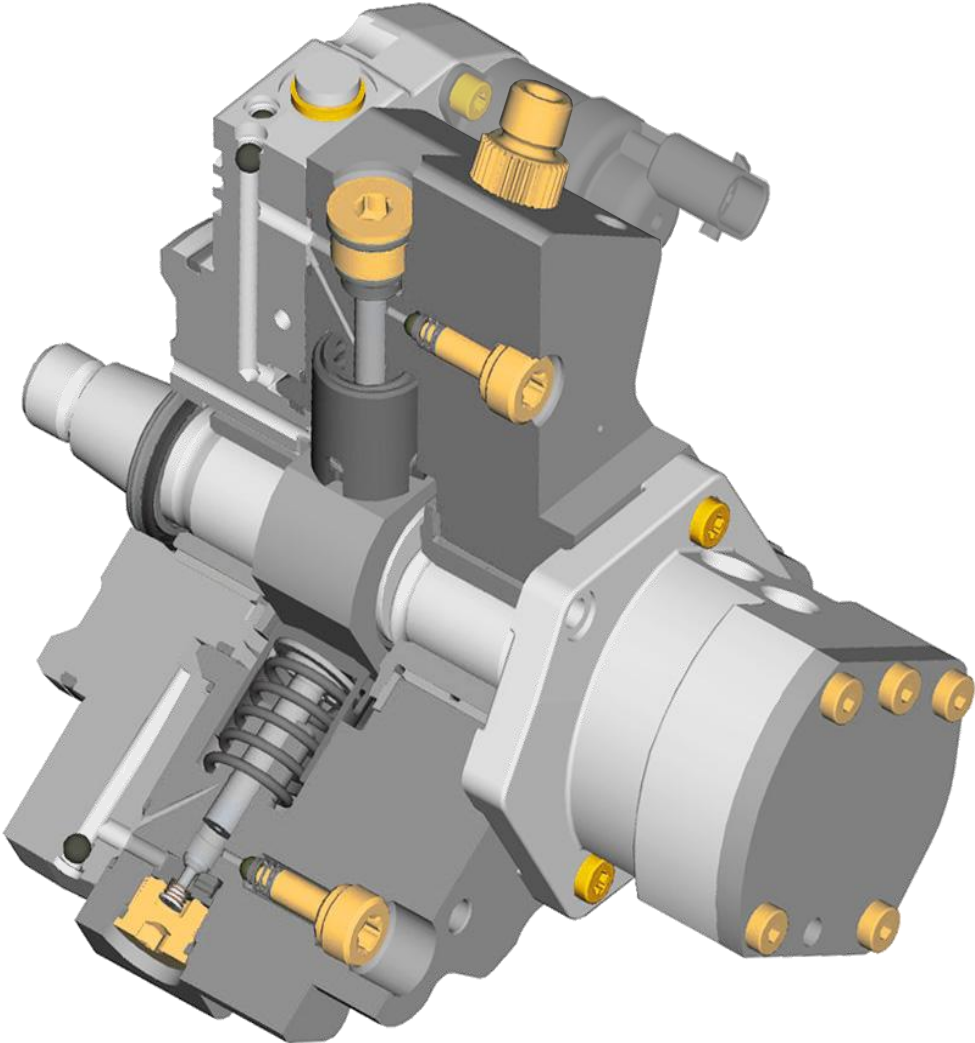


# CR Fuel Injector – 2. Generation (for Trucks)

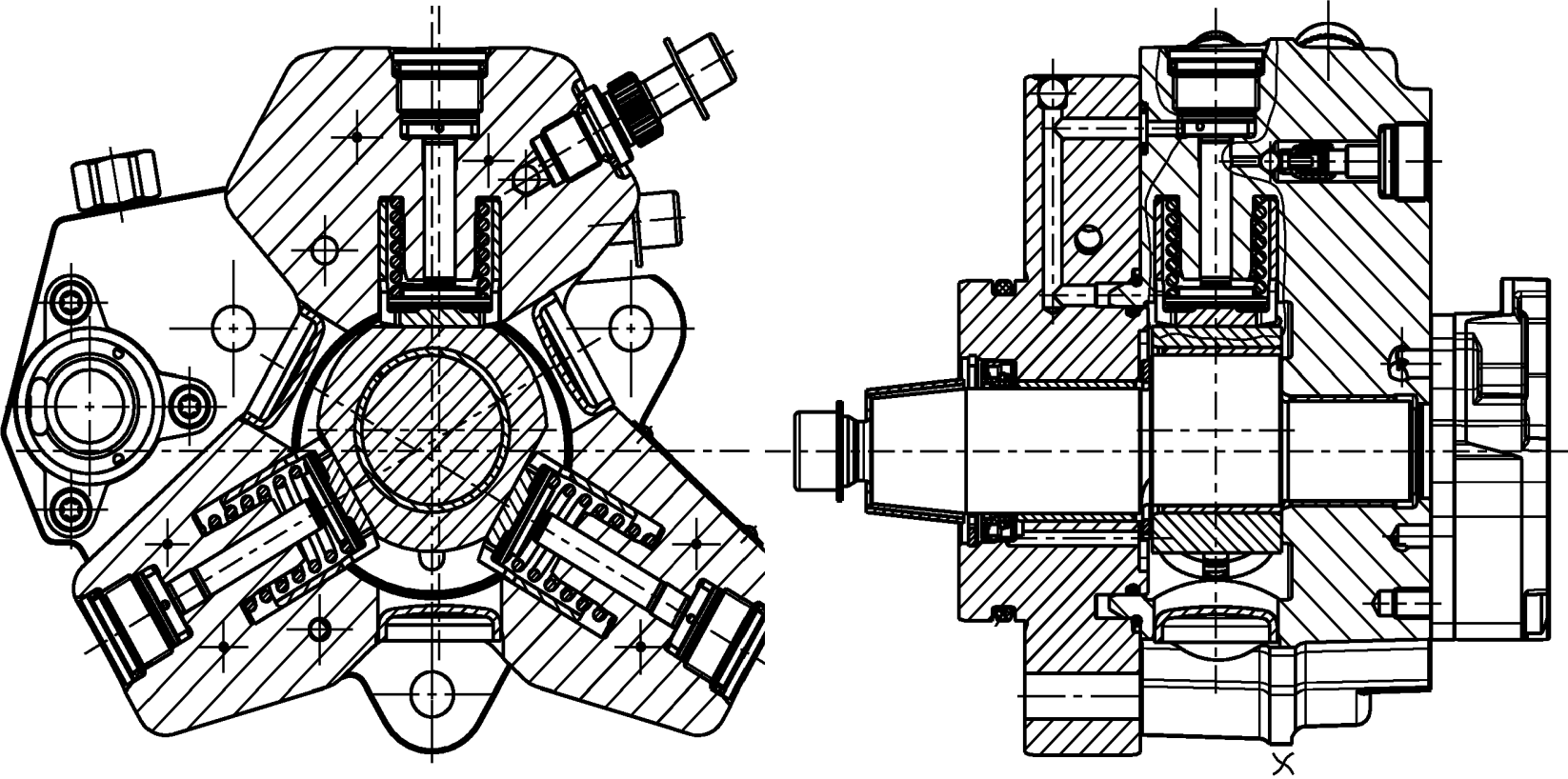




# High Pressure Pump



# High Pressure Pump



# CR Systems Status

## 1. Generation, 1350 bar systems

Euro III achieved with different applications

Low combustion noise with pilot-injection

Production of different applications (3 to 8 cylinders)

## 2. Generation, 1600 bar

Euro IV achieved with different applications

Combustion noise further reduced with 2 pilot-injections

# HD Diesel Engine – EURO IV

Advanced combustion characteristics

combustion chamber, swirl, number of nozzle holes

Advanced air management system

high excess-air ratio,  $\lambda \geq 1.8$  (steady state and transient)

