## Problem \# 1

Consider the system of two degrees-of-freedom shown:
a. Write down the equations of motion of the system by including the ground motion $v_{g}(t)$ and the external forces $P_{1}(t)$ and $P_{2}(t)$.
b. Evaluate the mass matrix $\mathbf{m}$, the rigidity matrix $\mathbf{k}$, and the fle xibility matrix $\mathbf{k}=\mathbf{d}^{-1}$,
c. Determine the circular frequencies and the periods of the free vibration $\omega_{i}$ and $T_{i}$ in terms of $E I, M$ and $h$. Obtain the corresponding two mode shapes $\phi_{i}$ and give their graphical representation ( $i=1,2$ ),
d. Check the orthogonality of the modes with respect to the mass matrix and the stiffness matrix $\phi_{1}{ }^{\mathbf{T}} \mathbf{m} \phi_{2}$, and $\phi_{1}{ }^{\mathbf{T}} \mathbf{k} \phi_{2}$.
e. Evaluate the generalized masses and stiffness $M_{i}=\phi_{\mathrm{i}}^{\mathrm{T}} \mathbf{m} \phi_{\mathrm{i}}$, and $K_{i}=\phi_{\mathrm{i}}^{\mathbf{T}} \mathbf{k} \phi_{\mathrm{i}}$, and assess $\omega_{i}{ }^{2}=K_{i} / M_{i}$. $(i=1,2)$,


## Problem \# 2:

Consider the distributed parameter system shown where $m$ is the mass per unit length and $E I$ is the bending rigidity of the cross section. The beam has a lumped mass of $M$ at the right end. Write down the boundary conditions for the free vibration of the beam. By assuming $M=m \ell, k=E I / \ell^{3}$ obtain all parameters in terms of $\alpha^{4}=\frac{m \ell^{4} \omega^{2}}{E I}$.
$\mathbf{m} \ddot{\mathbf{v}}(t)+\mathbf{k} \mathbf{v}(t)=\mathbf{p}(t)$

$$
\mathbf{v}(t)=\left[\begin{array}{ll}
v_{1}(t) & v_{2}(t)
\end{array}\right]^{T}
$$

$\mathbf{p}(t)^{T}=\left[\begin{array}{ll}P_{1}(t) & P_{2}(t)\end{array}\right]$
$\left(\mathbf{k}-\omega_{i}^{2} \mathbf{m}\right) \phi_{i}=0 \quad\left(\mathbf{I}-\omega_{i}^{2} \mathbf{d} \mathbf{m}\right) \phi_{i}=0$
$\left|\mathbf{I}-\omega_{i}^{2} \mathbf{d} \mathbf{m}\right|=0 \quad \omega_{i}=2 \pi / T_{i}$
$M_{i}=\phi_{i}^{T} \mathbf{m} \phi_{i}$
$\left|\mathbf{k}-\omega_{i}^{2} \mathbf{m}\right|=0$

$$
K_{i}=\phi_{i}^{T} \mathbf{k} \phi_{i}
$$

$M_{i} \ddot{Y}_{i}(t)+K_{i} Y_{i}(t)=\phi_{i}^{T} \mathbf{p}(t)$

$$
v(t)=\sum_{i=1}^{2} Y_{i}(t) \phi_{i}
$$

$Y_{i}(t)=\sum_{i=1}^{2} \phi_{i}{ }^{T} \mathbf{m} \mathbf{v} / M_{i}$

$Y_{i}(t)=\frac{\sin \omega_{i} t}{M_{i} \omega_{i}}\left[\phi_{i}^{T} \int_{o}^{t_{o}} \mathbf{p}(\tau) d \tau\right]$
$k=\frac{3 E I}{h^{3}}$
$k=\frac{12 E I}{h^{3}}$
$L_{i}=\phi_{i}^{T} \mathrm{~m} 1 \quad \Gamma_{i}=L_{i} / M_{\mathrm{i}} \quad M_{i}^{*}=\Gamma_{i} L_{i}$
$\phi(x)=A_{1} \sin a x+A_{2} \cos a x+A_{3} \sinh a x+A_{4} \cosh a x$

$$
\begin{array}{ll}
1^{\mathrm{T}}=\left[\begin{array}{ll}
1 & 1
\end{array}\right] & v(x, t)=\sum \phi_{i}(x) Y_{i}(t) \quad \ddot{Y}_{i}(t)+\omega_{i}^{2} Y_{i}(t)=0 \\
a^{4}=\frac{m \omega^{2}}{E I} & V(x, t)=-E I \frac{\partial^{3} v}{\partial x^{3}}
\end{array} \quad M(x, t)=-E I \frac{\partial^{2} v}{\partial x^{2}}
$$

