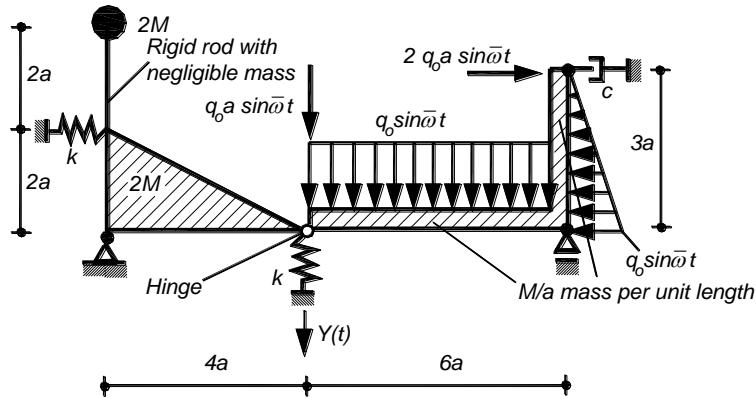
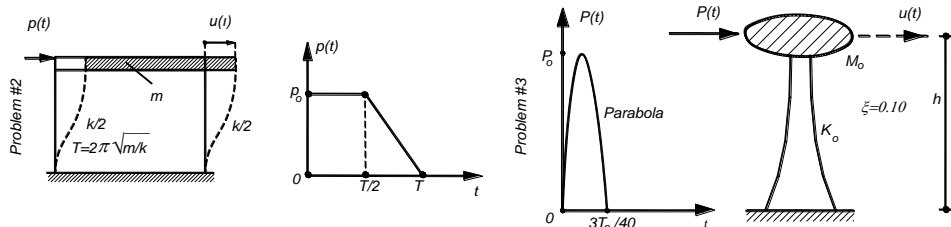


ADVANCED DYNAMICS OF STRUCTURES / HOMEWORK # 1; October 31, 2012

- Write down the equation of motion of the rigid-body assemblage in terms of  $Y(t)$  the vertical displacement of the hinge by using the principle of the virtual work. Obtain the free vibration period  $T = \alpha\sqrt{M/k}$  of the assemblage without considering the damping and determine  $\alpha$ . Find the resonance condition ( $\omega = \bar{\omega}$ ) in terms of the parameters of the system, when the damping is neglected.



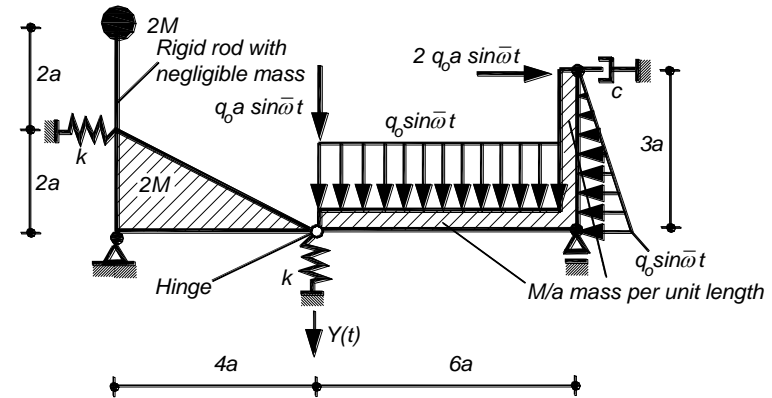
- A single degree of system of the mass  $m$ , the stiffness  $k$  is subjected to the external load  $p(t)$ . The variation of the external load is given as shown. Assuming the system starts from the rest position, i.e.,  $u(t=0) = 0$  and  $\dot{u}(t=0) = 0$ . Find the displacement function  $u(0 \leq t \leq T/2)$  by using the initial conditions and  $u(T/2 \leq t \leq T)$  and  $u(t \geq T)$  by using the continuity of the displacement and the velocity, where  $T$  is the free vibration period of the system. Draw the displacement variation  $u(0 \leq t \leq 2T) / u_{statik}$  where  $u_{statik} = p_0 / k$ .



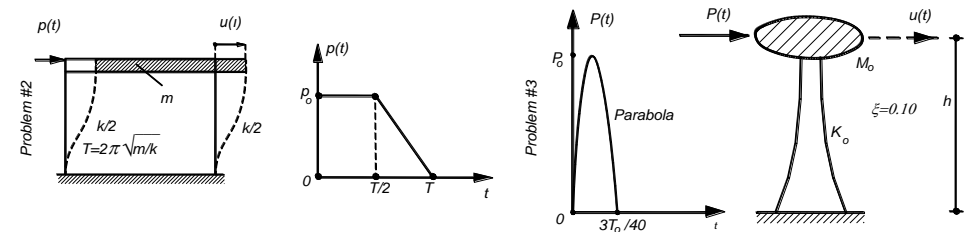
- The single-degree-of-freedom system shown is subjected to an external load of impulse characters by assuming that the system starts from the rest, i.e.,  $u(t=0) = 0$  and  $\dot{u}(t=0) = 0$ .  $M_0 \ddot{u}(t) + C_0 \dot{u}(t) + K_0 u(t) = P(t)$ . Find out the displacement  $u(t)$ , the velocity  $\dot{u}(t)$  and the acceleration  $\ddot{u}(t)$ . Obtain the maximum shear force and bending moment, where  $T_0$  is the free undamped vibration period of the system..  $M_0 g = 200kN$ ,  $K_0 = 1000kN/m$ ,  $\xi = 0.10$ ,  $P_0 = 100kN$ ,  $h = 8.0m$ .

ADVANCED DYNAMICS OF STRUCTURES / HOMEWORK # 1; October 31, 2012

- Write down the equation of motion of the rigid-body assemblage in terms of  $Y(t)$  the vertical displacement of the hinge by using the principle of the virtual work. Obtain the free vibration period  $T = \alpha\sqrt{M/k}$  of the assemblage without considering the damping and determine  $\alpha$ . Find the resonance condition ( $\omega = \bar{\omega}$ ) in terms of the parameters of the system, when the damping is neglected.



- A single degree of system of the mass  $m$ , the stiffness  $k$  is subjected to the external load  $p(t)$ . The variation of the external load is given as shown. Assuming the system starts from the rest position, i.e.,  $u(t=0) = 0$  and  $\dot{u}(t=0) = 0$ . Find the displacement function  $u(0 \leq t \leq T/2)$  by using the initial conditions and  $u(T/2 \leq t \leq T)$  and  $u(t \geq T)$  by using the continuity of the displacement and the velocity, where  $T$  is the free vibration period of the system. Draw the displacement variation  $u(0 \leq t \leq 2T) / u_{statik}$  where  $u_{statik} = p_0 / k$ .



- The single-degree-of-freedom system shown is subjected to an external load of impulse characters by assuming that the system starts from the rest, i.e.,  $u(t=0) = 0$  and  $\dot{u}(t=0) = 0$ .  $M_0 \ddot{u}(t) + C_0 \dot{u}(t) + K_0 u(t) = P(t)$ . Find out the displacement  $u(t)$ , the velocity  $\dot{u}(t)$  and the acceleration  $\ddot{u}(t)$ . Obtain the maximum shear force and bending moment, where  $T_0$  is the free undamped vibration period of the system..  $M_0 g = 200kN$ ,  $K_0 = 1000kN/m$ ,  $\xi = 0.10$ ,  $P_0 = 100kN$ ,  $h = 8.0m$ .