Advanced supercharging system

“super” VGT or electronic powered charger

Close loop EGR system with VGT and linear EGR valve

efficient charge air cooling and EGR cooling

Application with retarded SOI for low EGR Rate

High injection pressure  $\geq 1600$  bar

Oxidation Catalyst for soluble fraction

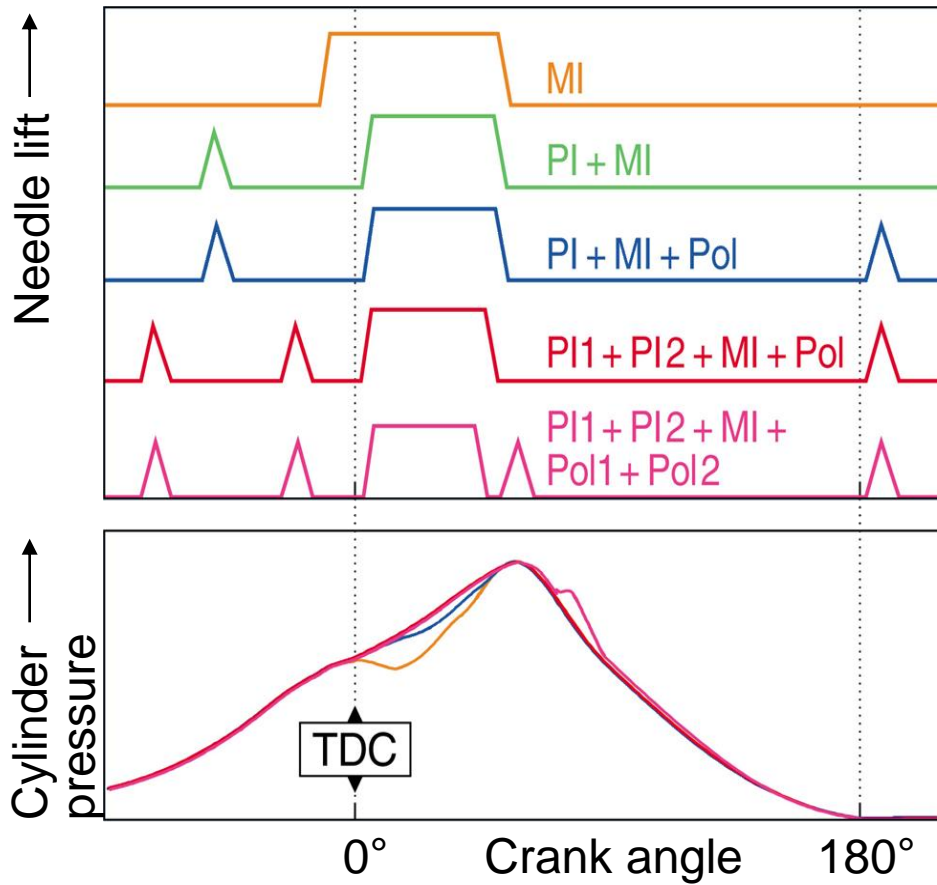
Sulphur free fuel

## **Conclusion:**

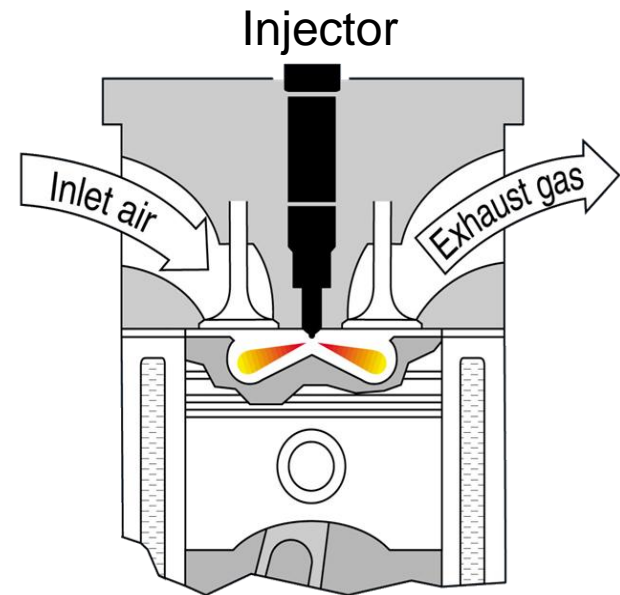
steady state test ESC possible

transient test ETC only possible with all mentioned measures

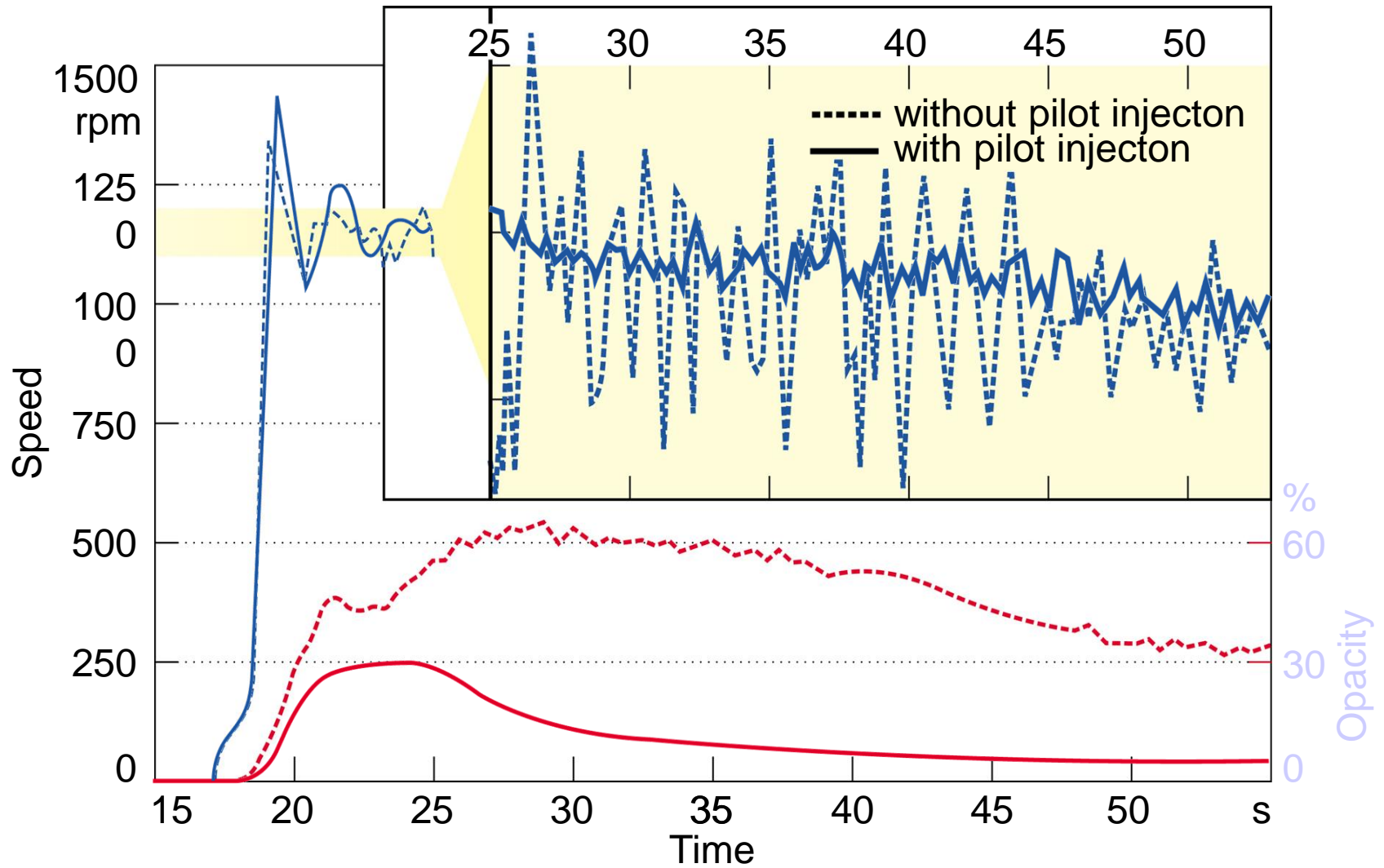
# CR System – Comparison of Injection Strategies



PI Pilot injection  
MI Main injection  
Pol Post Injection

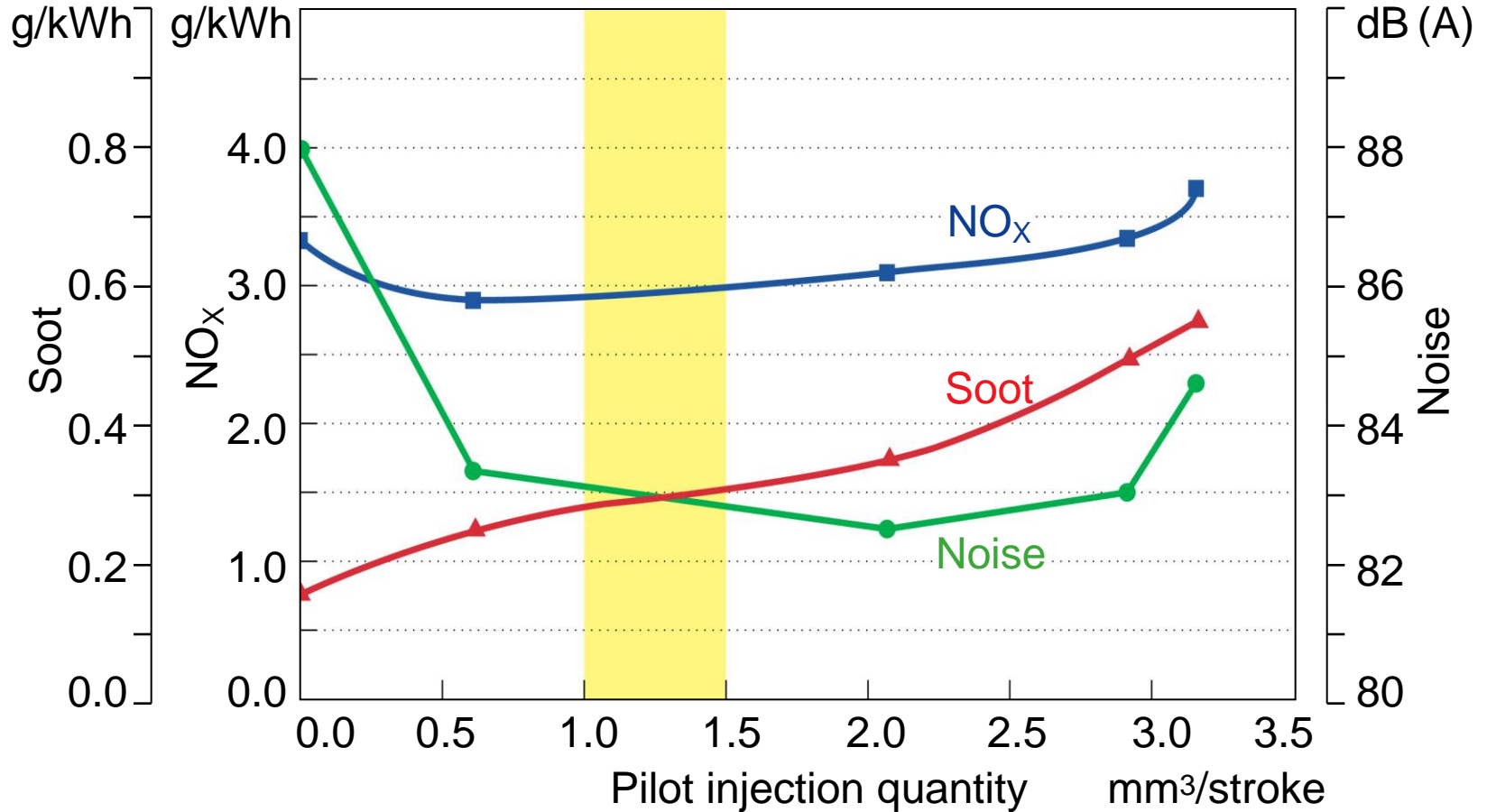


# DI Engine – warm up after cold start at -20° C



# Influence of Pilot Injection on Noise and Emissions

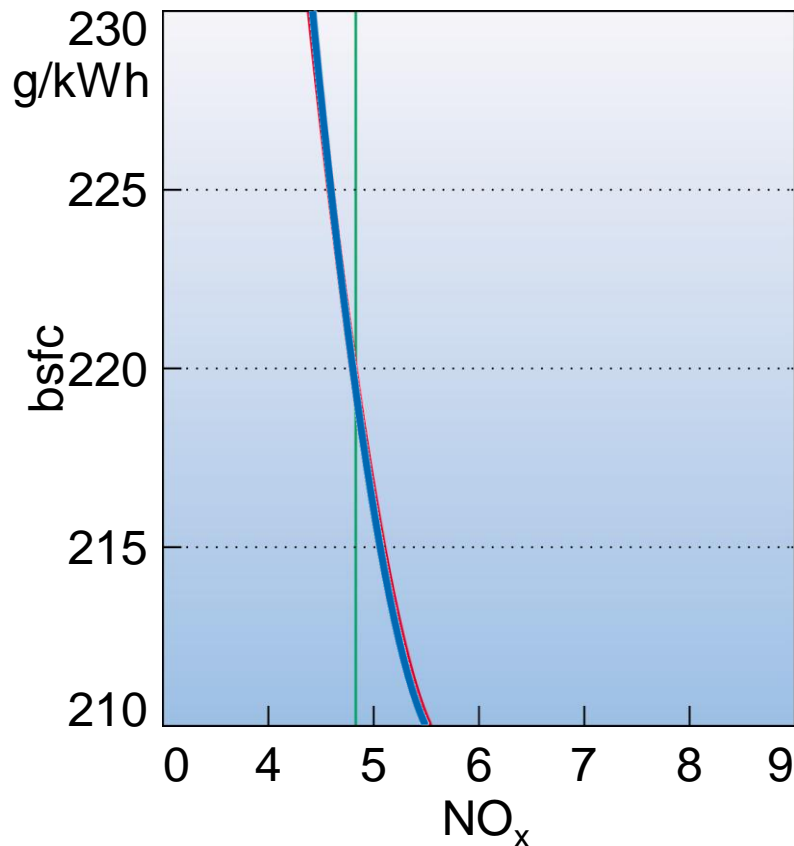
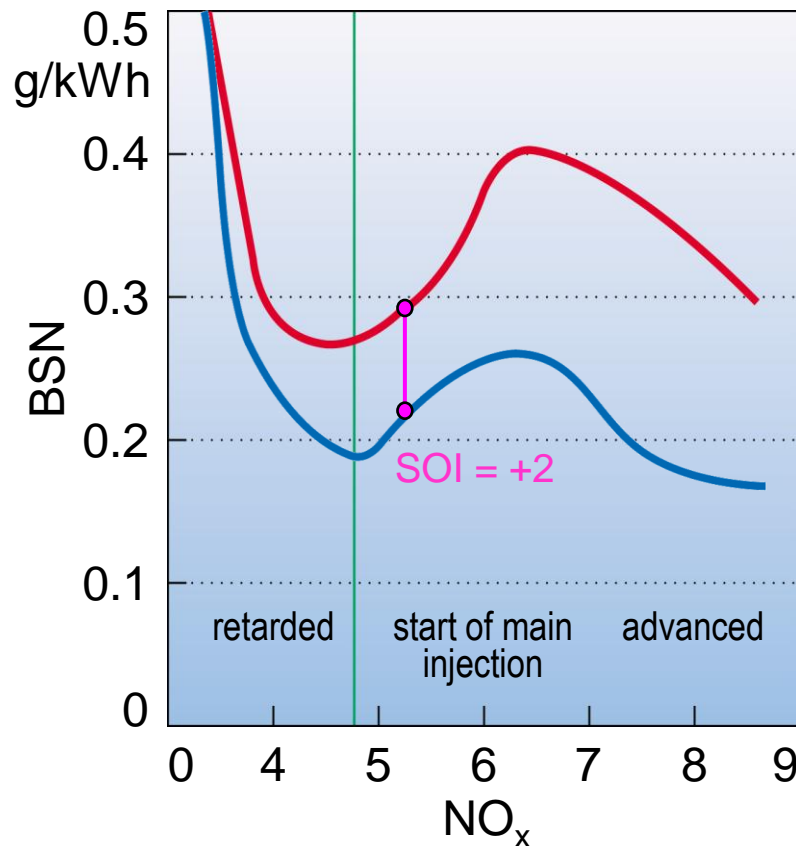
1.9 TDI with CR, speed = 1500 rpm,  $p_{me} = 5$  bar, rail pressure = 800 bar



# HD Diesel Engine – Post Injection

Variation of start of injection  
with appended post injection (rail pressure = 900 bar)  
Fuel quantity of post injection approx. 12 mg/str

