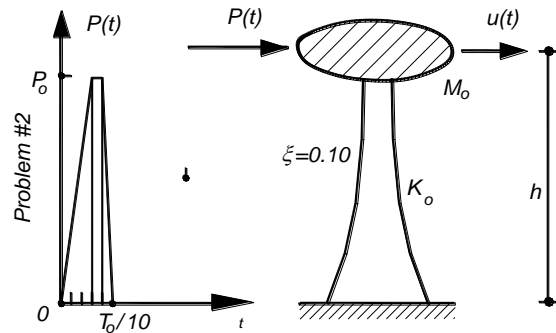
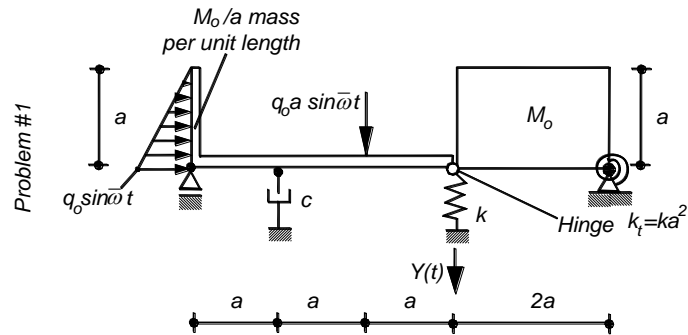


ADVANCED DYNAMICS OF STRUCTURES / Quiz # 1; November 7, 2012

1. Write down the equation of motion of the rigid-body assemblage in terms of $Y(t)$ the vertical displacement of the hinge by using the principle of the virtual work. Obtain the free vibration period $T = \alpha\sqrt{M/k}$ of the assemblage without considering the damping and determine α . Find the resonance condition ($\omega = \bar{\omega}$) in terms of the parameters of the system, when the damping is neglected.



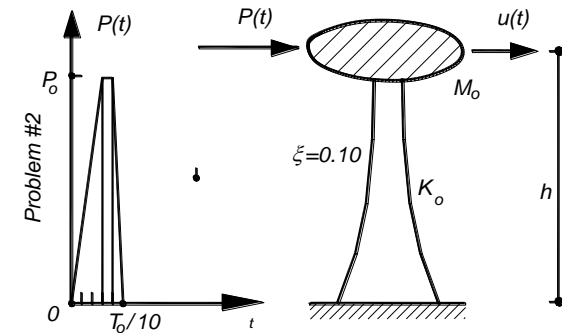
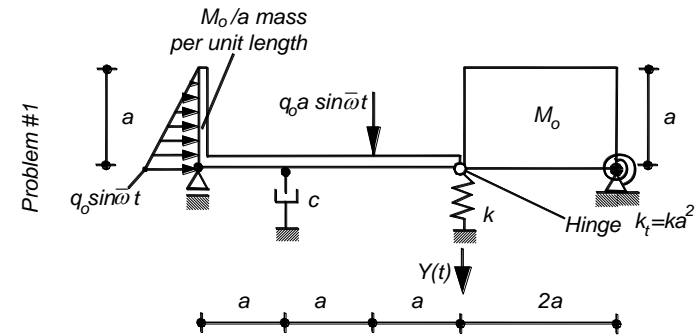
2. The single-degree-of-freedom system shown is subjected to an external load of impulse characters by assuming that the system starts from the rest, i.e., $u(t=0) = 0$ and $\dot{u}(t=0) = 0$. $M_0 \ddot{u}(t) + C_0 \dot{u}(t) + K_0 u(t) = P(t)$. Find out the displacement $u(t)$, the velocity $\dot{u}(t)$ and the acceleration $\ddot{u}(t)$. Obtain the maximum shear force and bending moment, where T_0 is the free undamped vibration period of the system.. $M_0 g = 200kN$, $K_0 = 800kN/m$, $\xi = 0.10$, $P_0 = 120kN$, $h = 10.0m$.

$$u(t) = \frac{I}{m \omega_D} e^{-\xi \omega t} \sin \omega_D t \quad I = \int_0^{t_1} p(t) dt \quad u(t)_{\max} = \frac{I}{m \omega} \exp \left[-\frac{\xi}{\sqrt{1-\xi^2}} \arctan \frac{\sqrt{1-\xi^2}}{\xi} \right]$$

$$u(t) = e^{-\xi \omega t} \left[\frac{\dot{u}(0) + u(0) \xi \omega}{\omega_D} \sin \omega_D t + u(0) \cos \omega_D t \right] + \frac{1}{m \omega_D} \int_0^t p(\tau) e^{-\xi \omega(t-\tau)} \sin \omega_D(t-\tau) d\tau$$

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