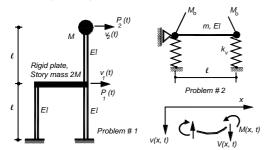
ADVANCED DYNAMICS OF STRUCTURES / Midterm Exam / December 15, 2010

H.Boduroğlu / Z. Celep

Problem #1

Consider the system of two degrees-of-freedom shown:

- a. Evaluate the flexibility **d** matrix, the mass matrix **m** and the rigidity matrix $\mathbf{k} = \mathbf{d}^{-1}$ and the load vector **p**.
- b. Determine the circular frequencies and the periods of the free vibration ω_i and T_i in terms of EI, m and ℓ . Obtain the corresponding two mode shapes ϕ_i and give their graphical representation (i=1,2),
- c. Check the orthogonality of the modes with respect to the mass matrix and the stiffness matrix $\phi_1^T \mathbf{m} \ \phi_2$, and $\phi_1^T \mathbf{k} \ \phi_2$.
- d. Evaluate the generalized masses and stiffness $M_i = \phi_i^T \mathbf{m} \phi_i$, and $K_i = \phi_i^T \mathbf{k} \phi_i$, and assess the relationship $\omega_i^2 = K_i/M_i$. (i = 1, 2),



Problem # 2:

Consider the distributed parameter system shown where m is the mass per unit length and EI is the bending rigidity of the cross section. The beam has two lumped masses of M_o at the two ends. Write down the boundary conditions for the free vibration of the beam. Obtain all

parameters in terms of $\beta^4 = \frac{m \ell^4 \omega^2}{EI}$ by assuming $M_o = m \ell$ and $k_v = EI/\ell^3$.

$$\mathbf{m} \ddot{\mathbf{v}}(t) + \mathbf{k} \mathbf{v}(t) = \mathbf{p}(t) \qquad \mathbf{v}(t) = \begin{bmatrix} v_1(t) & v_2(t) \end{bmatrix}^T \qquad \mathbf{p}(t)^T = \begin{bmatrix} P_1(t) & P_2(t) \end{bmatrix} \qquad (\mathbf{k} - \omega_i^2 \mathbf{m}) \, \phi_i = 0$$

$$(\mathbf{I} - \omega_i^2 \mathbf{d} \mathbf{m}) \, \phi_i = 0 \qquad \begin{vmatrix} \mathbf{k} - \omega_i^2 \mathbf{m} \end{vmatrix} = 0 \qquad \begin{vmatrix} \mathbf{I} - \omega_i^2 \mathbf{d} \mathbf{m} \end{vmatrix} = 0 \qquad \omega_i = 2 \, \pi / T_i \qquad M_i = \phi_i^T \mathbf{m} \, \phi_i$$

$$K_i = \boldsymbol{\phi}_i^T \mathbf{k} \; \boldsymbol{\phi}_i \qquad M_i \; \ddot{Y}_i(t) + K_i \; Y_i(t) = \boldsymbol{\phi}_i^T \mathbf{p}(t) \qquad Y_i(t) = \sum_{i=1}^2 \; \boldsymbol{\phi}_i^T \; \mathbf{m} \; \mathbf{v} / M_i \qquad k = \frac{3EI}{h^3} \quad k = \frac{12EI}{h^3}$$

$$Y_i(t) = \frac{\sin \omega_i t}{M_i \ \omega_i} \left[\mathbf{\phi}_i^T \int_0^{t_O} \mathbf{p}(\tau) \ d\tau \right] \qquad L_i = \phi_i^T \text{m 1} \qquad \Gamma_i = L_i \ / \ M_i \qquad M_i^* = \Gamma_i \ L_i$$

$$\mathbf{1}^{\mathrm{T}} = \begin{bmatrix} 1 & 1 \end{bmatrix} \quad \mathbf{v}(x,t) = \sum \mathbf{\phi}_{i}(x) Y_{i}(t) \qquad \ddot{Y}_{i}(t) + \omega_{i}^{2} Y_{i}(t) = 0 \qquad M(x,t) = -EI \frac{\partial^{2} v}{\partial x^{2}}$$

$$\phi(x) = A_1 \sin ax + A_2 \cos ax + A_3 \sinh ax + A_4 \cosh ax \qquad a^4 = \frac{m \omega^2}{EI} \qquad V(x, t) = -EI \frac{\partial^3 v}{\partial x^3}$$

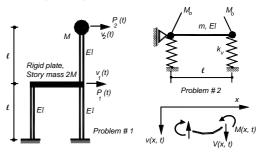
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