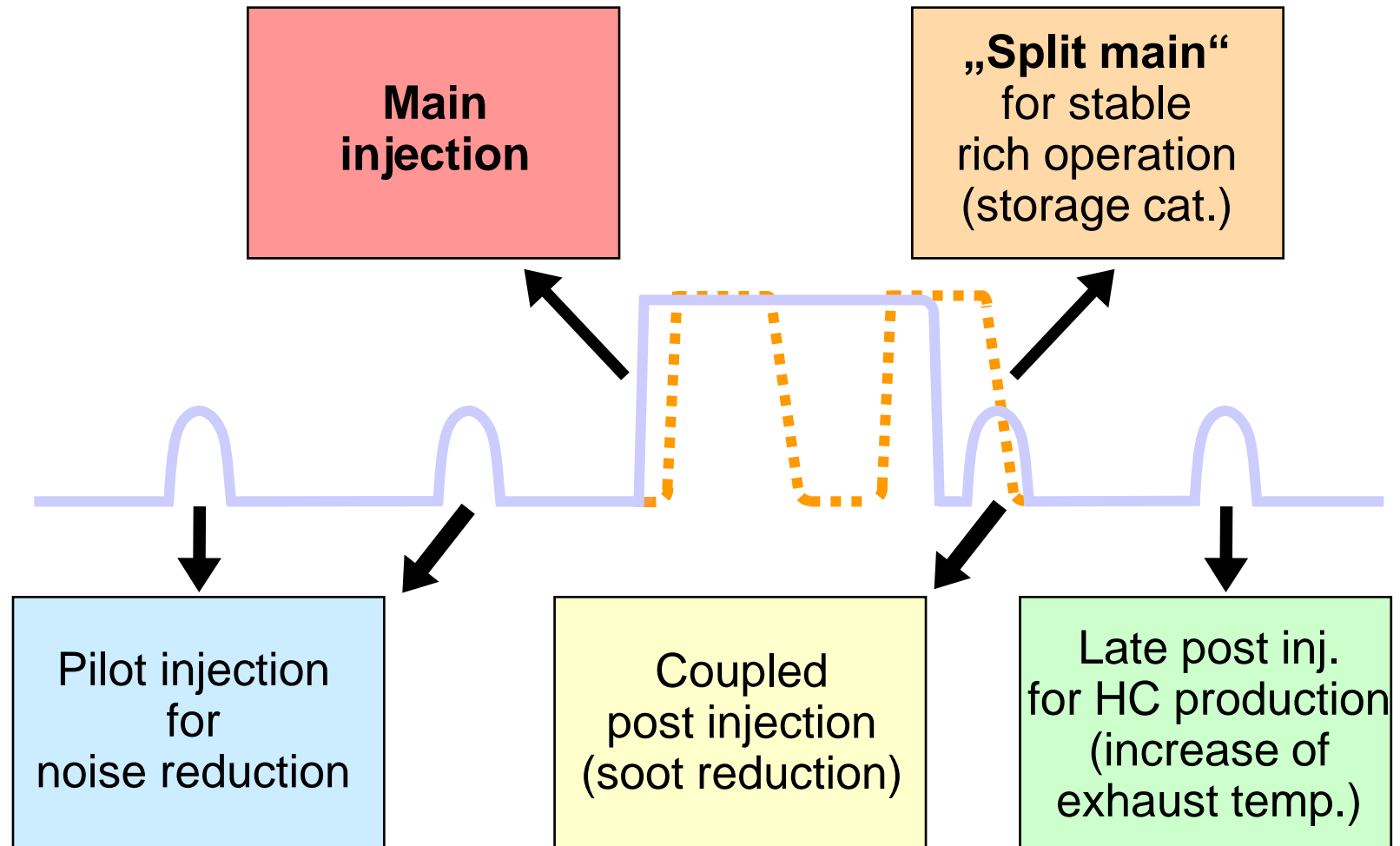
— with post injection  
— without post injection



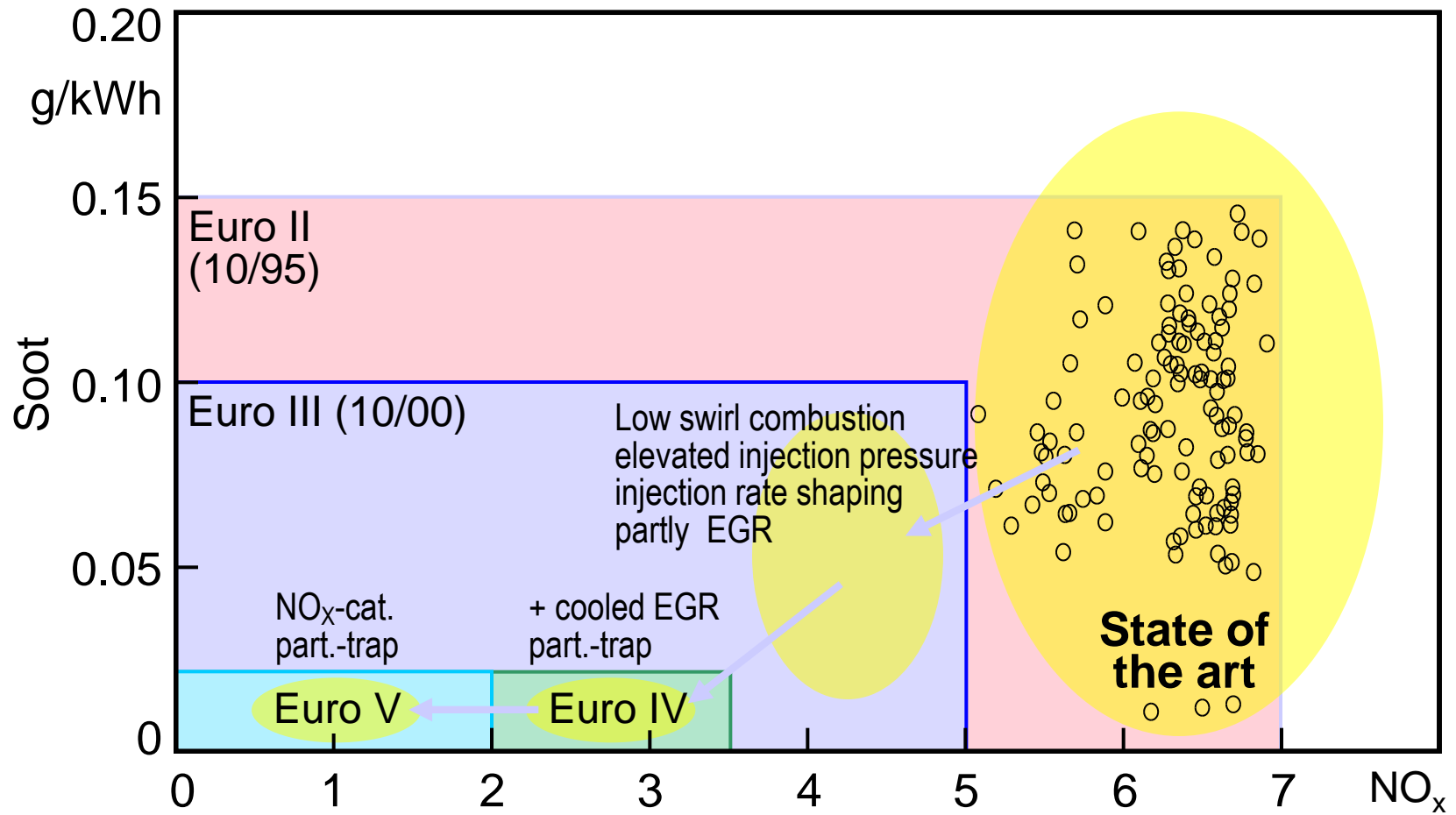
1400 rpm, 75% Load



# Common Rail System – Passenger Car



# Type Approval Data of HD Engines ( > 85 kW )



# Exhaust Gas Aftertreatment for P Cars

## **Assessment of aftertreatment for Diesel-PC, EURO IV and up**

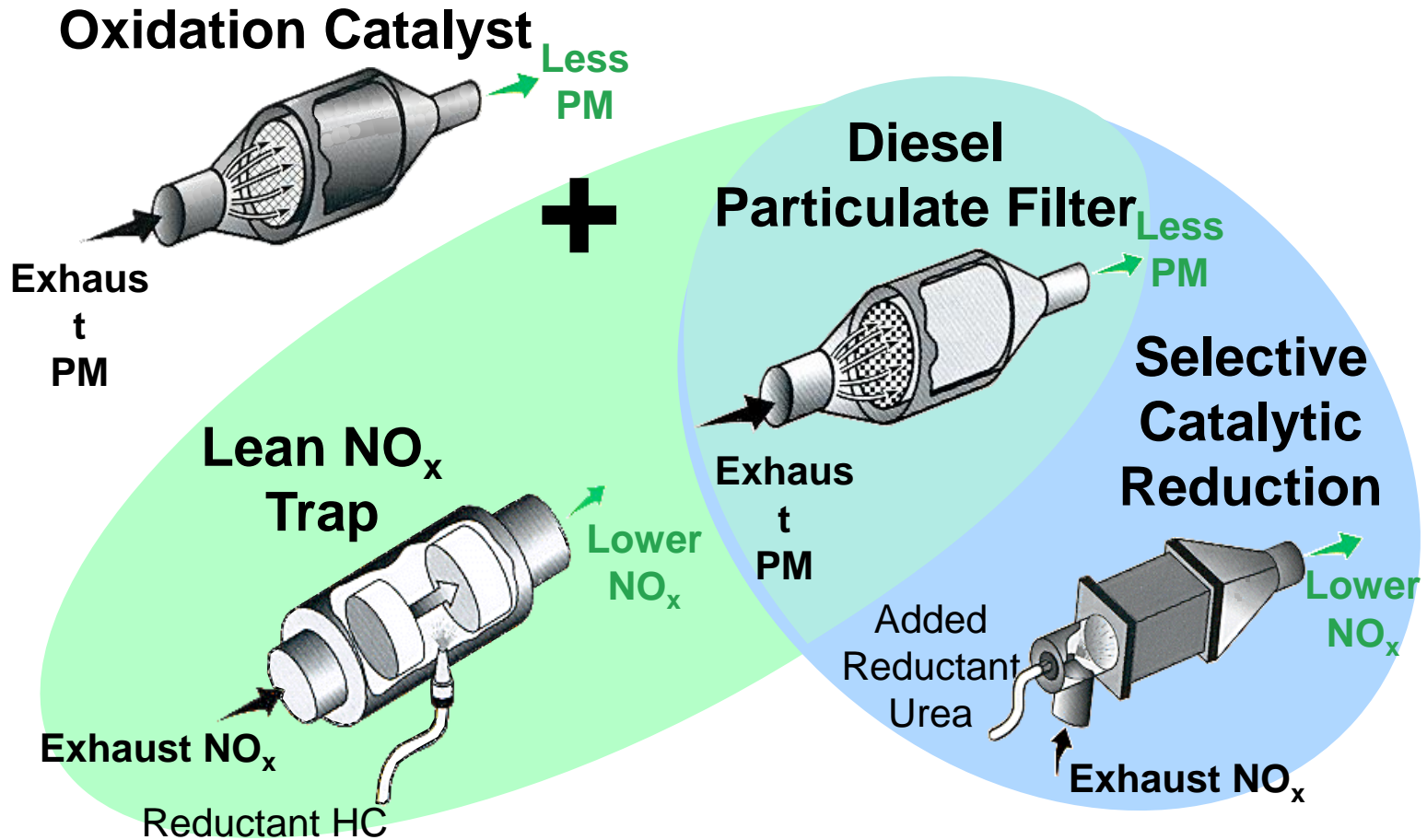
### **Priority I (PM Reduction):**

Particulate trap technology is politically necessary and technically suggestive; short-term market release

