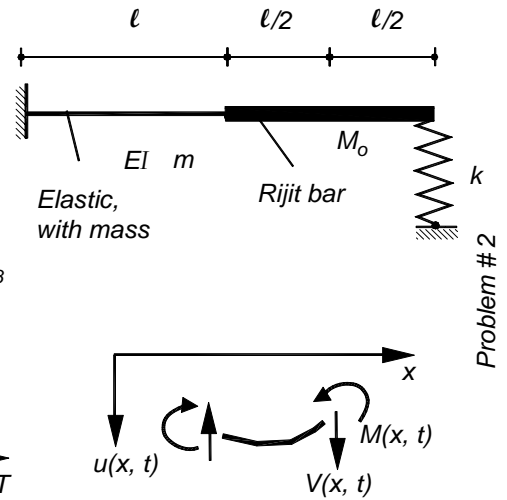
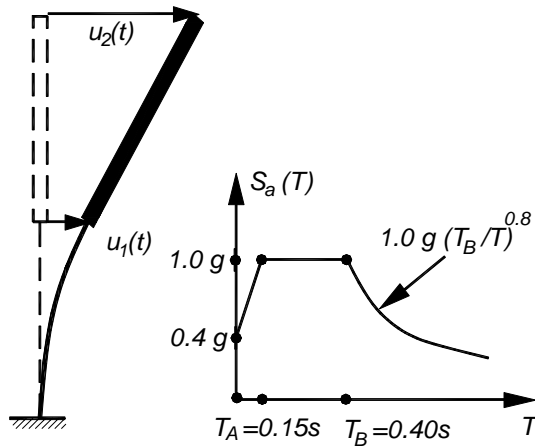
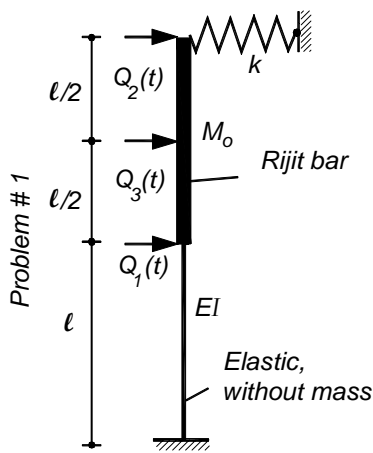


Problem # 1

Consider the system of two degrees-of-freedom shown:

- Write down the equations of motion of the system by including the external loads.
- Determine the two circular frequencies and the periods of the free vibration ω_i and T_i and the corresponding mode shapes ϕ_i . Give their graphical representation ($i = 1, 2$),
- Check the orthogonality of the modes with respect to the mass matrix and the stiffness matrix $\phi_1^T \mathbf{m} \phi_2$, and $\phi_1^T \mathbf{k} \phi_2$,
- Evaluate the generalized masses and stiffness $M_i = \phi_i^T \mathbf{m} \phi_i$, and $K_i = \phi_i^T \mathbf{k} \phi_i$, and assess $\omega_i^2 = K_i / M_i$ for $i = 1, 2$,
- The height of the column $\ell = 3 \text{ meter}$ and their cross section of $b/h = 0.25 \text{ m} \times 0.50 \text{ m}$, the first period of the system is $T_1 = 0.20 \text{ s}$ and $E = 30 \text{ GPa}$. Find the numerical values the parameter M_o and the second period T_2 of the system.
- Determine the effective modal masses M_1^* and M_2^* and assess that $M_1^* + M_2^* = M_o$
- Evaluate the base shear forces V_{b1} and V_{b2} and the overturning moments M_{b1} and M_{b2} corresponding to the two mode shapes. Obtain the base shear force V_b and the overturning moment M_b by using the SRSS combination rule.



Problem # 2

Write down the boundary conditions for the free vibration of the beam. Obtain the frequency determinant by assuming $M_o = m a$,

$k = EI / \ell^3$ in terms of β where $\beta^4 = \frac{m \ell^4 \omega^2}{EI}$

$$\mathbf{m} \ddot{\mathbf{u}}(t) + \mathbf{k} \mathbf{u}(t) = \mathbf{p}(t) \quad \mathbf{u}(t) = [u_1(t) \quad u_2(t)]^T \quad \mathbf{p}(t)^T = [P_1(t) \quad P_2(t)] \quad (\mathbf{k} - \omega_i^2 \mathbf{m}) \phi_i = 0 \quad \omega_i = 2 \pi / T_i$$

$$(\mathbf{I} - \omega_i^2 \mathbf{d} \mathbf{m}) \phi_i = 0 \quad |\mathbf{k} - \omega_i^2 \mathbf{m}| = 0 \quad K_i = \phi_i^T \mathbf{k} \phi_i \quad M_i \ddot{Y}_i(t) + K_i Y_i(t) = \phi_i^T \mathbf{p}(t) \quad \mathbf{u}(x, t) = \sum \phi_i(x) Y_i(t)$$

$$\ddot{Y}_i(t) + \omega_i^2 Y_i(t) = 0 \quad Y_i(t) = \frac{\sin \omega_i t}{M_i \omega_i} \left[\phi_i^T \int_0^t \mathbf{p}(\tau) d\tau \right] \quad Y_i(t) = \phi_i^T \mathbf{m} \mathbf{v} / M_i$$

$$M_i = \phi_i^T \mathbf{m} \phi_i \quad M(x, t) = -EI \frac{\partial^2 u}{\partial x^2} \quad V(x, t) = -EI \frac{\partial^3 u}{\partial x^3} \quad a^4 = \frac{m \omega^2}{EI}$$

$$\phi(x) = A_1 \sin ax + A_2 \cos ax + A_3 \sinh ax + A_4 \cosh ax \quad L_i = \phi_i^T \mathbf{m} \mathbf{1} \quad \Gamma_i = L_i / M_i \quad M_i^* = \Gamma_i L_i$$

$$I_\theta = \frac{M}{12} (a^2 + b^2) \quad V_{bj} = M_j^* A_{j \max} \quad f_{ij} = V_{bj} \frac{m_i \phi_{ij}}{\sum_{l=1}^n m_l \phi_{lj}} \quad M_j^* = \frac{\left(\sum_{i=1}^n m_i \phi_{ij} \right)^2}{\sum_{i=1}^n m_i \phi_{ij}^2}$$