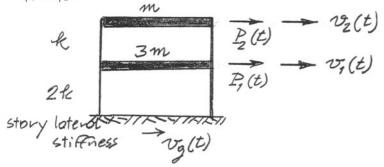
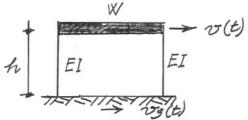
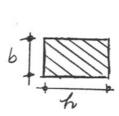
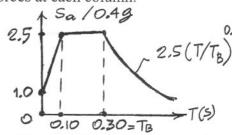
- (1) Consider the system of two-degrees-of-freedom shown:
- a Write down equations of motion of the system by including the ground motion $v_g(t)$ and the external loads $P_1(t)$ and $P_2(t)$. Obtain the mass matrix **m** and the rigidity matrix **k**.
- b Determine the two circular frequencies and the two periods of the free vibration $\omega_1 < \omega_2$, and $T_1 > T_2$ and the corresponding mode shapes ϕ_1 and ϕ_2 . Give their graphical representations.
- c Check the orthogonality of the modes with respect to the mass matrix $\phi_1^T \mathbf{m} \phi_2$ and the stiffness matrix $\phi_1^T k \phi_2$.



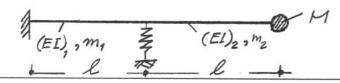
- (2) Consider the system of single-degrees-of-freedom shown:
- a. Write down equation of motion of the system by including the ground motion $v_g(t)$ and obtain the period of the system.
- b. Assuming h = 3m and the cross section of the columns b = 0.30m, h = 0.40m and E = 30 GPa, T = 0.2sobtain the weight of the system. By using the spectral curve given, obtain the base shear force V_b and the corresponding lateral deformation of the system. Determine the shear forces at each column.







Consider the cantilever beam having a lumped mass and a lateral spring at the tip where m_1 and m_2 are the masses per unit length and $(EI)_1$ and $(EI)_2$ the bending rigidities of the cross sections, k the constant of the spring and M the lumped mass at the tip. Write down the boundary conditions of the beam that experiences free vibration for obtaining the free vibration frequencies.



$$\mathbf{m} \ \ddot{\mathbf{v}}(t) + \mathbf{k} \ \mathbf{v}(t) = \mathbf{p}(t)$$

$$\mathbf{v}(t) = [v_1(t), v_2(t)]^T$$
 $\mathbf{p}(t)^T = [P_1(t), P_2(t)]$

$$\mathbf{p}(t)^T = \left[P_1(t), P_2(t) \right]$$

$$(\mathbf{k} - \omega_i^2 \mathbf{m}) \, \phi_i = 0$$

$$\left|\mathbf{k} - \omega_i^2 \mathbf{m}\right| = 0$$

$$\omega_i = 2 \pi / T_i$$

$$M_i = \phi_i^T \mathbf{m} \, \phi_i$$

$$K_i = \phi_i^T \mathbf{k} \ \phi_i$$

$$v(x,t) = \phi(x) Y(t)$$

$$\ddot{Y}(t) + \omega^2 Y(t) = 0$$

$$M(x,t) = -EI \partial^2 v / \partial x^2$$
 $V(x,t) = -EI \partial^3 v / \partial x^3$

$$V(x,t) = -EI \partial^3 v / \partial x$$

$$\phi(x) = A_1 \sin ax + A_2 \cos ax + A_3 \sinh ax + A_4 \cosh ax$$

$$a^4 = m\omega^2 / EI$$