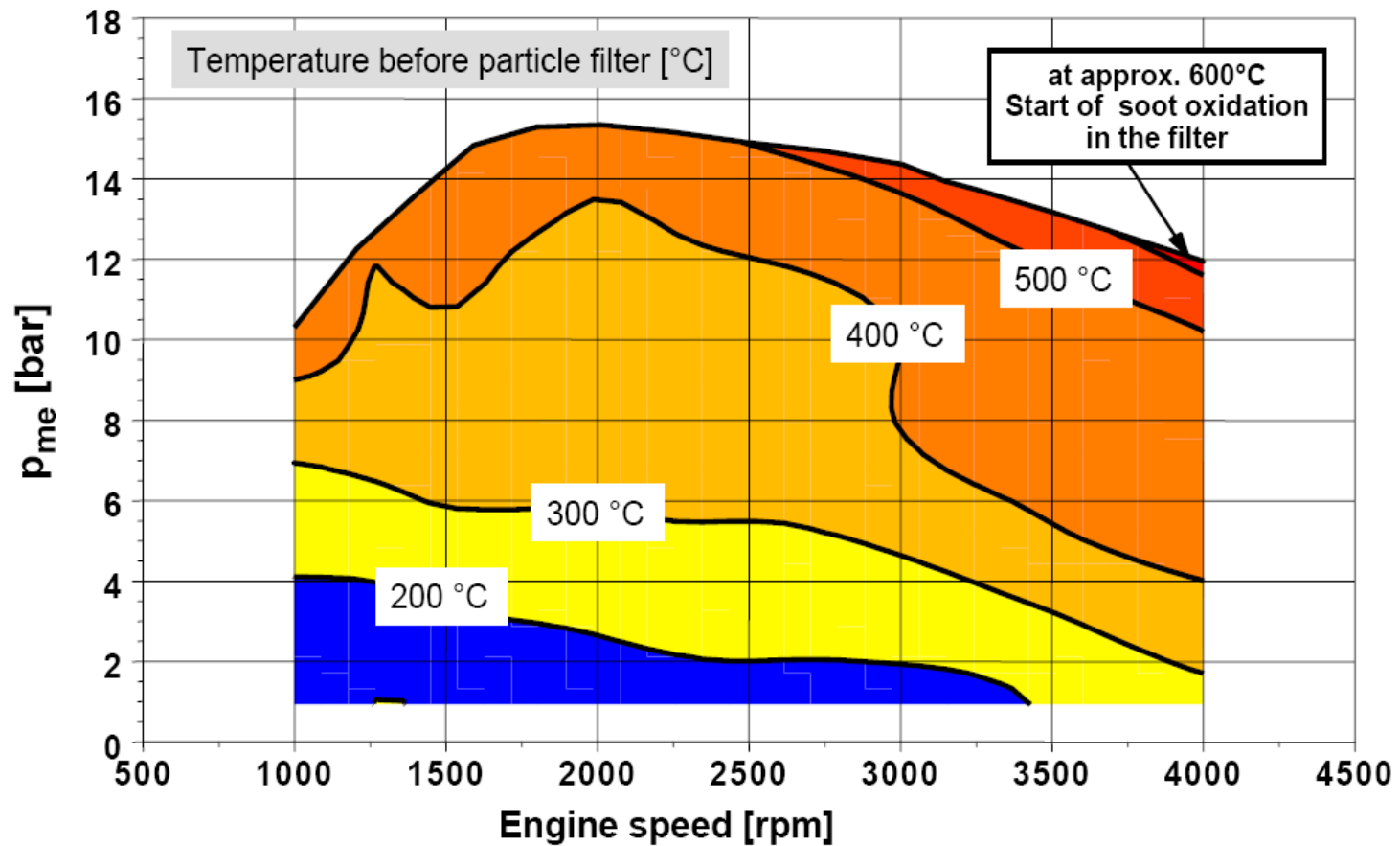
### **Priority II (NOx Reduction):**

With market release of low-sulphur fuel NOx trap has high potential; further development necessary; medium-term market release  
NOx reduction with urea/water remains an individual solution

# Diesel Emission Reduction Technologies – P Cars

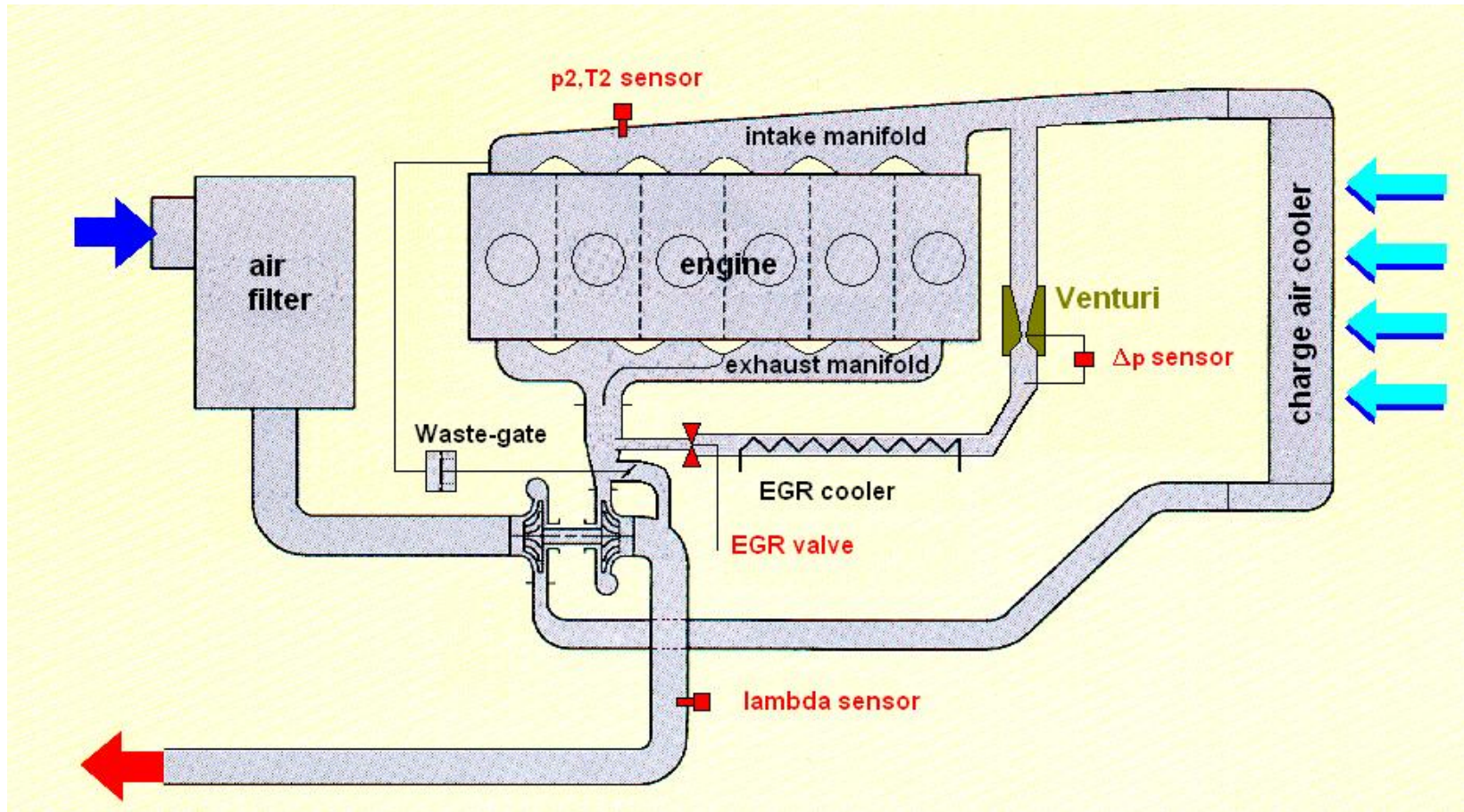


# Influence of Pilot Injection on Noise and Emissions

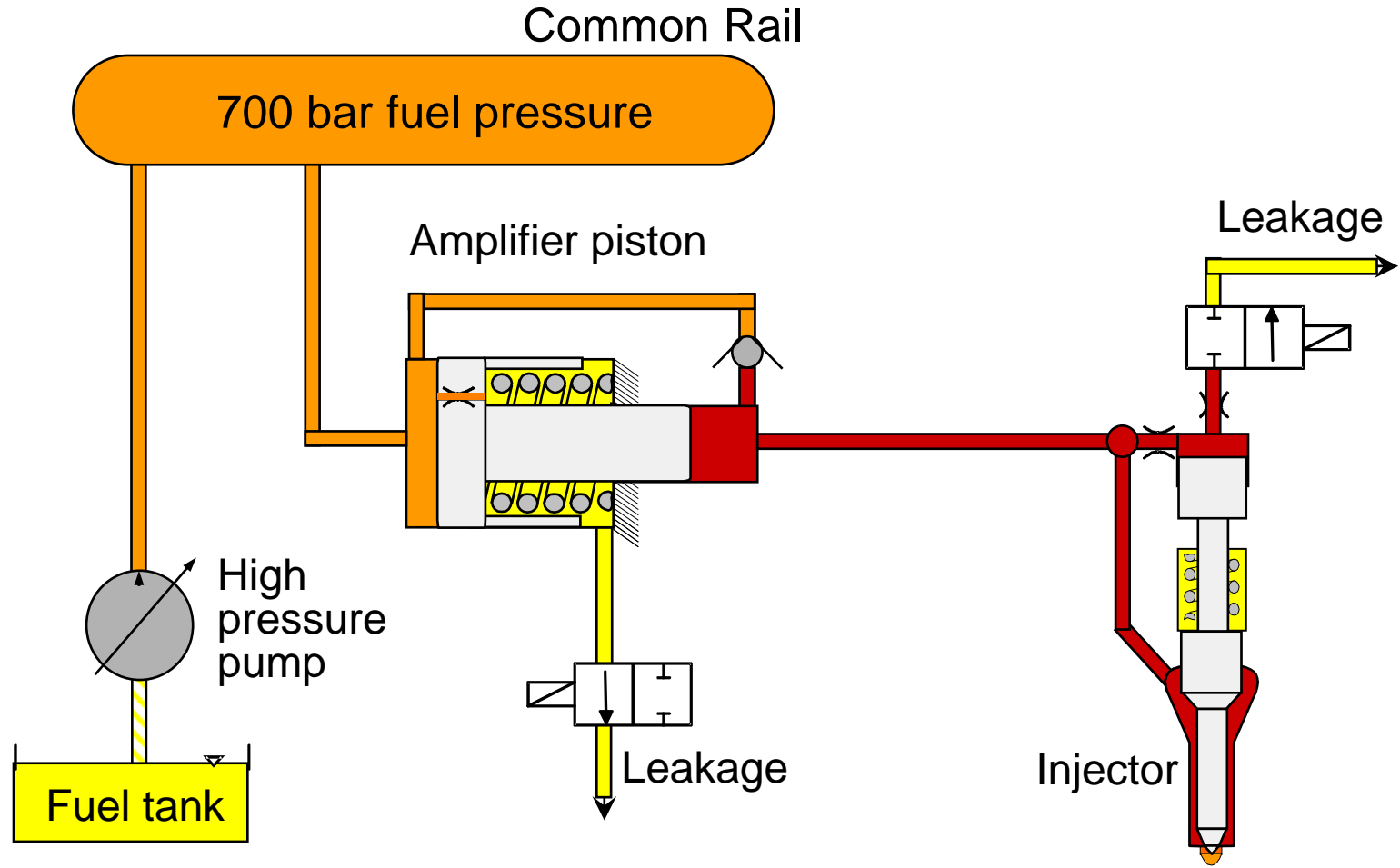


Regeneration of the particle filter is the challenge!

# Diesel Emission Reduction Technologies - EGR



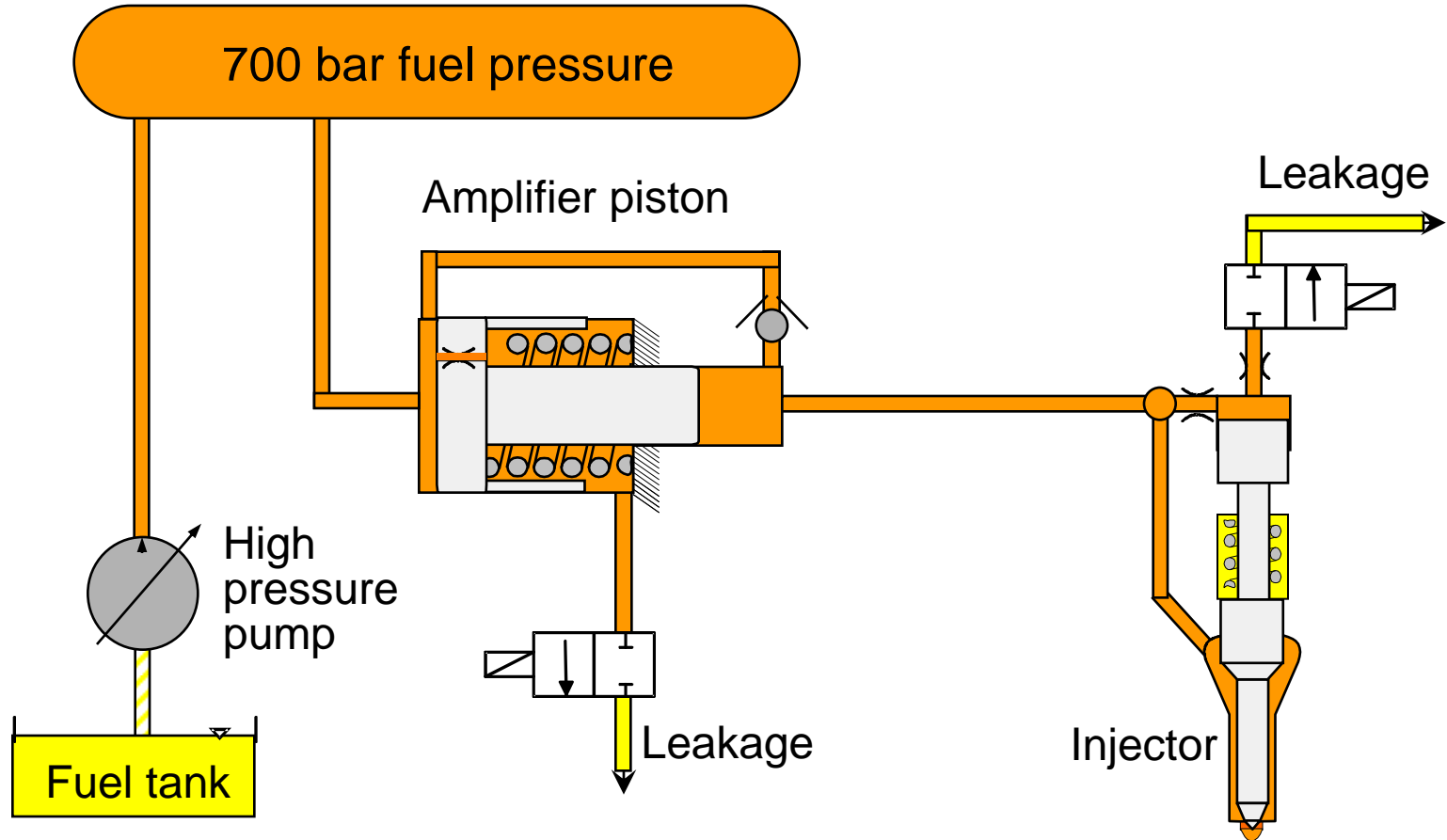
# High Pressure Amplifier Piston CR System APCRS – Schematic View



