## ADVANCED DYNAMICS OF STRUCTURES / Final Exam / December 22, 2014

## Question \# 1:

Consider the system of two degrees-of-freedom shown shown:
a. Write down the equations of motion of the system by considering the freebody diagram of the two story masses. Evaluate the mass matrix $\mathbf{m}$, the stiffness matrix $\mathbf{k}$.
b. Determine the two circular frequencies and periods of the free vibration $\omega_{i}$ and $T_{i}$ in terms of $M_{o}, E I$ and $\ell$. Obtain the corresponding mode shapes $\phi_{i}$ and give their graphical representations ( $i=1,2$ ).
a. Determine the effective modal masses $M_{1}^{*}$ and $M_{2}^{*}$, and assess the identity $M_{1}^{*}+M_{2}^{*}=3 M_{o}$
b. Assuming the heights of the stories are $\ell=3 \mathrm{~meter}$, the columns with a cross section of $b / h=0.40 \mathrm{~m} / 0.60 \mathrm{~m}$ and the first period of the system $T_{1}=0.15 s$ and the modulus elasticity $E=30 G P a$, find the numerical values of the mass $M_{o}$ and the second period $T_{2}$ of the system.
c. Evaluate the base shear forces $V_{b 1}, V_{b 2}$ corresponding to the two mode shapes by using the acceleration spectrum $S_{a}(T)$ given. Obtain the base shear force and their combined value $V_{b}$ by using the SRSS combination rule.

## Question \# 2:

Consider a beam having a length $\ell$, a mass per unit length $m$ and a bending rigidity $E I$. The beam is clamed at its right end and has a lumped mass $M_{o}$ at the left end which supported by a linear spring having a spring constant $k$. By considering free vibration of the beam, write down the boundary conditions and evaluate the frequency determinant.

## Question \# 3:

Express how the displacement spectrum of a seismic record can be obtained. Give mathematical definitions and symbolic variations for displacement, velocity and acceleration spectra for two damping ratios $\xi$. Write down the relationships between these three spectra. Discuss properties of these three spectra for the cases $T \rightarrow 0$ and $T \rightarrow$ large .

$m \ddot{u}+c \dot{u}+k u=-m \ddot{u}_{g}(t) \quad \ddot{u}+2 \xi \omega \dot{u}+\omega^{2} u=-\ddot{u}_{g}(t) \quad \omega_{D}=\omega \sqrt{\left(1-\xi^{2}\right)}$
$u(t, \xi, \omega)=-\frac{1}{\omega_{D}} \int_{o}^{t} \ddot{u}_{g}(\tau) \exp [-\xi \omega(t-\tau)] \sin \left[\omega_{D}(t-\tau)\right] d \tau$
$\dot{u}(t, \xi, \omega)=-\int_{o}^{t} \ddot{u}_{g}(\tau) \exp [-\xi \omega(t-\tau)] \cos \left[\omega_{D}(t-\tau)\right] d \tau-\xi \omega u(t, \xi, \omega)$
$\mathbf{m} \ddot{\mathbf{u}}(t)+\mathbf{k} \mathbf{u}(t)=\mathbf{p}(t) \quad \mathbf{u}(t)=\left[\begin{array}{ll}u_{1}(t) & u_{2}(t)\end{array}\right]^{T} \mathbf{p}(t)^{T}=\left[\begin{array}{ll}P_{1}(t) & P_{2}(t)\end{array}\right] \quad \omega_{i}=2 \pi / T_{i}$

$\left(\mathbf{k}-\omega_{i}^{2} \mathbf{m}\right) \phi_{i}=0 \quad\left(\mathbf{I}-\omega_{i}^{2} \mathbf{d} \mathbf{m}\right) \phi_{i}=0 \quad\left|\mathbf{k}-\omega_{i}^{2} \mathbf{m}\right|=0 \quad\left|\mathbf{I}-\omega_{i}^{2} \mathbf{d} \mathbf{m}\right|=0 \quad M_{i}=\boldsymbol{\phi}_{i}^{T} \mathbf{m} \boldsymbol{\phi}_{i}$
$K_{i}=\phi_{i}^{T} \mathbf{k} \phi_{i} \quad M_{i} \ddot{Y}_{i}(t)+K_{i} Y_{i}(t)=\phi_{i}^{T} \mathbf{p}(t) \quad Y_{i}(t)=\sum_{i=1}^{2} \phi_{i}{ }^{T} \mathbf{m} \mathbf{v} / M_{i} \quad k=\frac{3 E I}{h^{3}} \quad k=\frac{12 E I}{h^{3}}$
$Y_{i}(t)=\frac{\sin \omega_{i} t}{M_{i} \omega_{i}}\left[\phi_{i}^{T} \int_{o}^{t_{o}} \mathbf{p}(\tau) d \tau\right] \quad L_{i}=\phi_{i}^{T} \mathbf{m} 1 \quad \Gamma_{i}=L_{i} / M_{\mathrm{i}} \quad M_{i}^{*}=\Gamma_{i} L_{i} \quad 1=\left[\begin{array}{ll}1 & 1\end{array}\right]^{T} \quad V_{b j}=M_{j}^{*} S_{a}\left(T_{j}\right)$
$u(x, t)=\sum \phi_{i}(x) Y_{i}(t) \quad \ddot{Y}_{i}(t)+\omega_{i}^{2} Y_{i}(t)=0 \quad M(x, t)=-E I \frac{\partial^{2} u}{\partial x^{2}} \quad V(x, t)=-E I \frac{\partial^{3} u}{\partial x^{3}} \quad f_{n j}=V_{b j} \frac{m_{n} \phi_{n j}}{\sum_{k} m_{k} \phi_{k j}}$
$\phi(x)=A_{1} \sin a x+A_{2} \cos a x+A_{3} \sinh a x+A_{4} \cosh a x \quad a^{4}=\frac{m \omega^{2}}{E I}$

