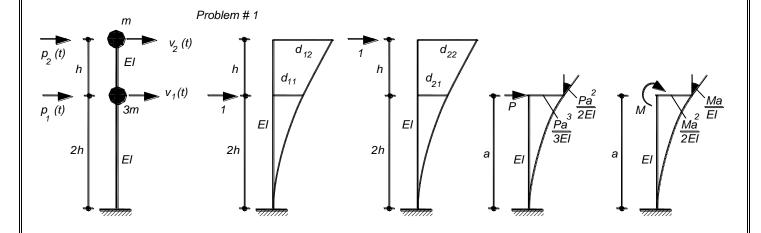
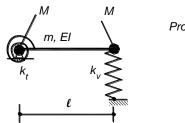
Problem #1

Consider the system of two degrees-of-freedom shown:

- a. Evaluate the flexibility **d** matrix, the mass matrix **m** and the rigidity matrix $\mathbf{k} = \mathbf{d}^{1}$ and the load vector **p**.
- b. Determine the circular frequencies and the periods of the free vibration ω_i and T_i in terms of EI, m and h. Obtain the corresponding two mode shapes ϕ_i and give their graphical representation (i=1,2),
- c. Check the orthogonality of the modes with respect to the mass matrix and the stiffness matrix $\phi_1^T m \phi_2$, and $\phi_1^T k \phi_2$.
- d. Evaluate the generalized masses and stiffness $M_i = \phi_i^T \mathbf{m} \phi_i$, and $K_i = \phi_i^T \mathbf{k} \phi_i$, and assess the relationship $\omega_i^2 = K_i / M_i$. (i = 1, 2),









Problem #2:

Consider the distributed parameter system shown where m is the mass per unit length and EI is the bending rigidity of the cross section. The beam has two lumped masses of M at the two ends. Write down the boundary conditions for the free vibration of the

beam. Obtain all parameters in terms of $\beta^4 = \frac{m \, \ell^4 \, \omega^2}{EI}$ by assuming $M = m \, \ell$, $k_v = E \, I / \ell^3$ and $k_t = E \, I / \ell$.

$$\begin{aligned} \mathbf{m} \ \ddot{\mathbf{v}}(t) + \mathbf{k} \ \mathbf{v}(t) &= \mathbf{p}(t) \\ \mathbf{k} - \omega_i^2 \ \mathbf{m} \end{vmatrix} = 0 \\ \mathbf{k} - \omega_i^2 \ \mathbf{m} \end{vmatrix} = 0 \\ \mathbf{k} - \omega_i^2 \ \mathbf{m} \end{vmatrix} = 0 \\ \mathbf{v}(t) &= \begin{bmatrix} v_1(t) & v_2(t) \end{bmatrix}^T \\ \mathbf{p}(t)^T &= \begin{bmatrix} P_1(t) & P_2(t) \end{bmatrix} \\ \mathbf{k} - \omega_i^2 \ \mathbf{m} \end{vmatrix} \phi_i \\ \mathbf{k} &= \phi_i^T \mathbf{k} \ \phi_i \\ \mathbf{k} &= \phi_i^T \mathbf{p}(t) \\ \mathbf{k} &= \frac{3EI}{h^3} \\ \mathbf{k} &= \frac{12EI}{h^3} \\ \mathbf{k} &= \frac{12EI}{h^3} \\ \mathbf{k} &= \phi_i^T \mathbf{m} \mathbf{1} \\$$