# High Pressure Amplifier Piston CR System APCRS – Schematic View

Pilot Injection

Common Rail

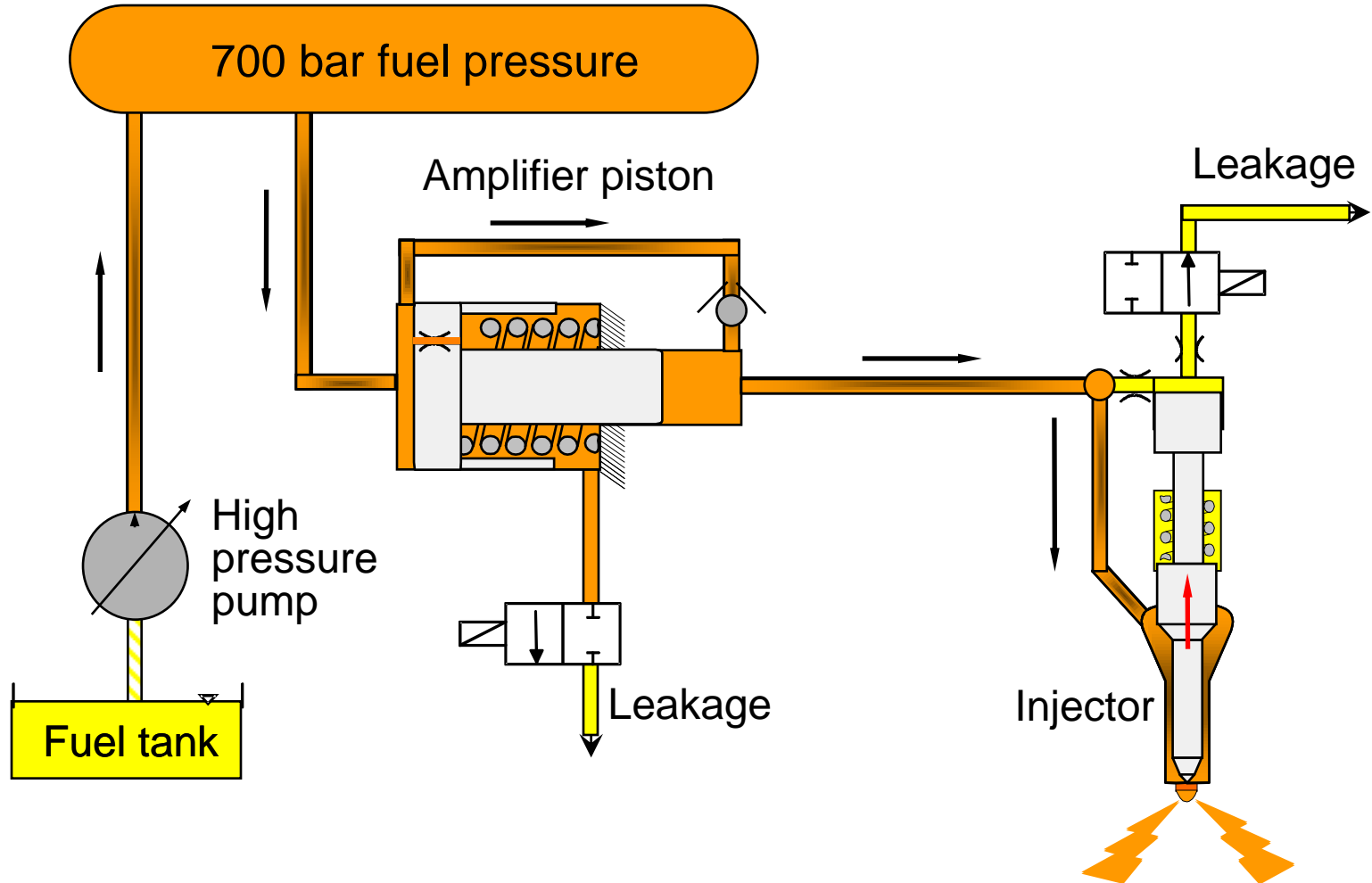




# High Pressure Amplifier Piston CR System APCRS – Schematic View

Pilot Injection

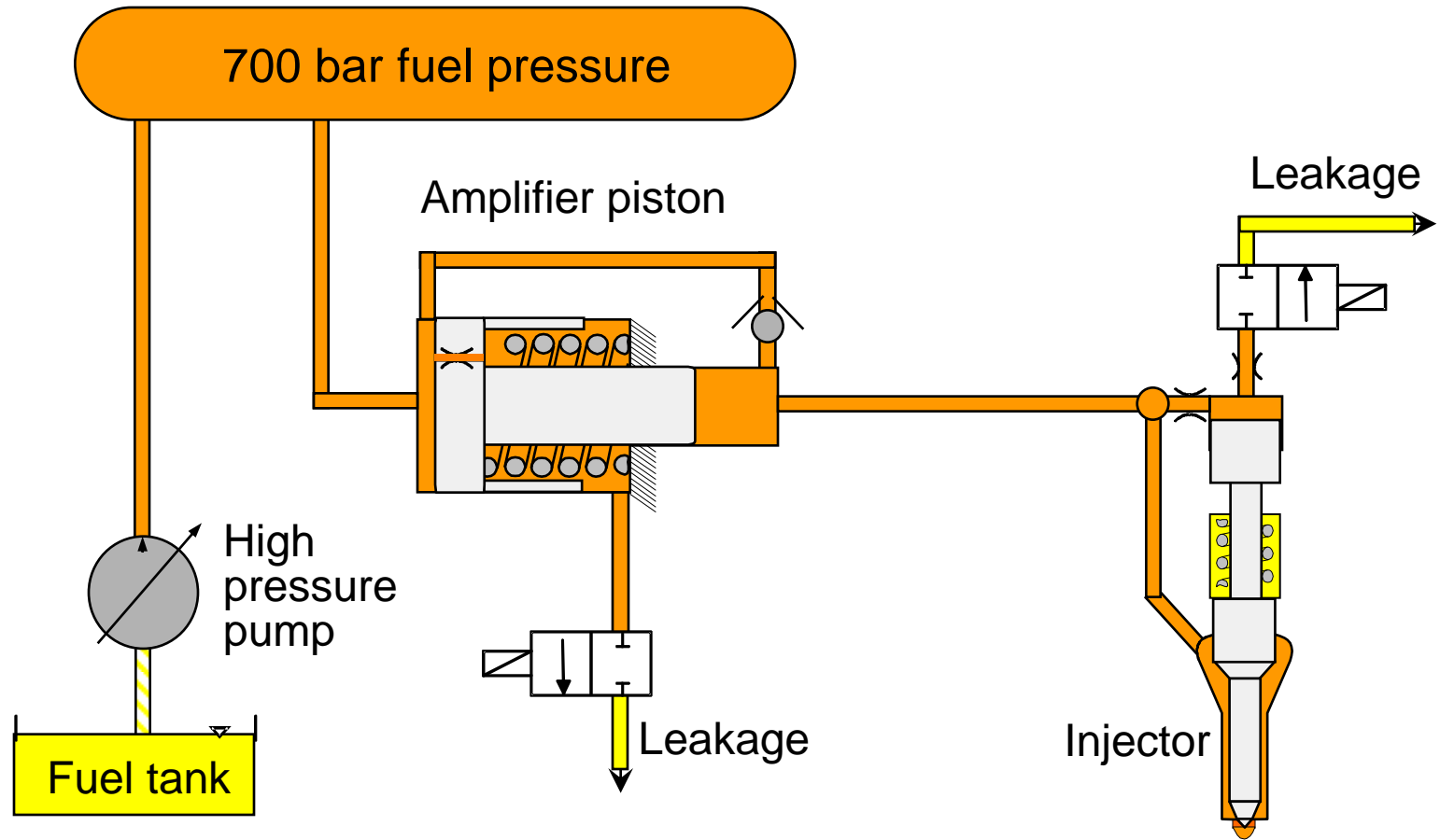
Common Rail



# High Pressure Amplifier Piston CR System APCRS – Schematic View

Main Injection

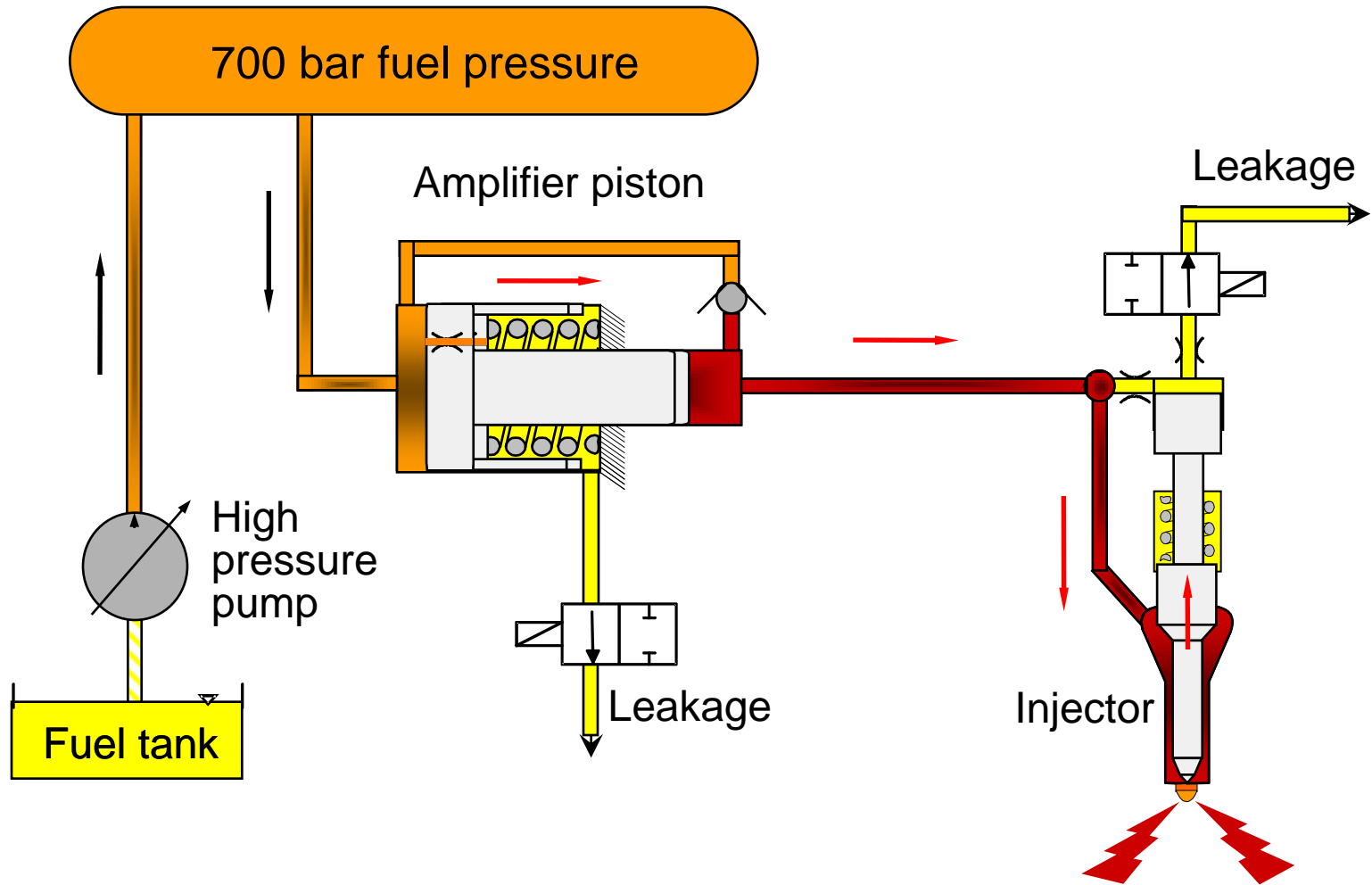
Common Rail



# High Pressure Amplifier Piston CR System APCRS – Schematic View

Main Injection

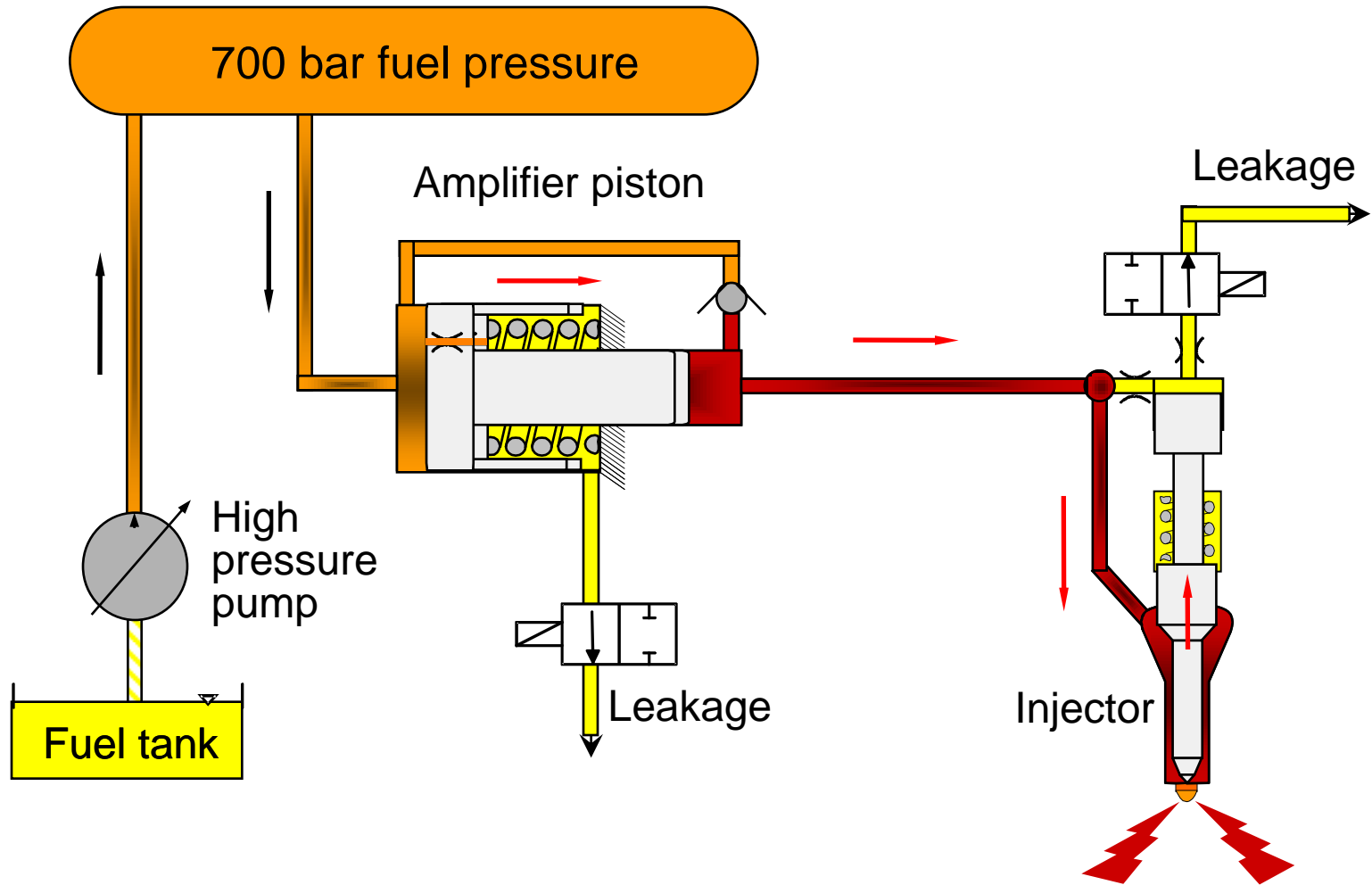
Common Rail



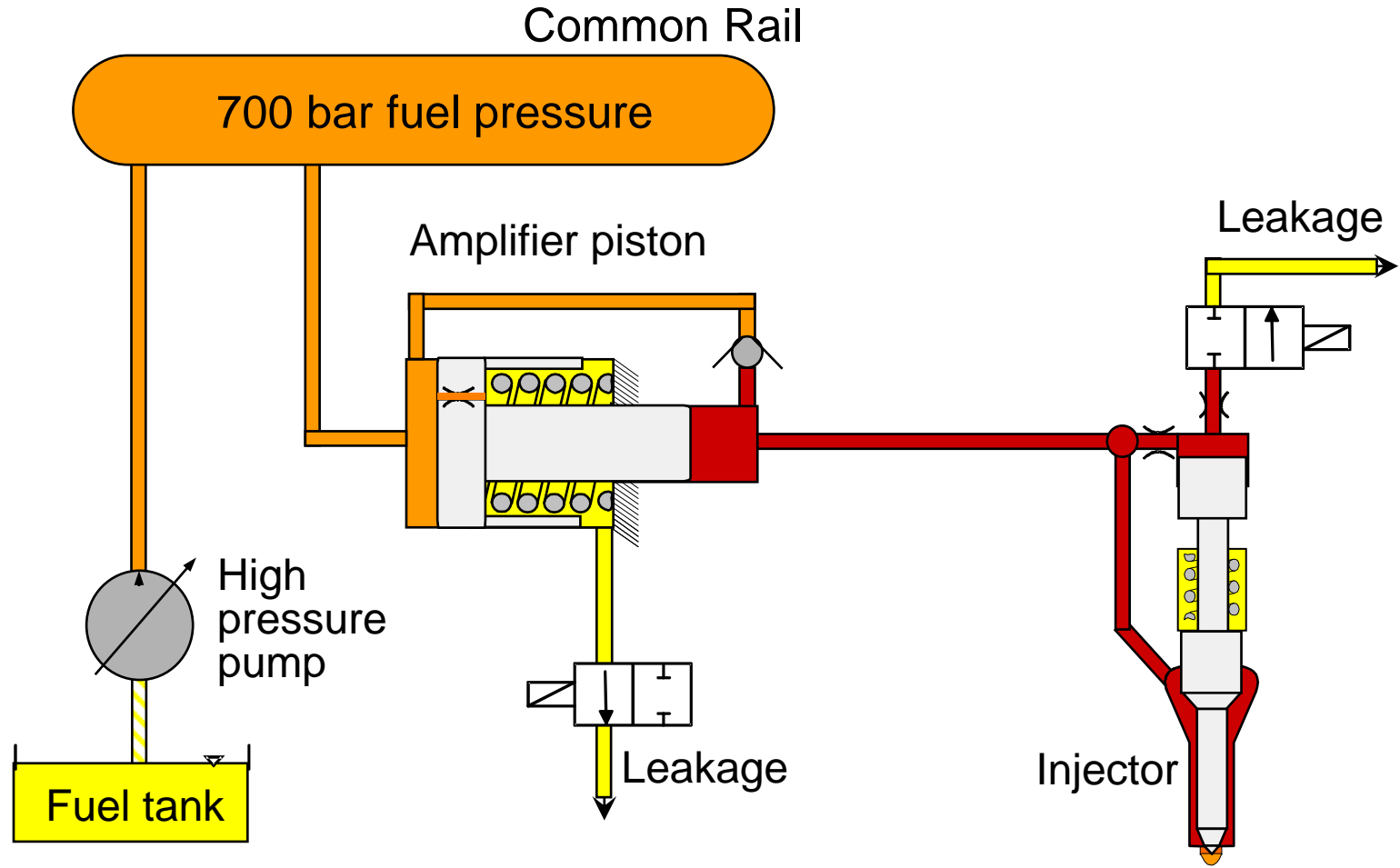
# High Pressure Amplifier Piston CR System APCRS – Schematic View

Main Injection

Common Rail



# High Pressure Amplifier Piston CR System APCRS – Schematic View

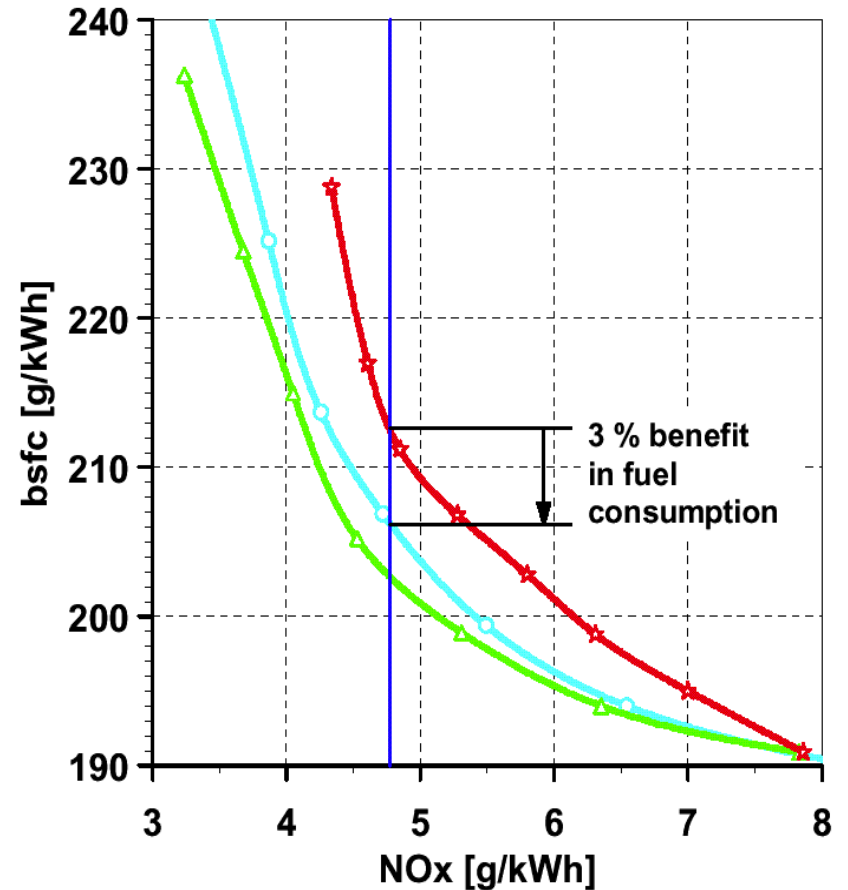
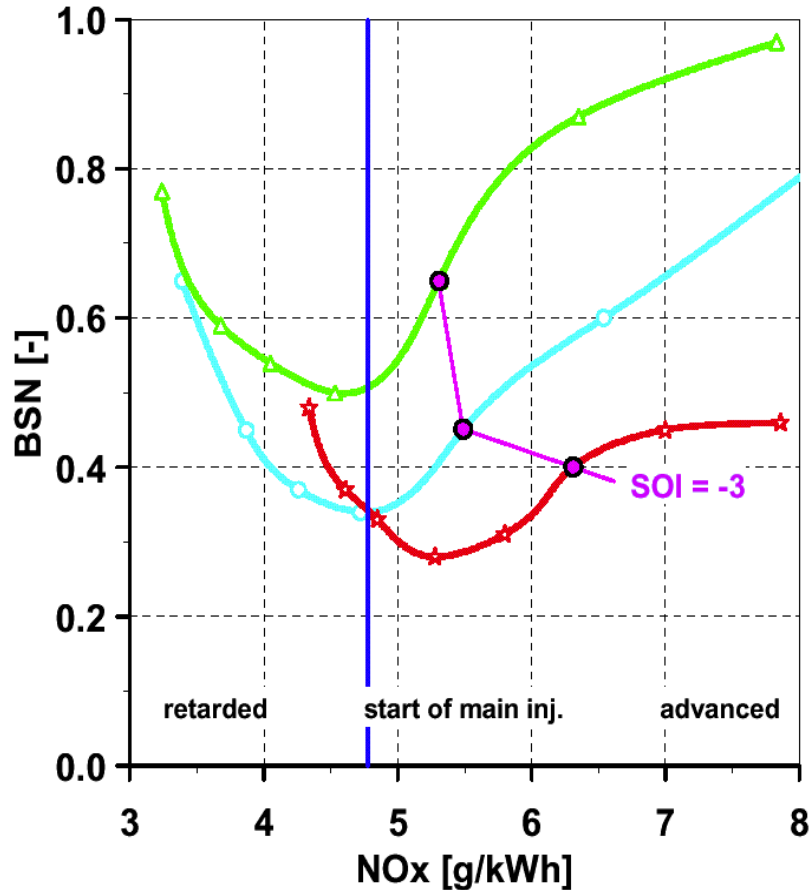


# HD-Engine with CRS, Pilot Injection

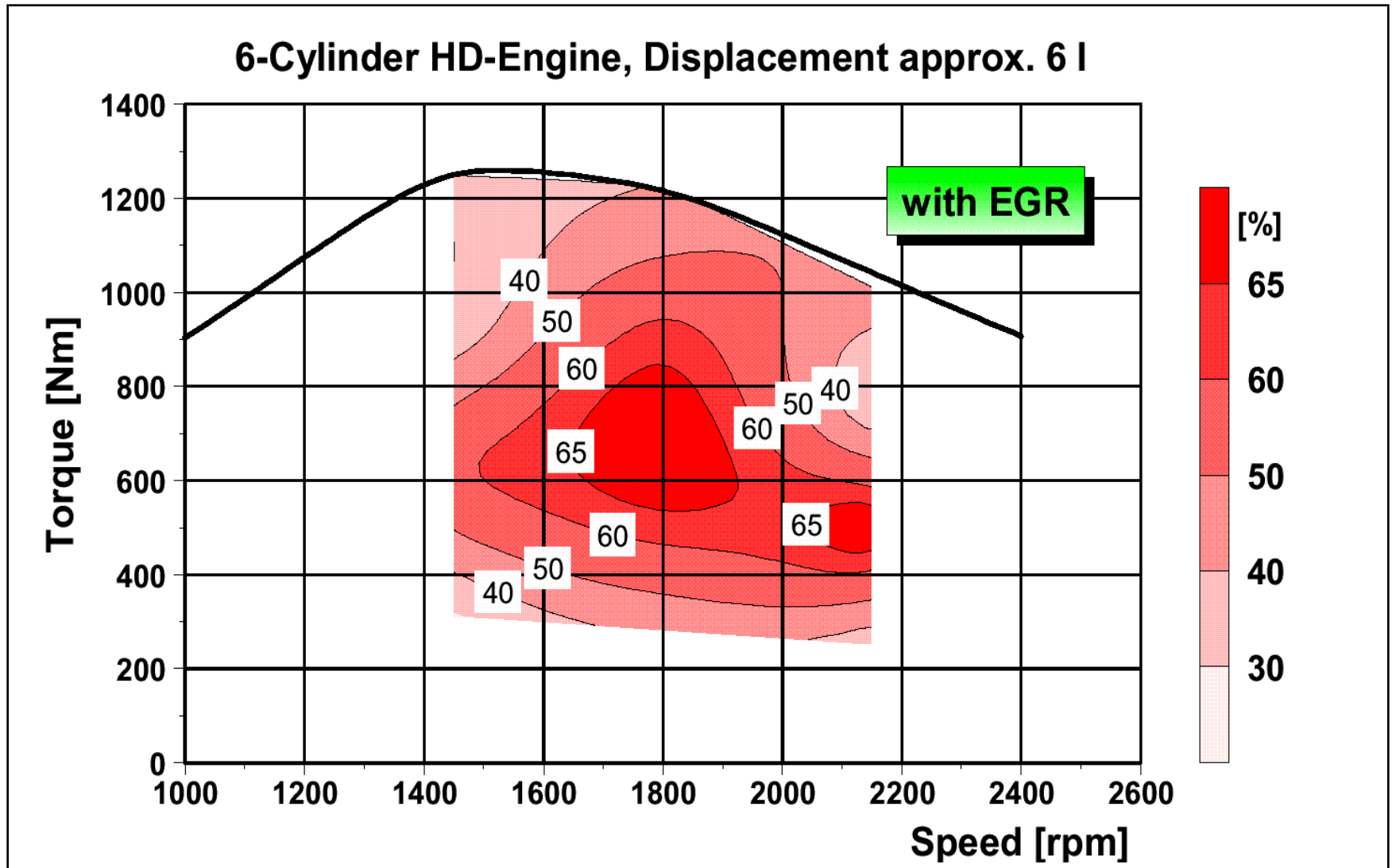
n = 1200 rpm, full load

variation of start of injection  
with pilot injection (rail pressure = 800 bar)  
distance between start of pilot and main inj. = 4 deg. cr.

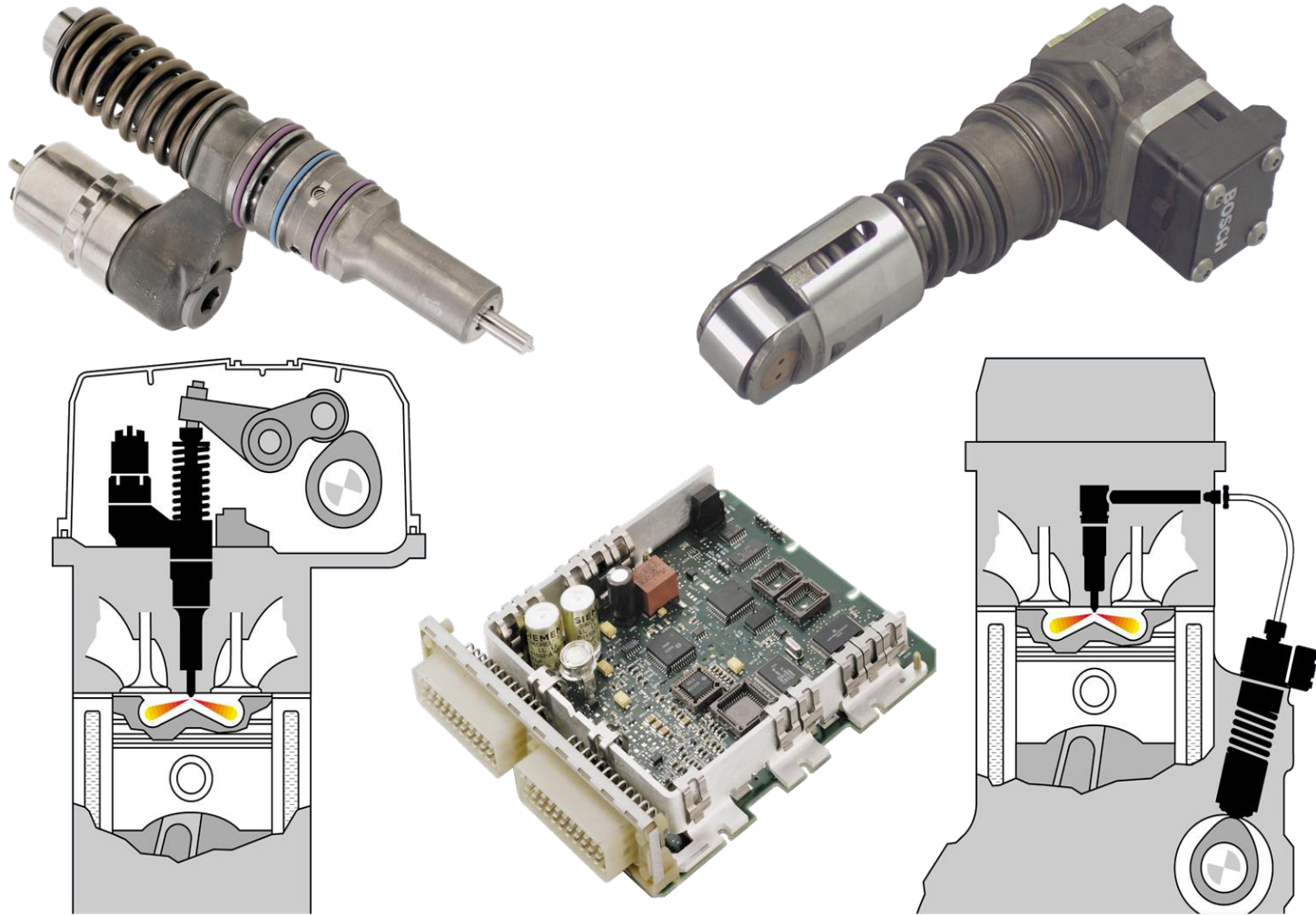
- without pilot injection
- with pilot injection (~1 mg/str)
- with pilot injection (~7 mg/str)



# Possible Black Smoke Reduction with Post-Injection

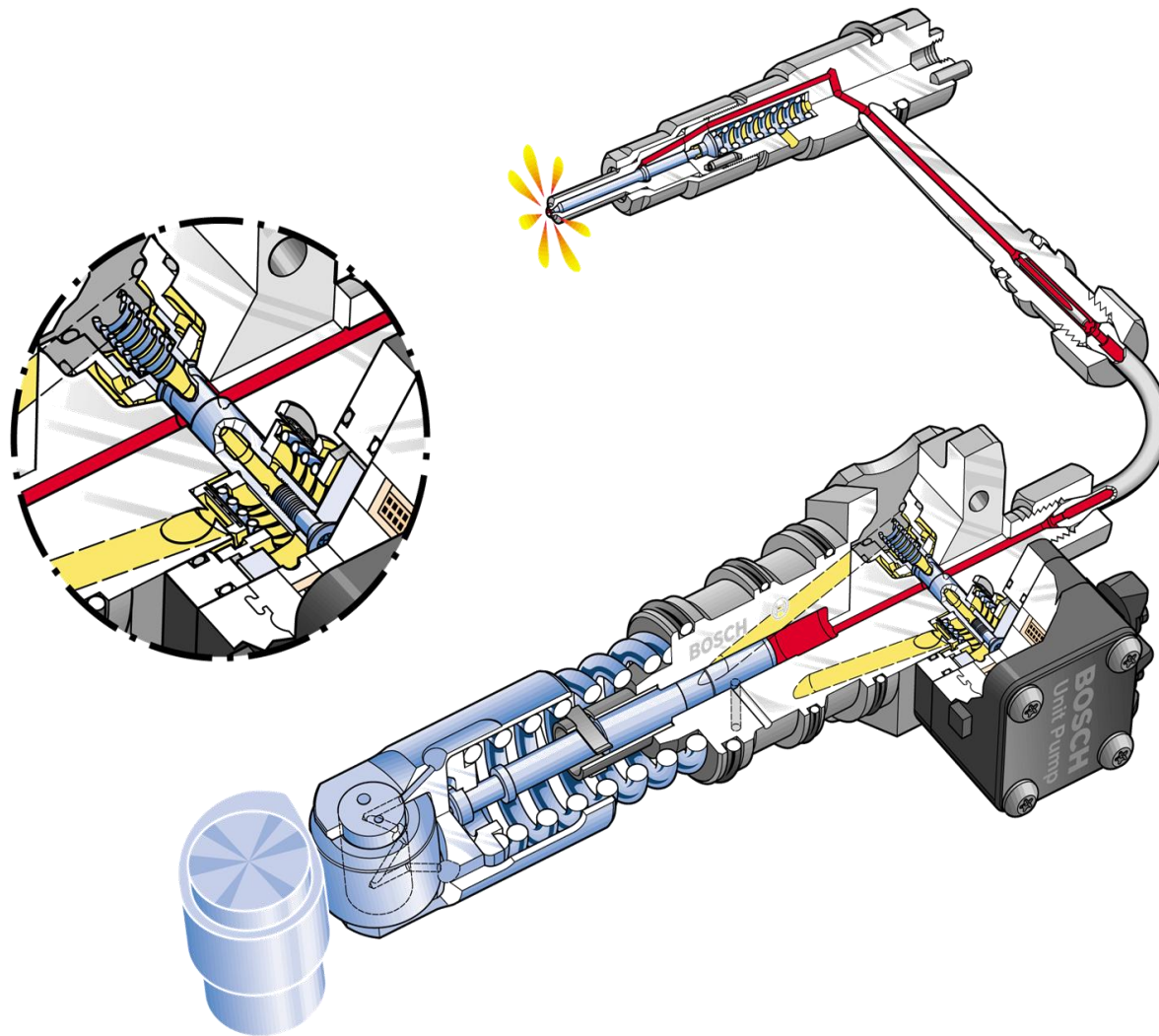


# Unit Injector and Unit Pump for Commercial Vehicles

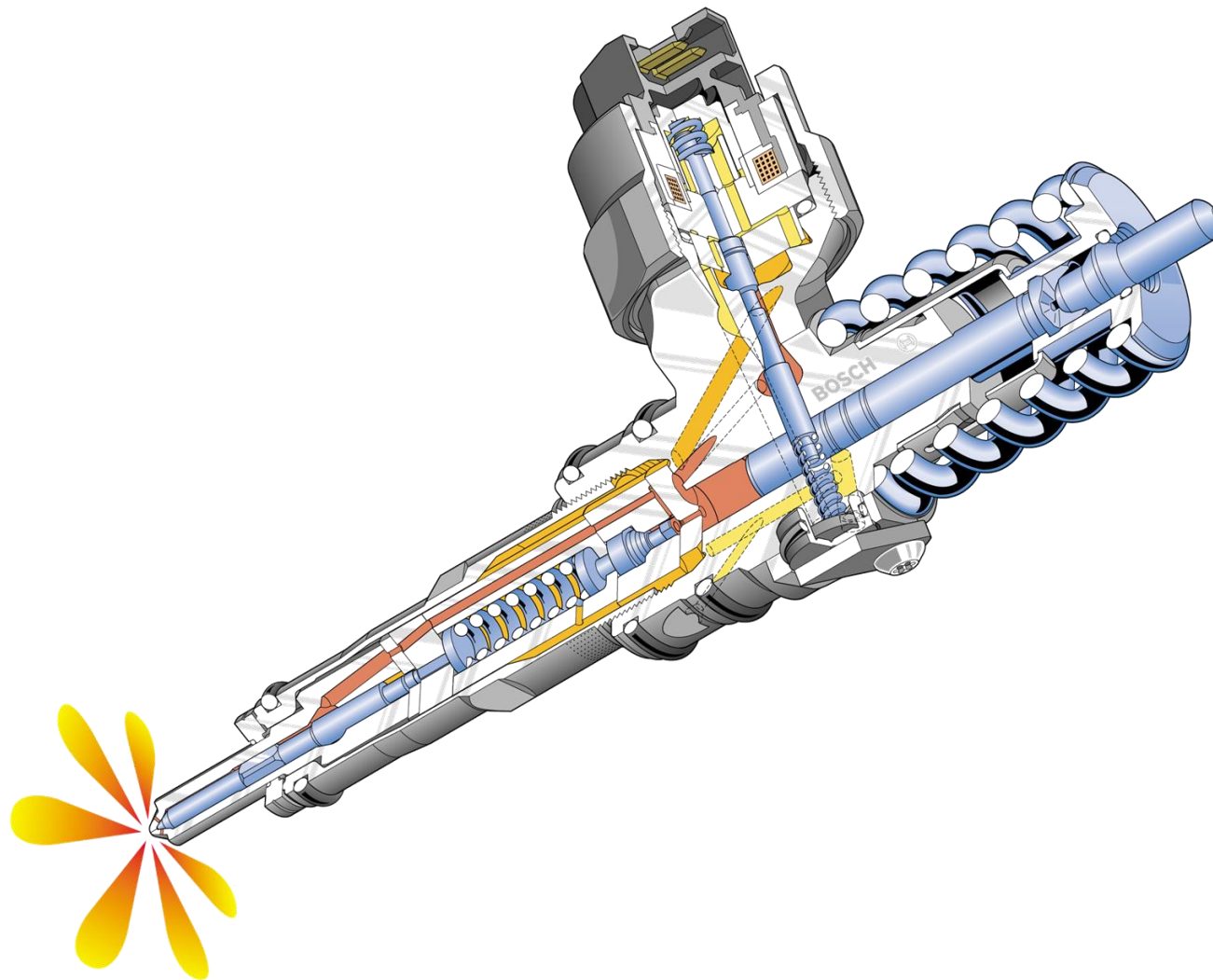




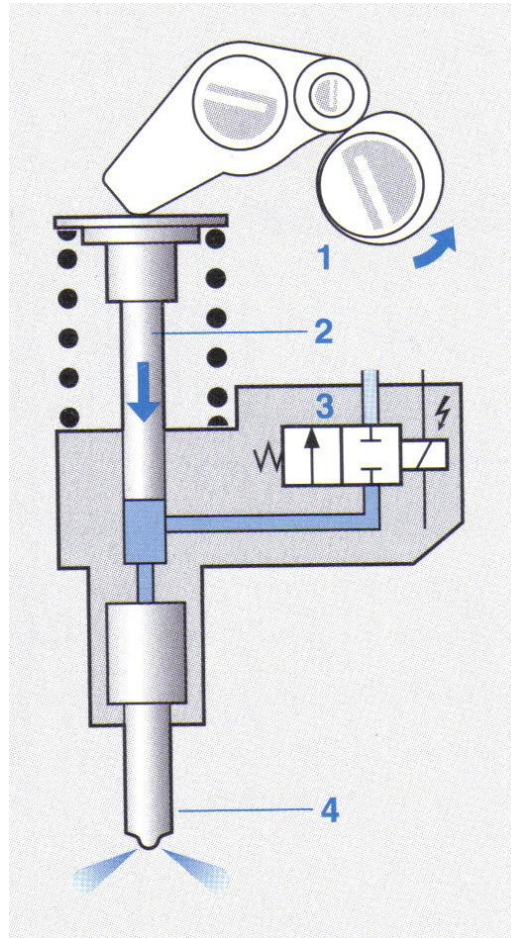
# Unit Pump for Commercial Vehicles



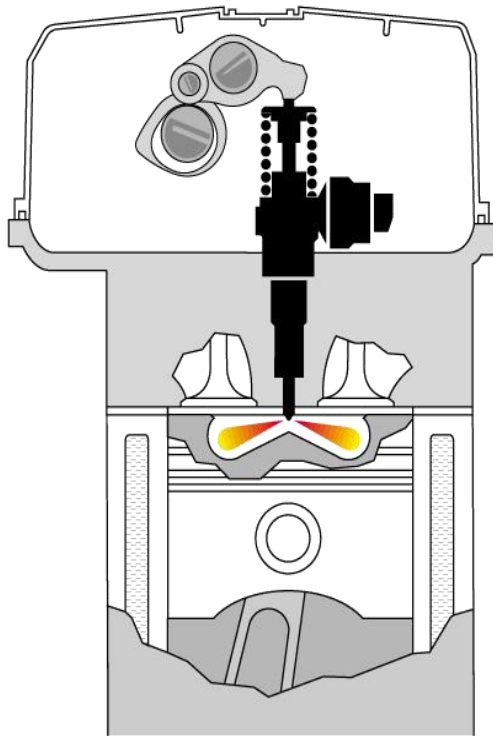
# Unit Injector for Passenger Cars



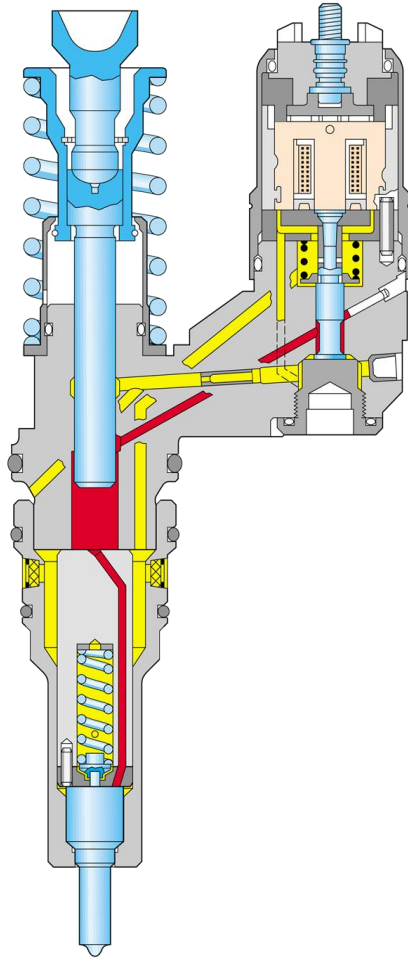
# Unit Injector for Passenger Cars



# Unit Injector System for Passenger Cars

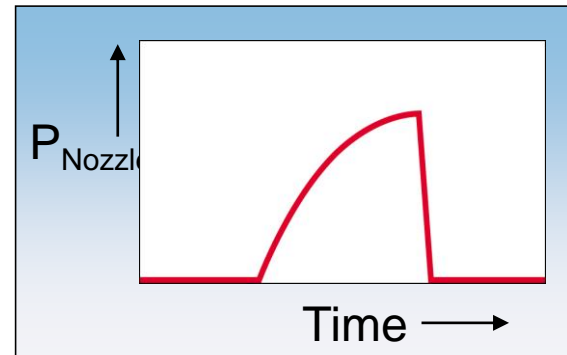


# Unit Injector System – Generations G1 and G2

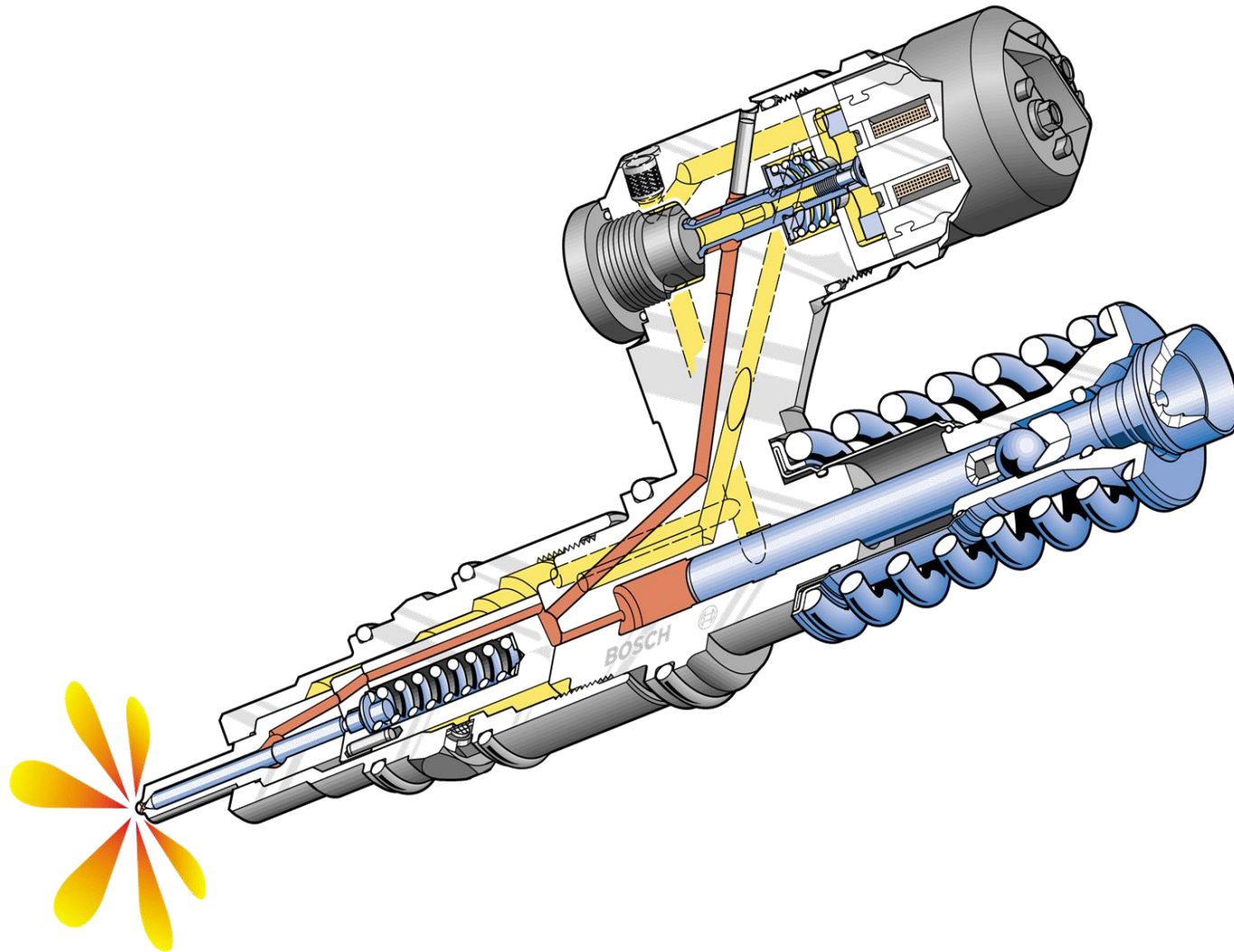


Generation N1 : 1600 bar






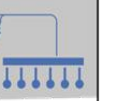

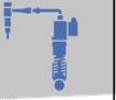





















Generation N2 : 1800 bar



# Unit Injector for Commercial Vehicles



# Applications of Diesel Fuel-Injection Equipment

Common Rail System						
Unit Pump System						
Unit Injector System						
Radial piston distributor pump						
Axial piston distributor pump						
Camshaftless pump						
Inline pump	